

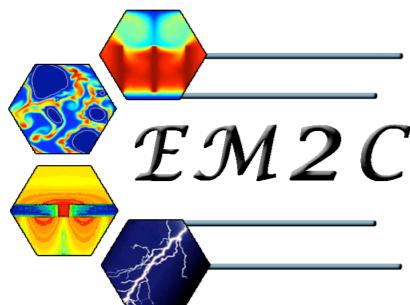


Combustion for Low Emission Application of Natural Gas



**Modelling, accuracy and impact of radiative heat transfer in
the coupled simulation of an atmospheric premixed flame
confined by semi-transparent viewing windows.**

Lorella Palluotto, Ronan Vicquelin, Olivier Gicquel

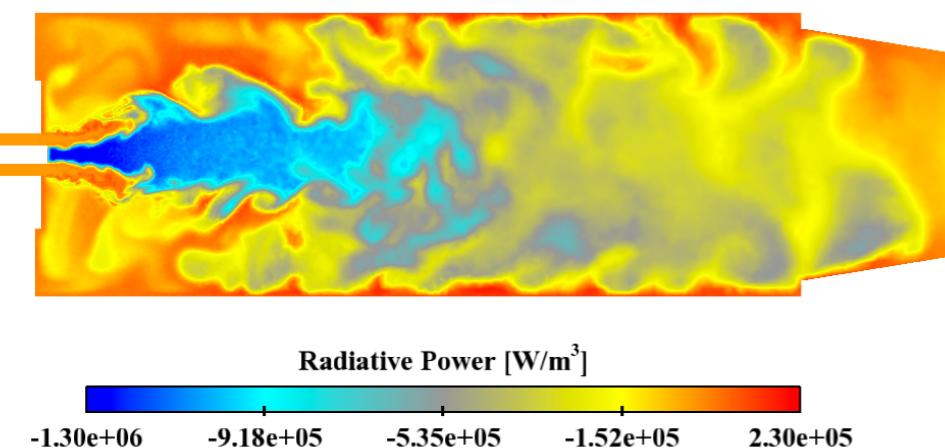
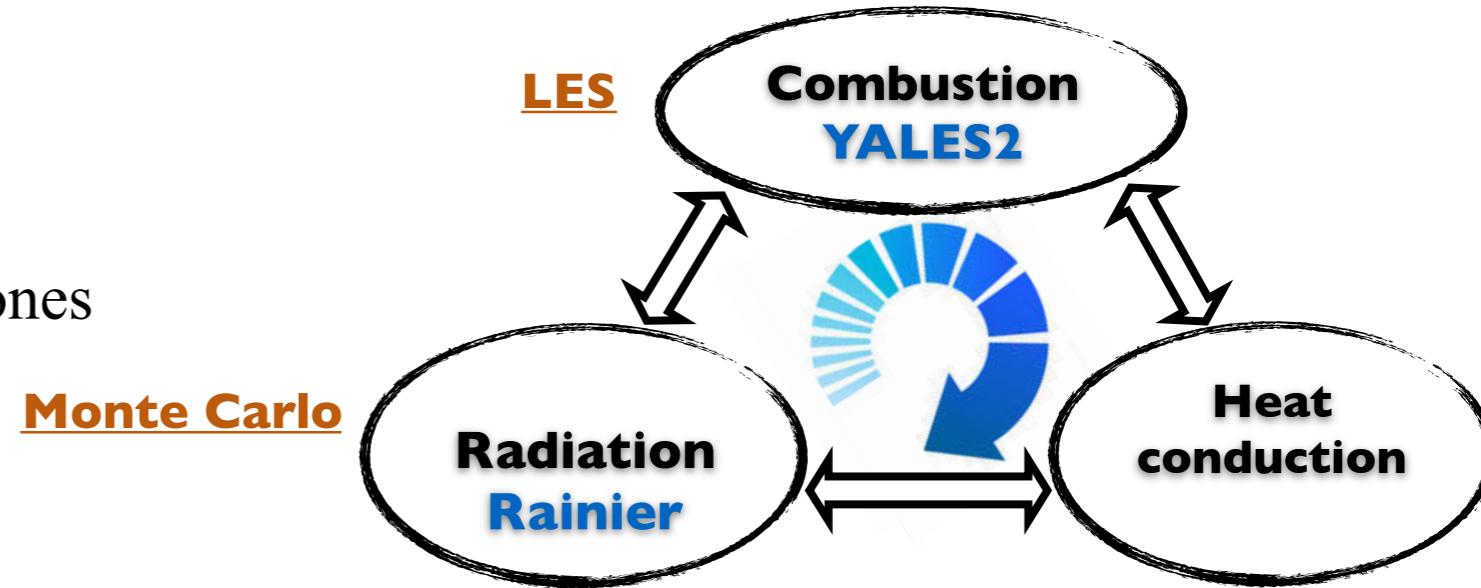


Context

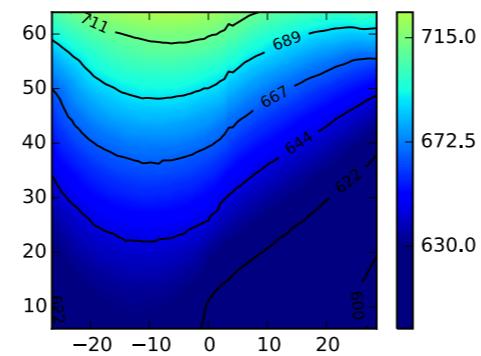
Importance of radiative heat transfer in combustion applications

Semi-industrial burner

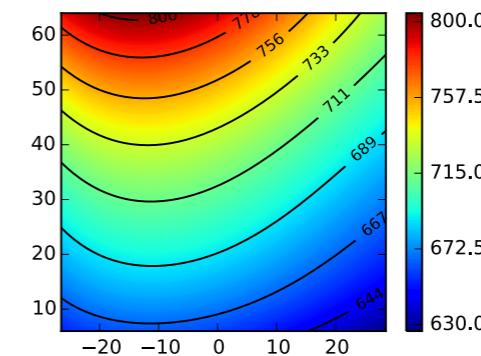
- Fully coupled methodology
- Radiative fluxes same order than convective ones
- Wall temperature prediction



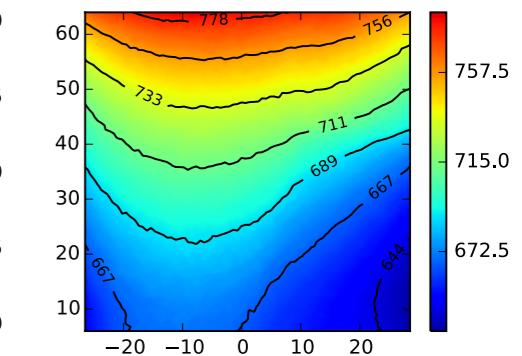
No radiation



Exp



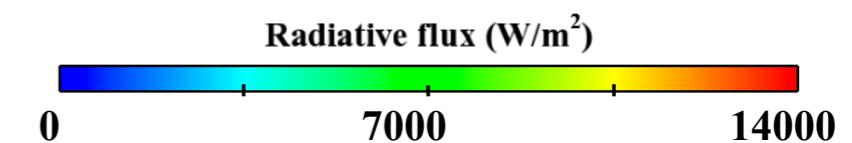
With radiation



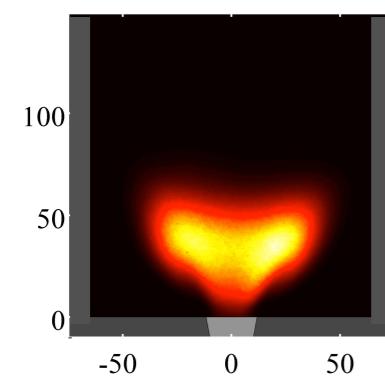
Importance of radiative heat transfer in combustion applications

Oxy-combustion:

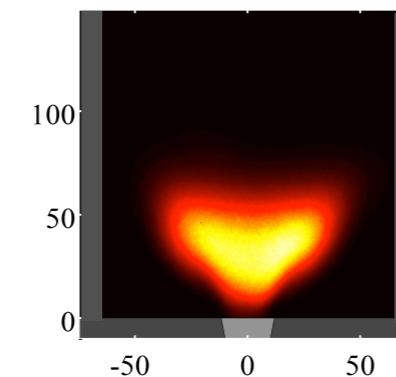
- Highly CO₂-concentrated exhaust gases
- Facilitate the CO₂ capture process
- No NOx



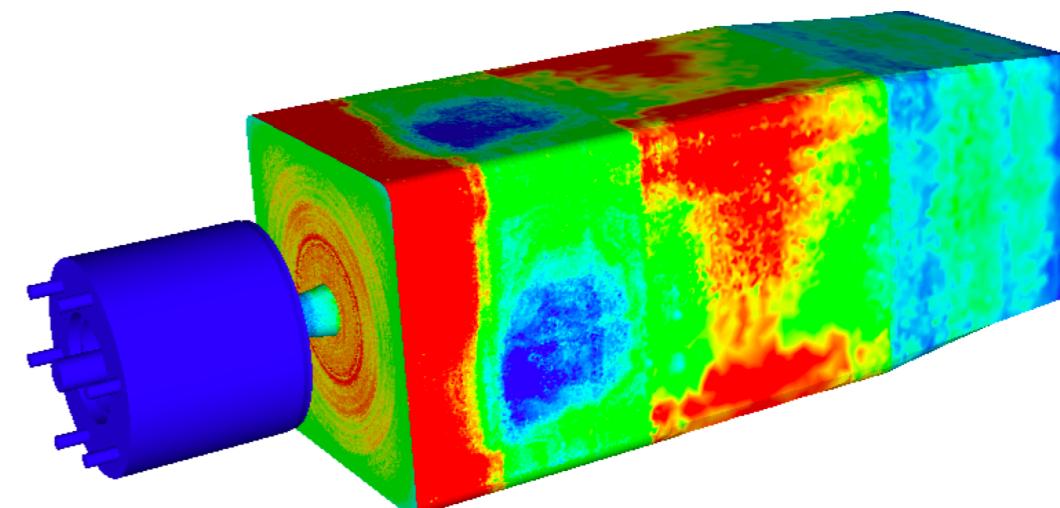
OXYtec chamber^[1] at EM2C lab



Flame A
CH4/Air



Flame B
CH4-O₂-CO₂



Radiative Fluxes = 20% of Pth

High CO₂ concentration

→ Radiation is significative

CH4-O₂-CO₂

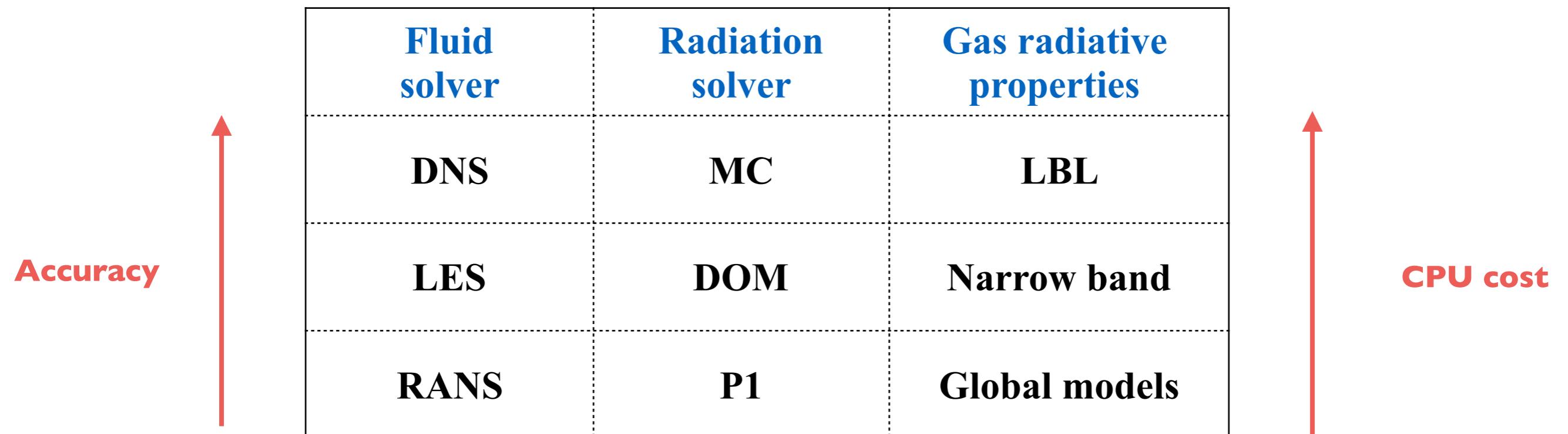
Radiative flux may be higher

Burnt gases	CO ₂	H ₂ O
A	0.14	0.12
C	0.83	0.12

Simulations in progress...

Coupled simulations: state of the art

Commonly used approaches for radiation and combustion



Coupled simulations: state of the art

References	Fluid solver	Radiation solver	Gas rad properties	Application
<i>Adams and Smith 1995</i>	RANS	DOM	Gray Gas	Confined turbulent sooting flame
<i>Coelho et al. 2003</i> <i>Habibi et al. 2007</i> <i>Wang et al. 2008</i>	RANS	DOM	Global	Turbulent jet flame Steam cracking furnace Turbulent jet flame
<i>Snegirev et al. 2004</i> <i>Mehta et al. 2009</i>	RANS	MC	Global	Turbulent flames Turbulent sooting flames
<i>Tessé et al. 2004</i>	RANS	MC	CK	Turbulent sooting flame
<i>Zhao et al. 2013</i>	RANS	MC	LBL	Oxy-combustion
<i>Poitou et al. 2012</i> <i>Berger et al. 2016</i>	LES	DOM	Global	Confined turbulent flame Helicopter combustion chamber
<i>Jones and Paul et al. 2005</i>	LES	DOM	Gray Gas	Gas turbine combustor
<i>Gupta et al. 2013</i>	LES	MC	LBL	Turbulent jet flame (1M cells)
<i>Koren et al. 2018</i> <i>Rodrigues et al. 2018</i>	LES	MC	CK	Laboratory-scale combustor
<i>Dos Santos et al. 2008</i>	LES	Ray tracing	CK	2D burner
<i>Wu et al. 2005</i>	DNS	MC	Gray Gas	1D turbulent flame
<i>El Hafi et al. 2002</i> <i>Zhang et al. 2013</i>	DNS	MC	CK	1D flame Turbulent channel flow

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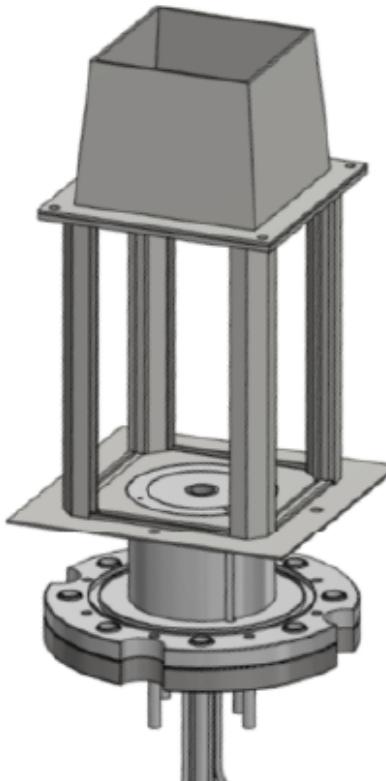
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Objectives

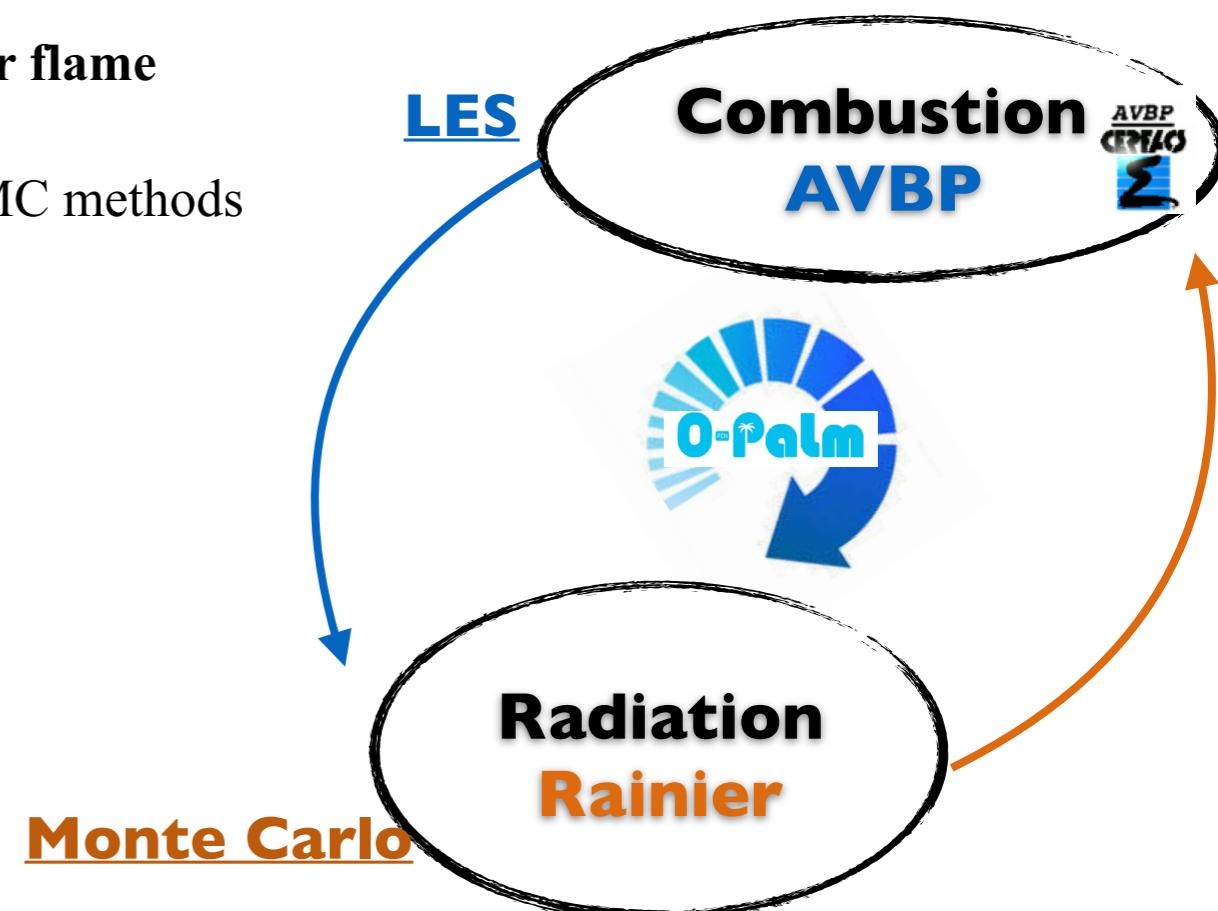
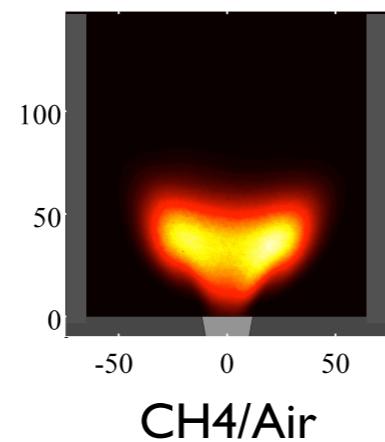
Quantification of thermal effects in air-methane and oxy-methane flames

Complex target application



Coupled simulations of a CH₄-Air flame

LES of turbulent reactive flows + MC methods



Fluid solver	Radiation solver	Gas radiative properties
LES	MC	Narrow band

Outline

LES of a premixed swirled flame

- *Presentation of Oxytec chamber*
- *LES results*

Coupled simulations: numerical set-up

- *Monte Carlo solver*
- *CPU cost-accuracy trade off*

First results of coupled simulations

- *Impact of spectral properties of quartz*
- *Radiative heat transfer in a methane air flame*

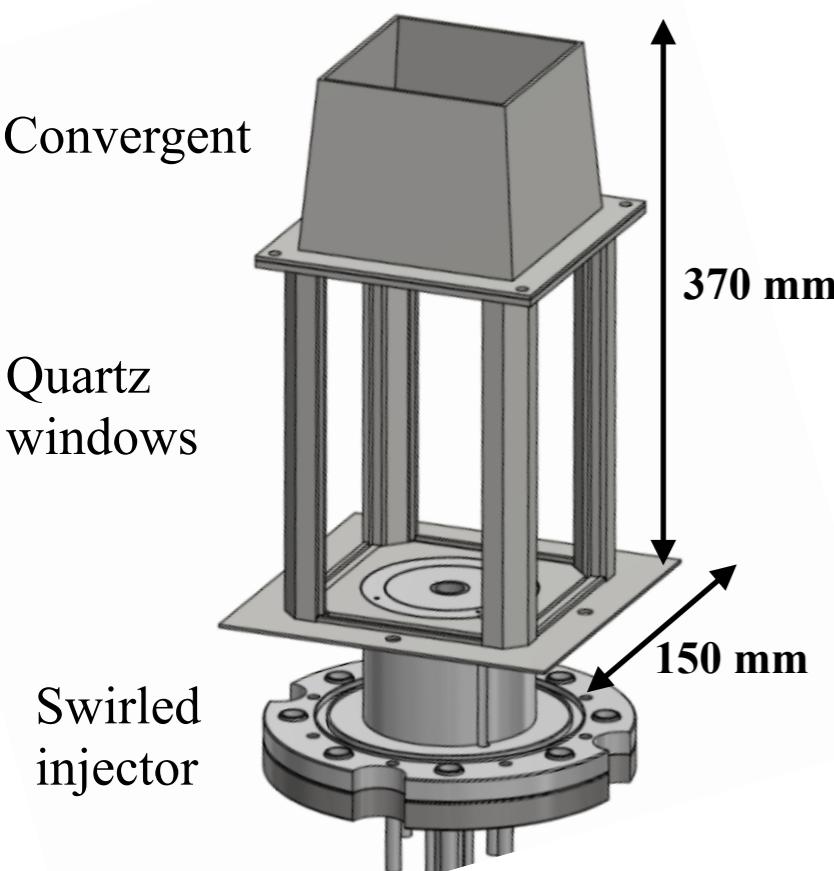
AIR LIQUIDE/ANR Oxytec chamber

Experimental rig at EM2C

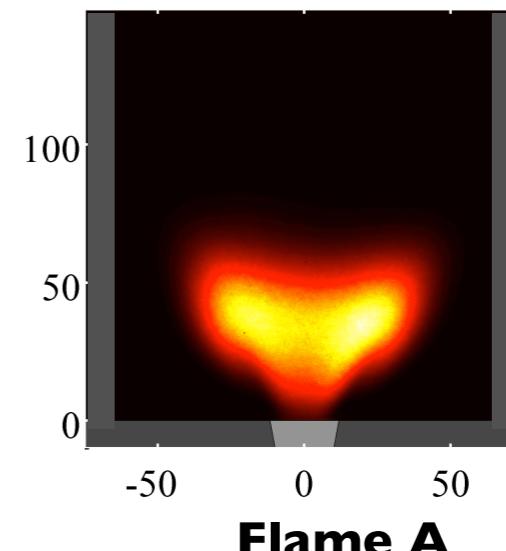


Target application: OxyTec chamber

- Experimental rig.^[1,2,3,4] located at EM2C lab
- Premixed swirled CH₄-Air flame



OH-chemiluminescence images^[2]



Y_{CH4}	Y_{O2}	Y_{N2}	S_L (m/s)	T_{ad} (K)	P (kW)	Eq.Ratio	Swirl number
0.05	0.22	0.73	0.36	2186	14	0.95	0.85

¹ Jourdaine, P., Mirat, C., Beaunier, J., Caudal, J., Joumani, Y., & Schuller, T. (2016, June). ASME Turbo Expo 2016

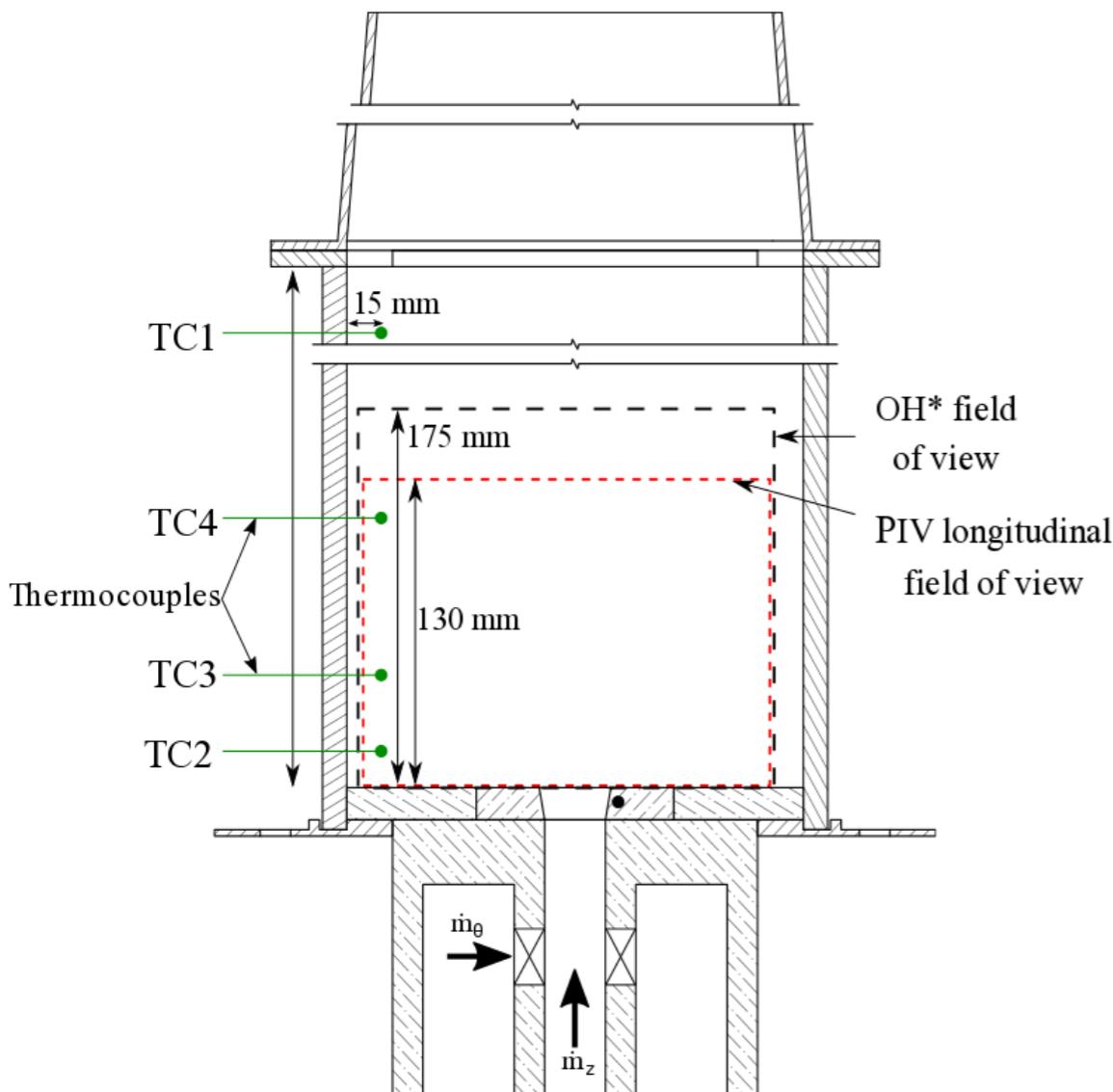
² P. Jourdaine, C. Mirat, J. Caudal, T. Schuller. Stabilization mechanisms of swirling premixed flames with an axial-plus-tangential swirler. ASME 2017

³ Jourdaine, P., Mirat, C., Caudal, J., & Schuller, T. (2018). Journal of Engineering for Gas Turbines and Power, 140(8), 081502.

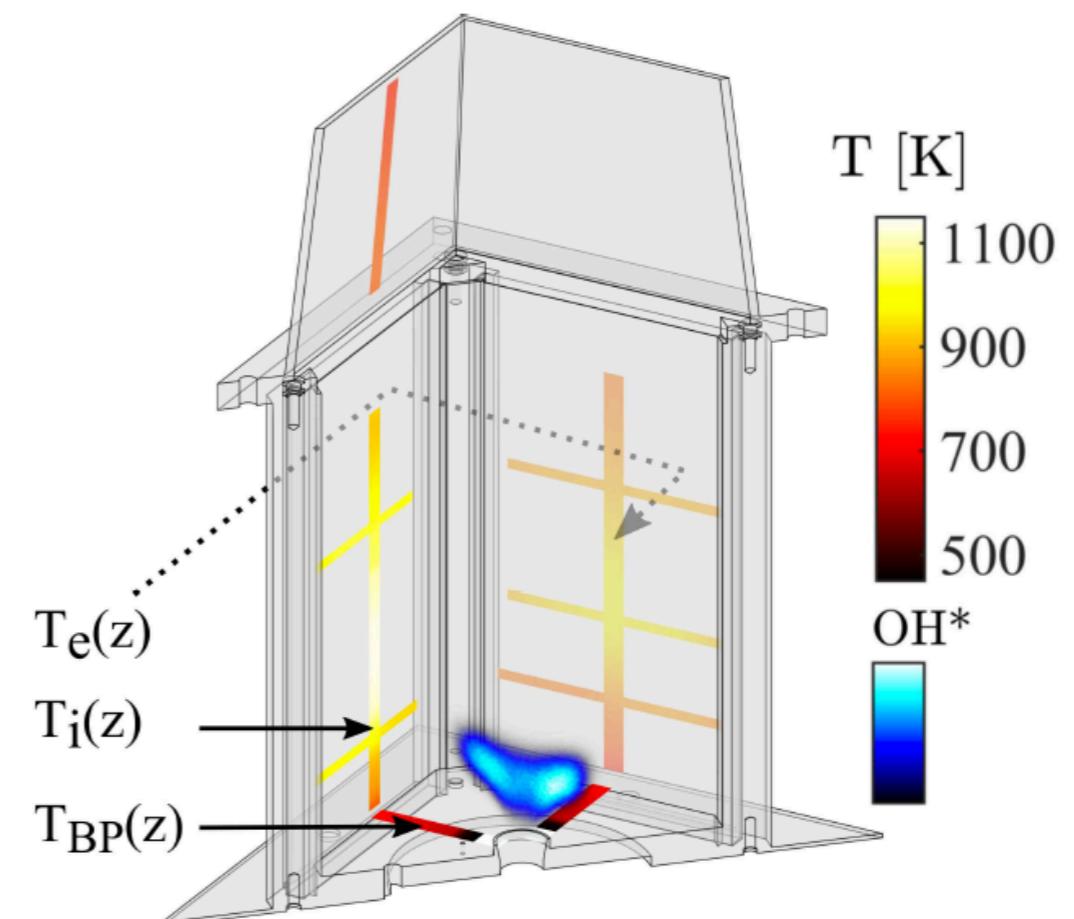
⁴ A. Degenève, P. Jourdaine, C. Mirat, J. Caudal, R. Vicquelin, and T. Schuller. 2nd International Workshop on Oxy-Fuel Combustion, Bochum, Germany, (2018)

AIR LIQUIDE/ANR Oxytec chamber Measurements

- **Velocity:** PIV, LDV
- **Temperature:** thermocouples, LIP
- **Flame:** OH-chemiluminescence, OH-PLIF



LIP measurements from A. Degenèvre^[1]



Large Eddy Simulations

AVBP (CFD code developed by CERFACS and IFPEN)

- 3-D compressible Navier-Stokes
- Unstructured and hybrid grids
- Massively parallel

Case Set-up

- Computational grid:
 - 50 millions cells
- Numerical scheme:
TTGC (Two-Step Taylor Galerkin 'C' - third order accurate in space and time)
- LES subgrid model: Sigma model^[1]
- Combustion model: Thickened Flame model for LES^[2]
 - 2-step mechanism^[3]
 - Dynamic thickening, Charlette efficiency model^[4]

1 : Nicoud, F., Toda, H. B., Cabrit, O., Bose, S., and Lee, J "Using singular values to build a subgrid-scale model for large eddy simulations." *Physics of Fluids*. 2011

2 : O. Colin et al., A Thickened flame model for large eddy simulations of turbulent premixed combustion, *Physics of Fluids* (2000)

3 : G. Boudier, Methane/air flame with 2-step chemistry: 2s ch4 cm2, Tech. report, CERFACS, 2007

4 : F. Charlette et al., A power-law flame wrinkling model for LES of premixed turbulent combustion. *Combustion and Flame* (2002)

5 : A. Degenève, P. Jourdaine, C. Mirat, J. Caudal, R. Vicquelin, and T. Schuller. 2nd International Workshop on Oxy-Fuel Combustion, Bochum, Germany, (2018)

Large Eddy Simulations



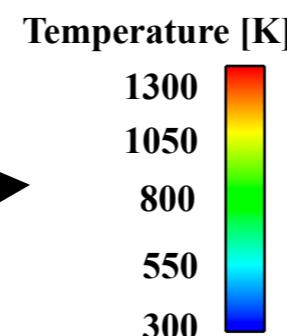
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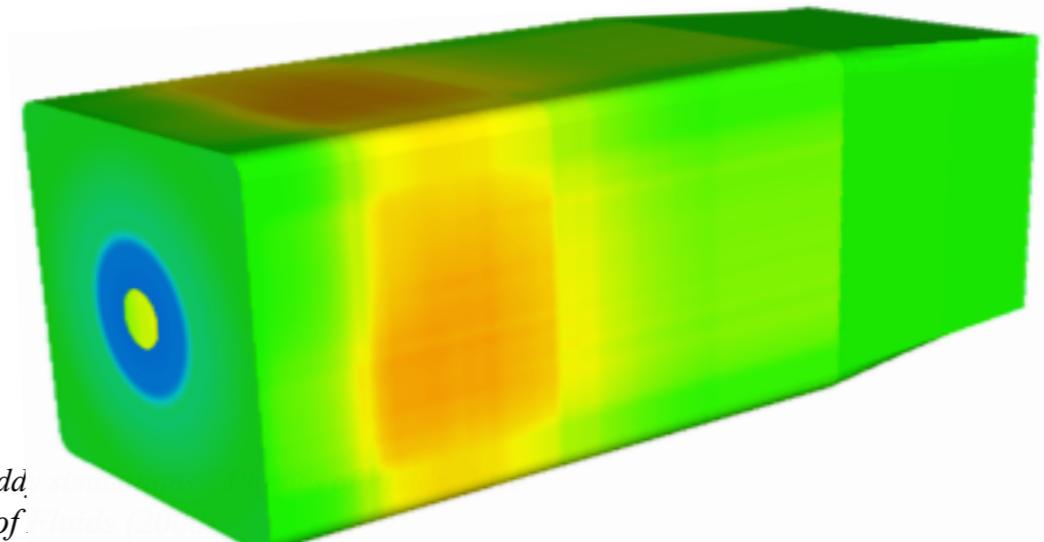
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- Walls:
 - Imposed temperature



Laser Induced Phosphorescence (LIP)^[5]



1 : Nicoud, F., Toda, H. B., Cabrit, O., Bose, S., and Lee, J "Using singular values to build a subgrid-scale model for large eddies" (2002)

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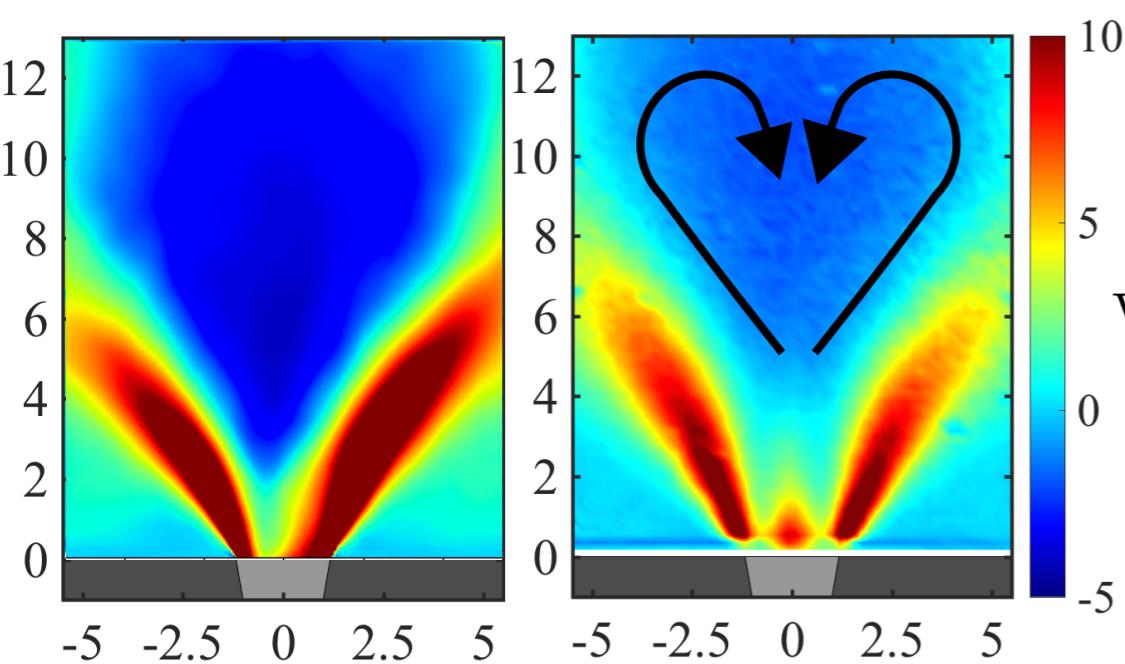
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Oxytec chamber: Validation

Axial velocity

LES

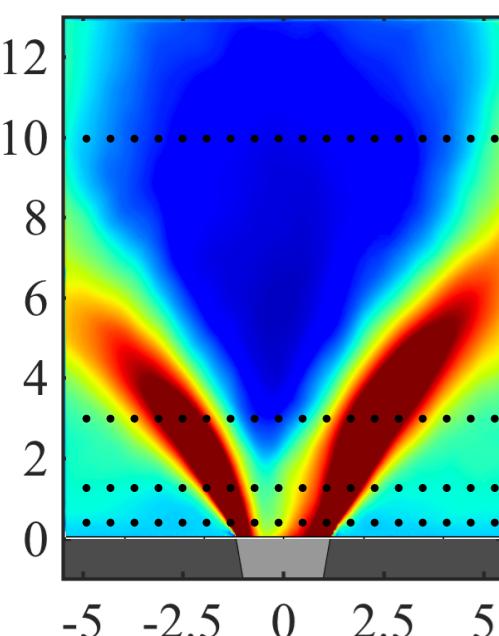


Experiments

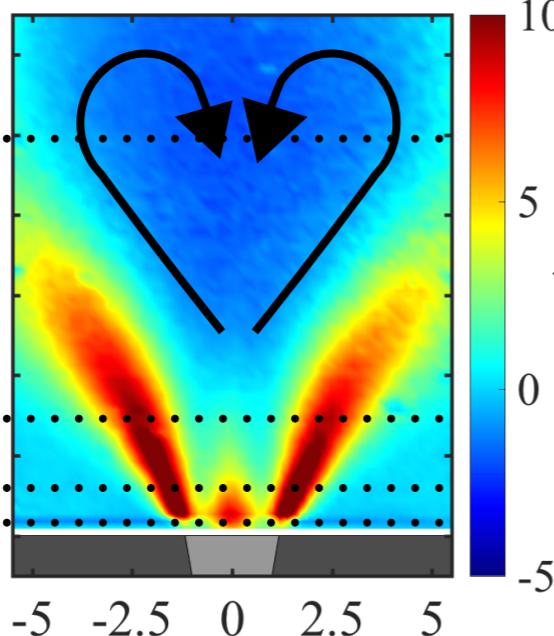
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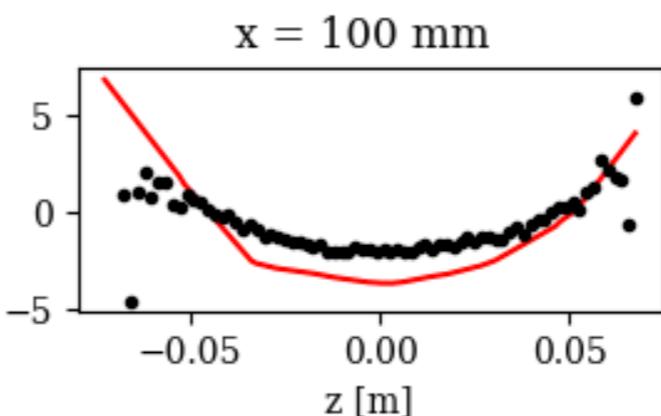
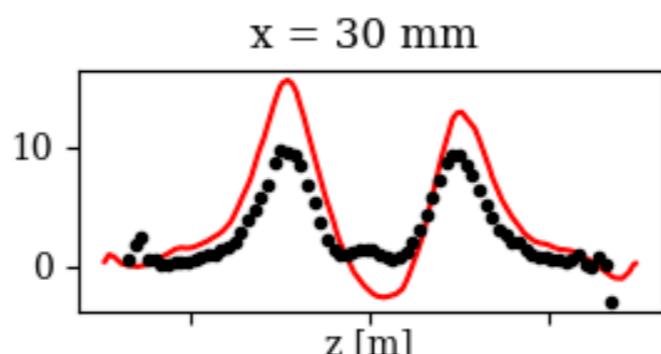
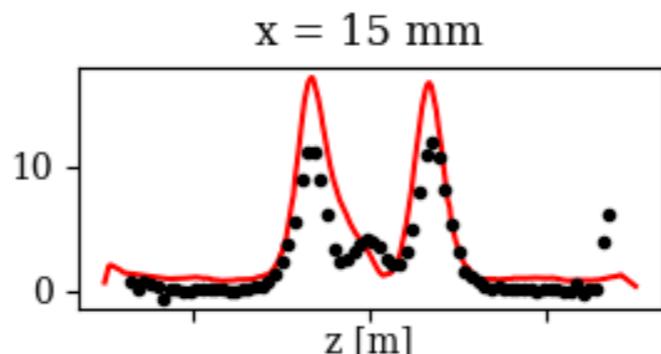
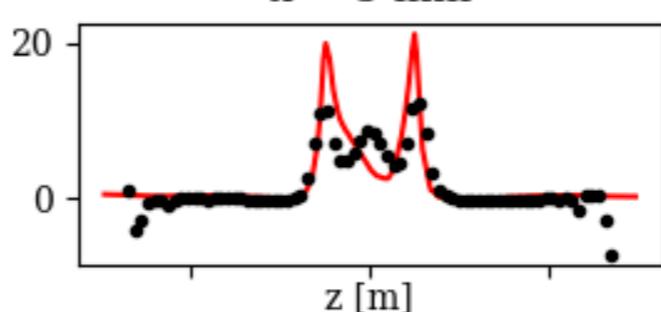
LES



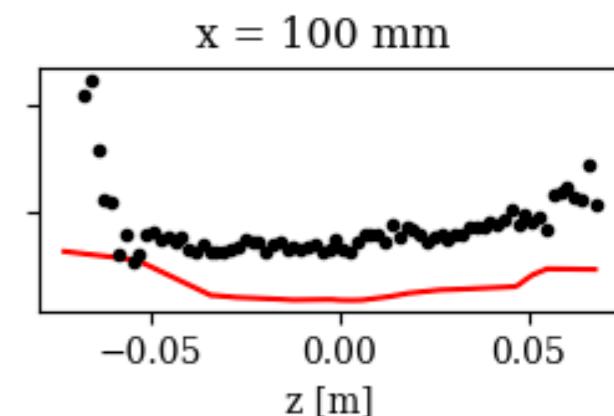
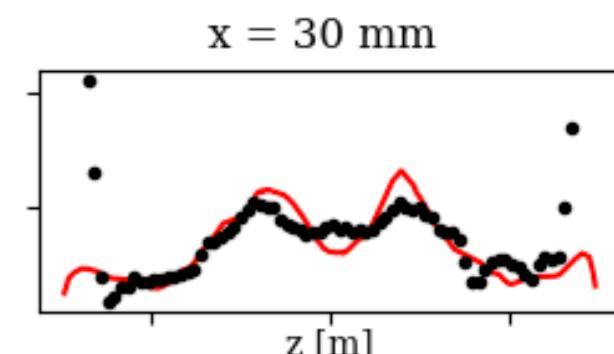
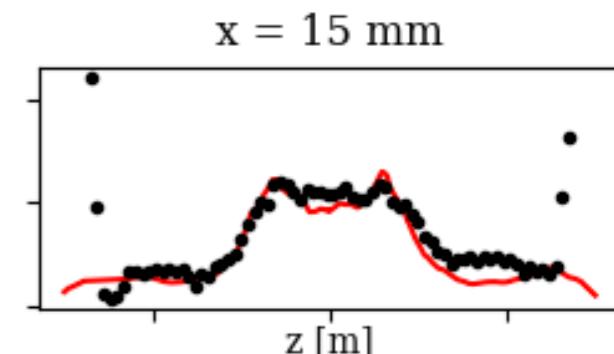
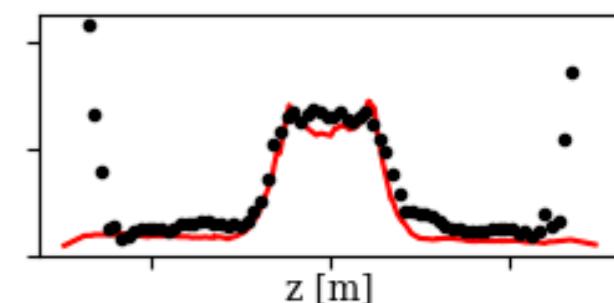
Experiments



— LES
.... PIV Axial velocity [m/s]



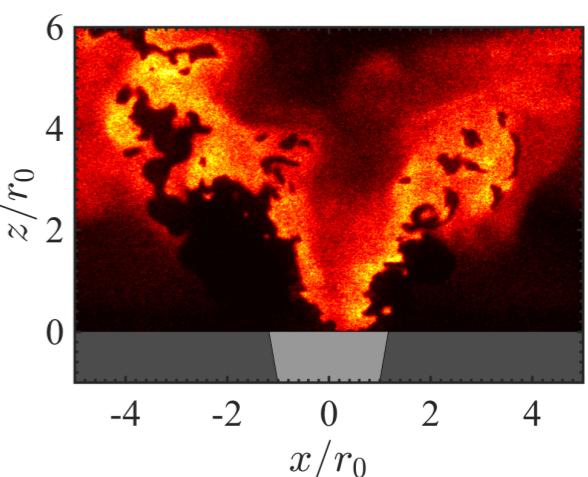
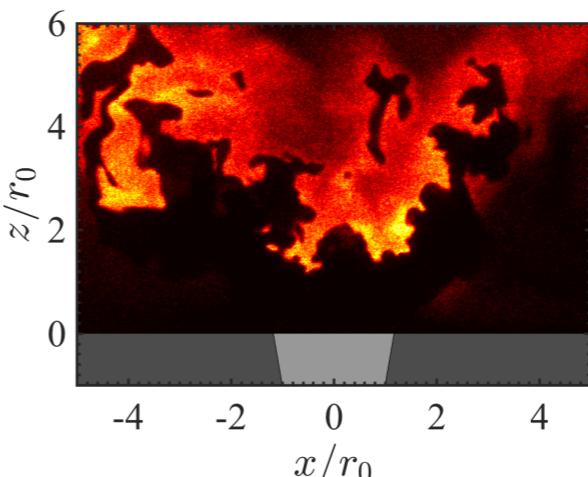
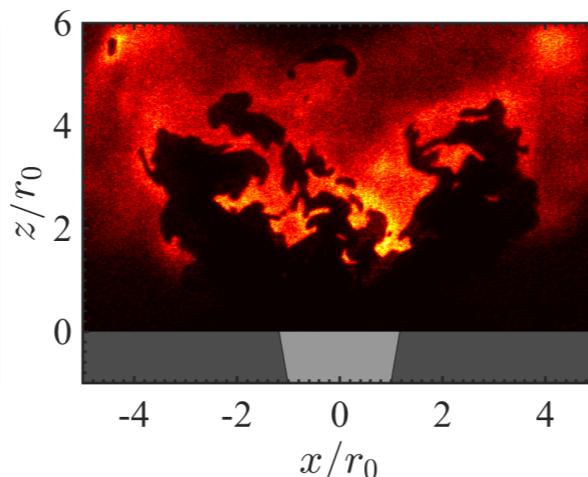
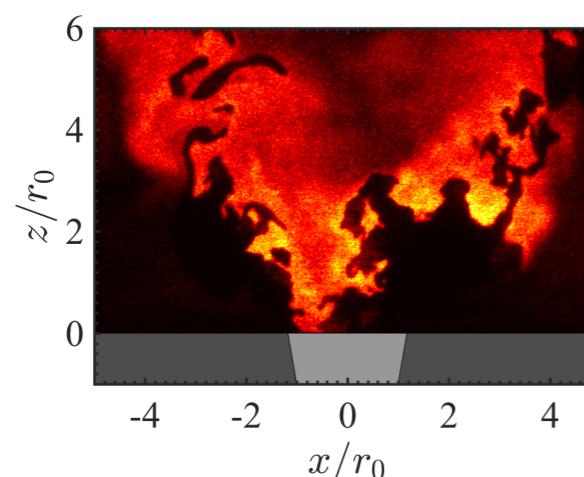
RMS of Axial velocity [m/s]
x = 5 mm



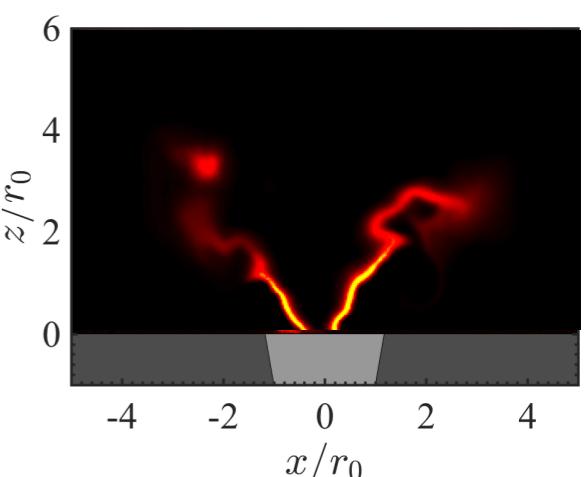
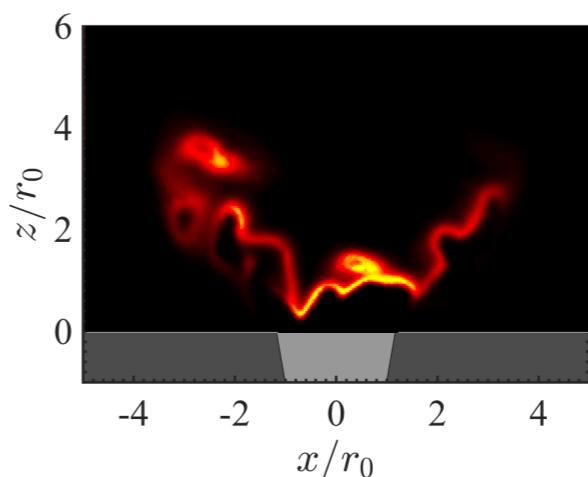
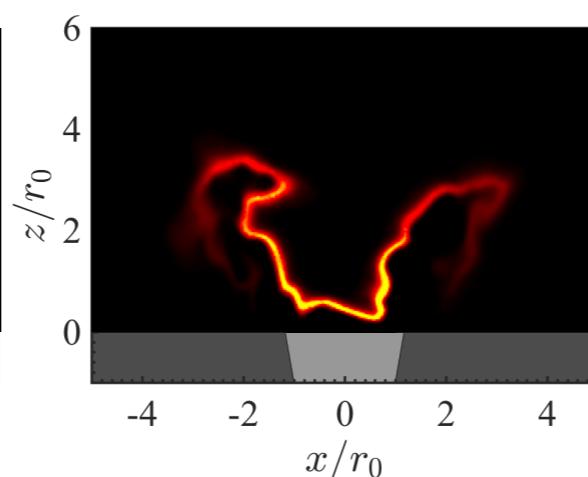
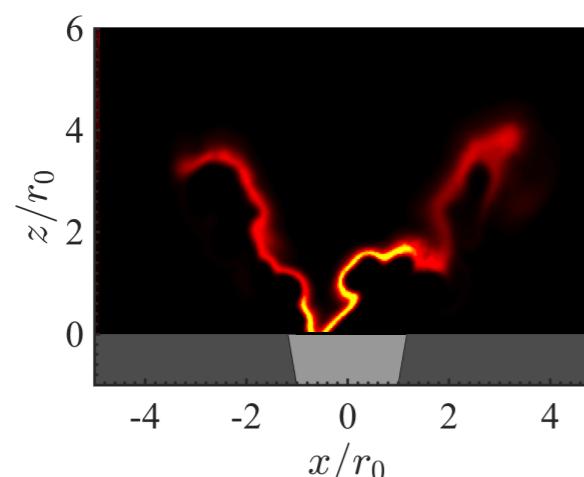
- Position of stagnation point not retrieved
- Central velocity peak not retrieved

Oxytec chamber: flame front

Exp
OH-PLIF



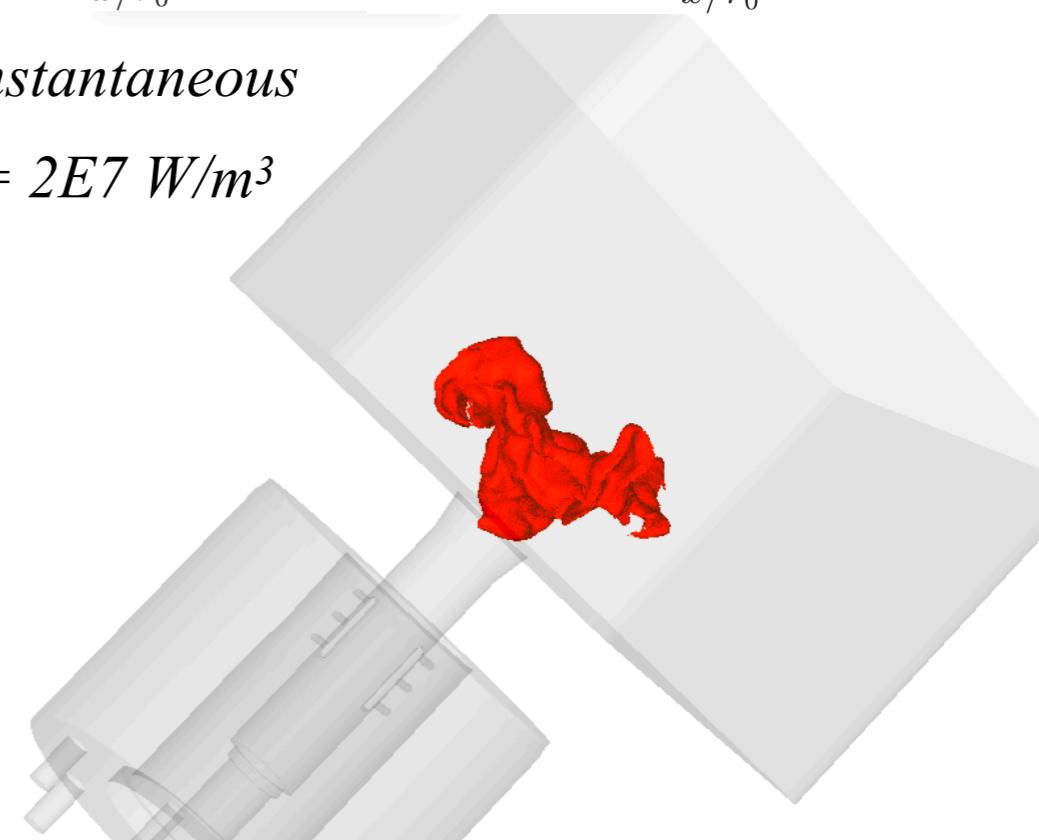
LES



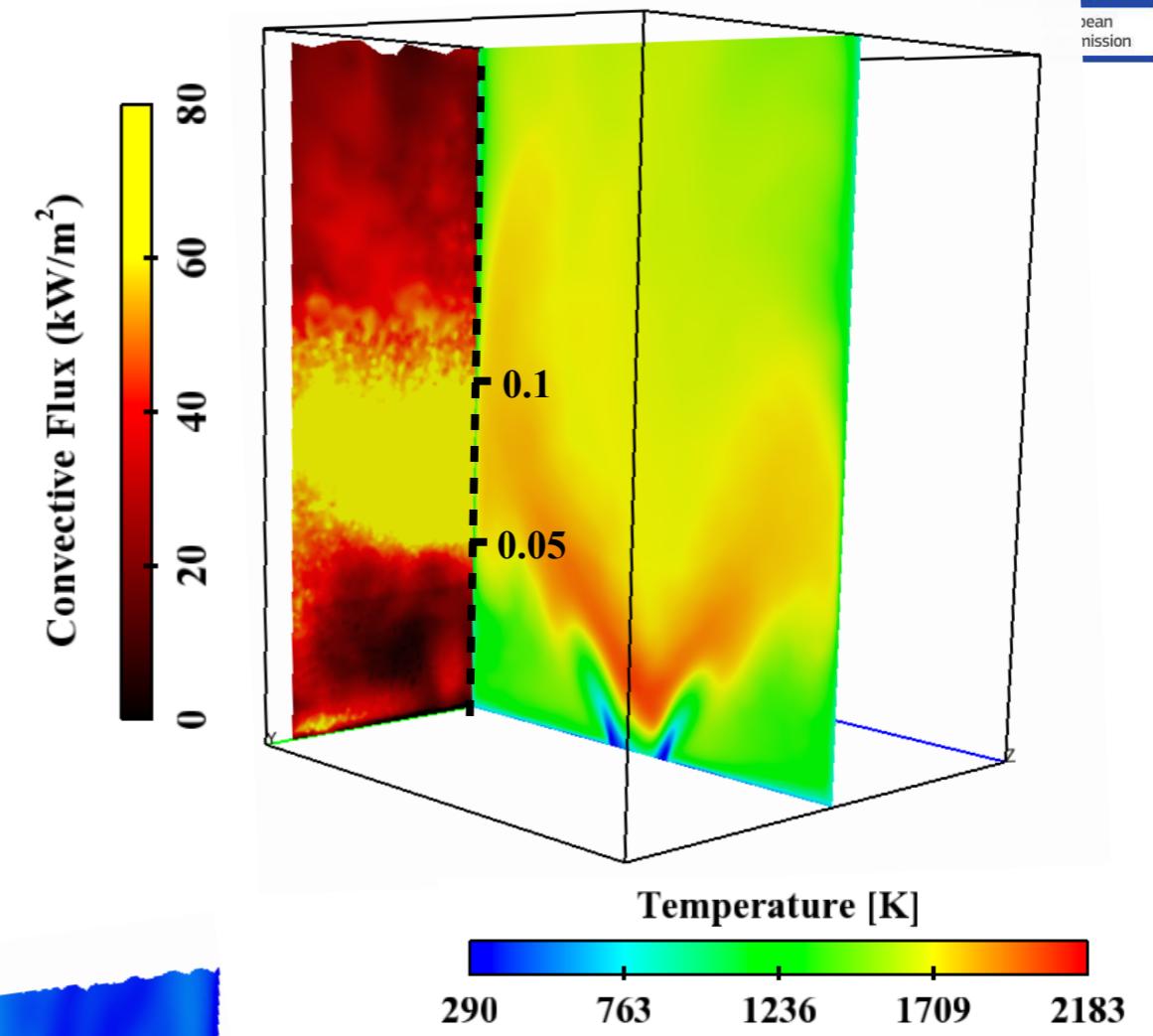
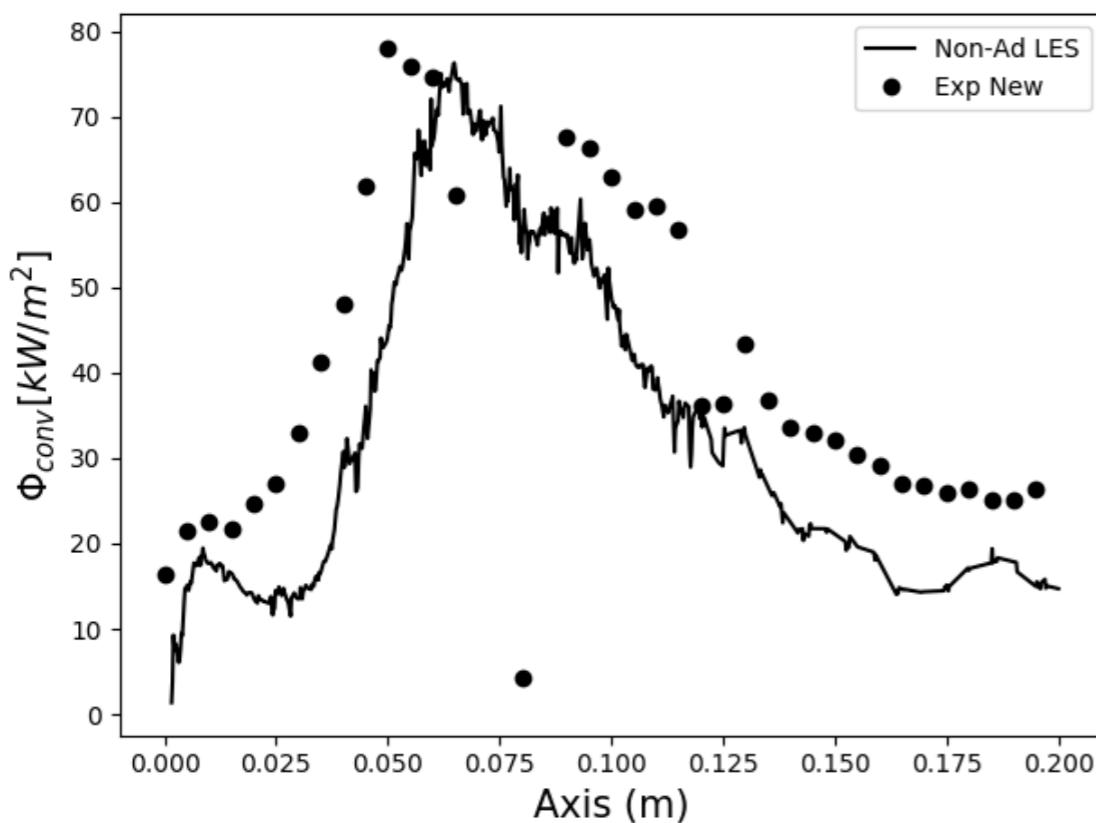
Instantaneous HR-field

*Iso-surface of instantaneous
Heat Release = $2E7 \text{ W/m}^3$*

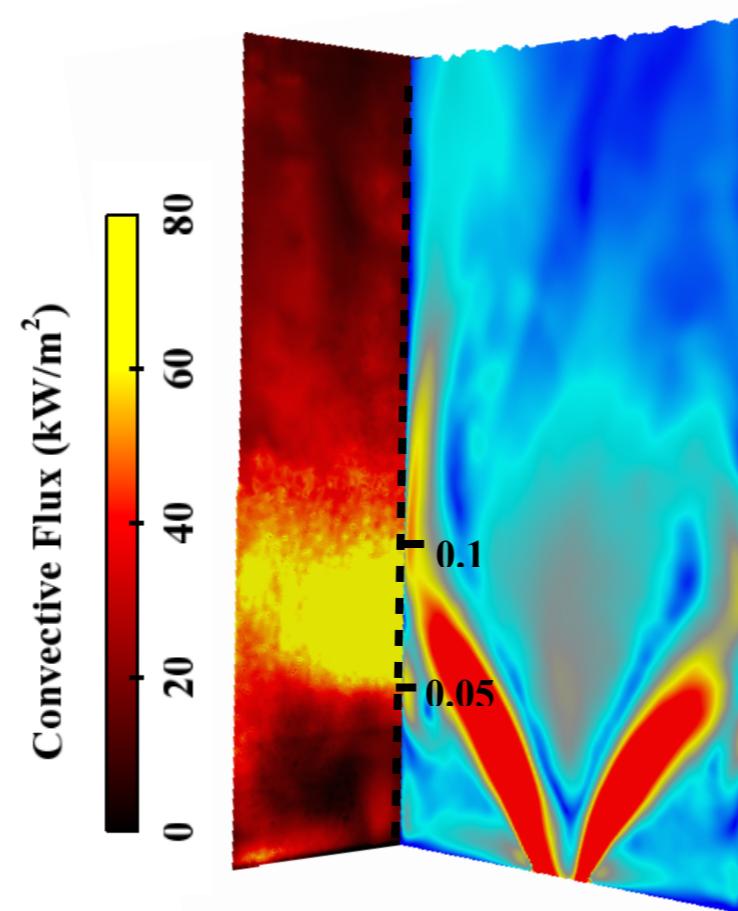
Highly fluctuating flame



Wall heat fluxes



- Good agreement
- Numerical fluxes are underestimated in some regions



Outline

LES of a premixed swirled flame

- *Presentation of Oxytec chamber*
- *LES results*



Conclusions

- Some discrepancy about:*
- *Flow topology*
 - *Conductive fluxes*

Coupled simulations: numerical set-up

- *Monte Carlo solver*
- *CPU cost-accuracy trade off*

First results of coupled simulations

- *Impact of spectral properties of quartz*
- *Radiative heat transfer impact on a methane air flame*

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Coupled simulations: numerical set-up

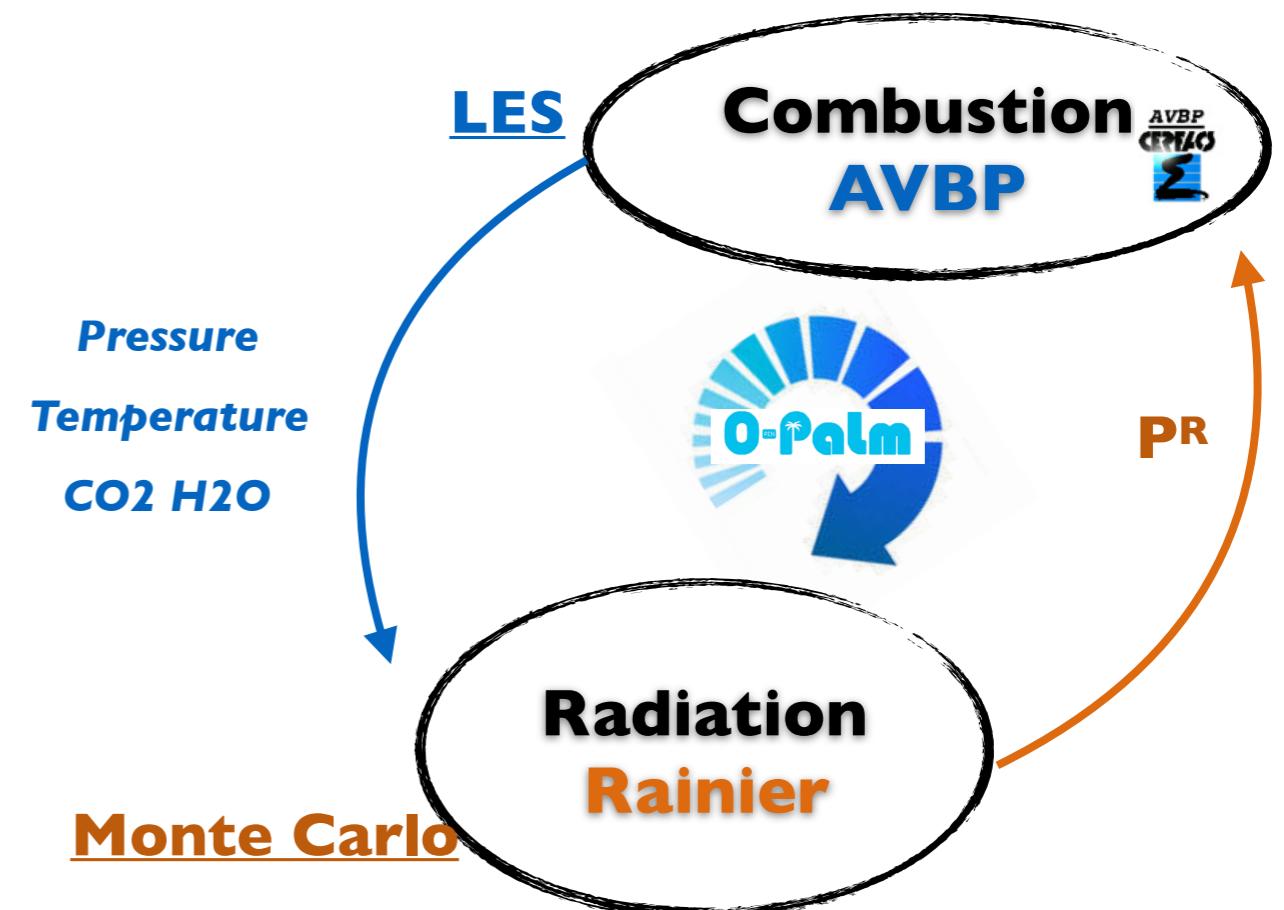
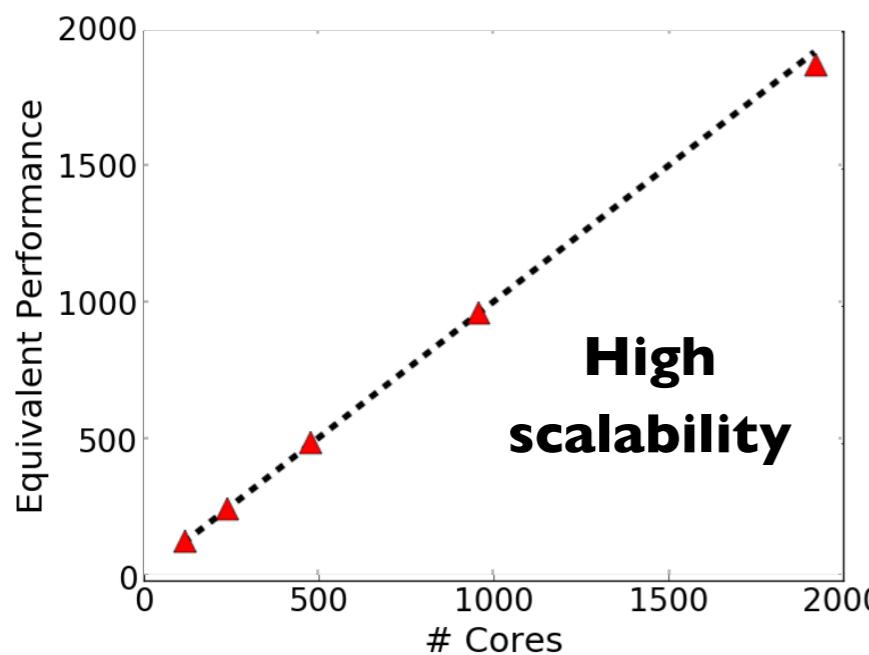
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Radiation solver: Rainier

- Parallel code that solves the RTE
- Non-scattering medium
- SGS TRI are not taken into account
- **Quasi-Monte Carlo** method [1]:
Emission-based Reciprocity Method (**ERM**) [2]
- Accurate model for gas radiative properties:
correlated c-k distribution[3]



Radiative source term in energy conservation equation: **PR**

$$\mathbf{PR} = \int_0^{\infty} \int_{4\pi} \kappa_{\nu} I_{\nu} d\Omega d\nu - 4\pi \int_0^{\infty} \kappa_{\nu} I_{\nu}^0(T) d\nu$$

Pabsorbed **Pemitted**

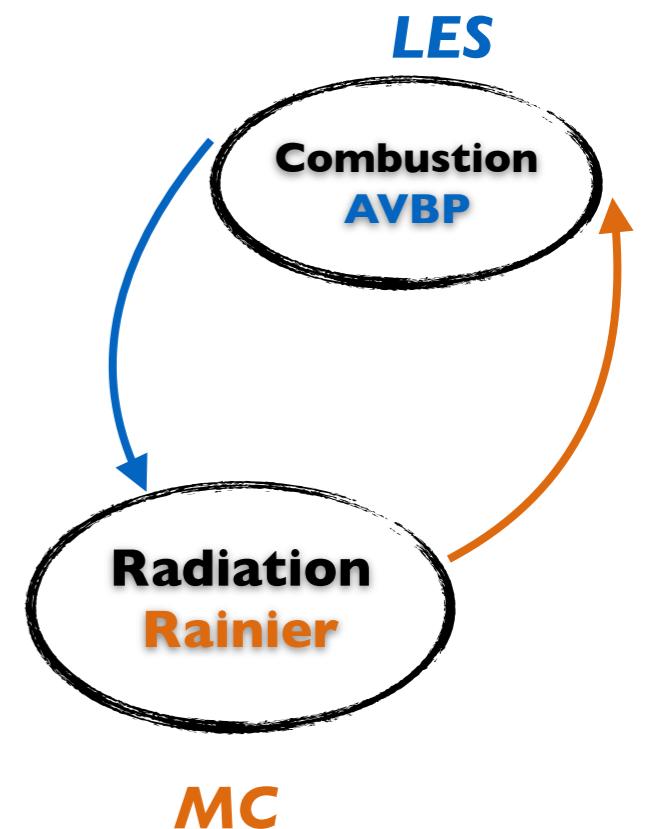
1 : Palluotto, L., Dumont, N., Rodrigues, P., Koren, C., Vicquelin, R., & Gicquel, O. 2017. « Comparison of Monte Carlo Methods Efficiency to Solve Radiative Energy Transfer in High Fidelity Unsteady 3D Simulations. » ASME Turbo Expo 2017

2 : L. Tessé, F. Dupoirieux, B. Zamuner and J. Taine, International Journal of Heat and Mass Transfer

3: Taine, J., and Soufiani, A., 1999. "Gas radiative properties: From spectroscopic data to approximate models". Vol. 33 of Advances in Heat Transfer . Elsevier, pp. 295 – 414.

Questions:

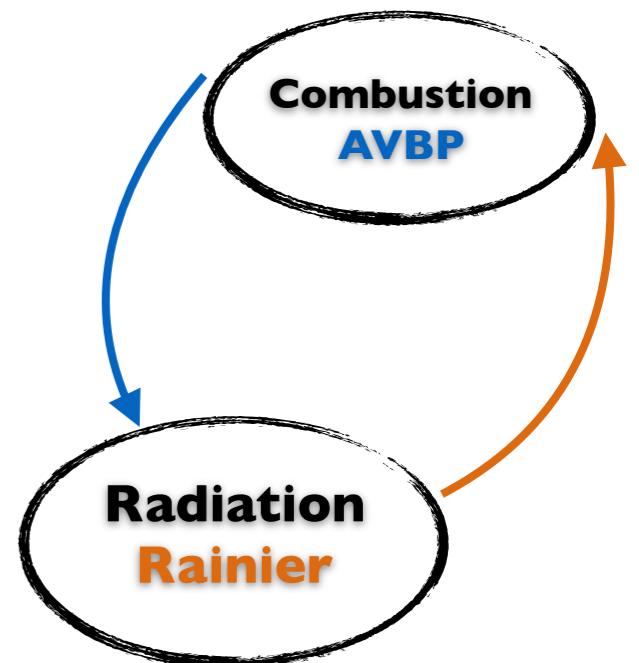
- Computational grid for radiative heat transfer simulation?
- Mesh convergence
- How often the solvers exchange data?
 - Coupling error estimation



CPU cost- accuracy trade off need to be found

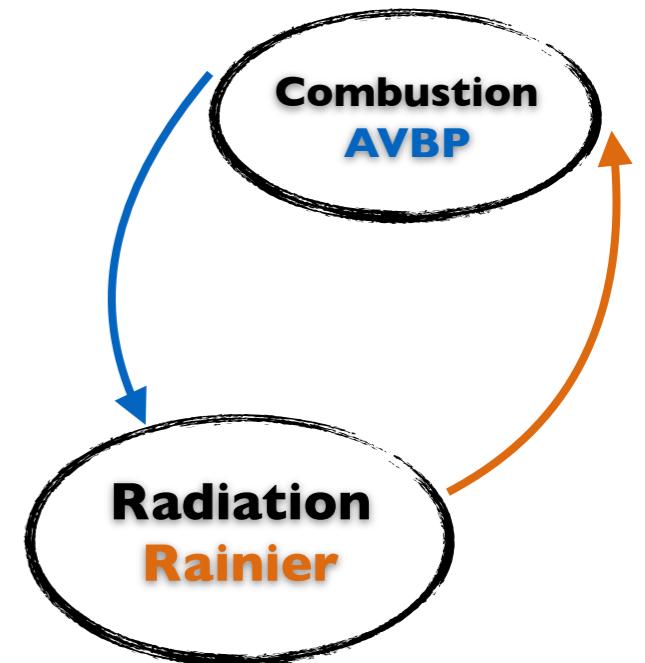
Coupled Simulations: Mesh Convergence

Mesh LES	Mesh nodes (millions)
Mesh I	9



Coupled Simulations: Mesh Convergence

Mesh LES	Mesh nodes (millions)
Mesh I	9



Computational grid for radiative heat transfer simulation?

$$e_1 = \frac{\Phi_{rad} - \Phi_{rad,Sc1}}{\Phi_{rad,Sc1}}$$

Mesh Rainier	Mesh nodes (millions)	e ₁ (%)	CPU time ratio
Mesh I	9	-	-
Mesh 2	4	1.7 %	3
Mesh 3	2.7	2.2 %	5.6

Coupled Simulations: Choice of Coupling Frequency

How often LES and radiation solvers exchange data?

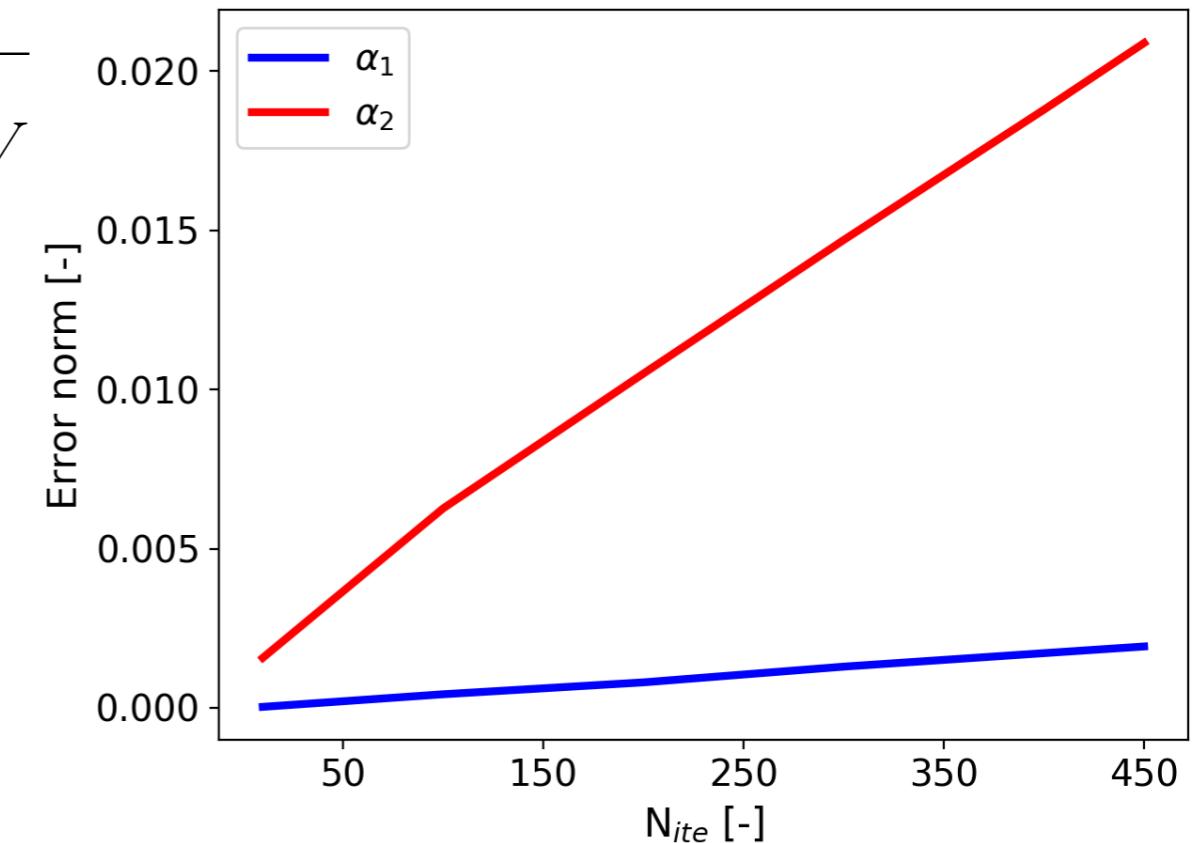
Error evaluated in terms of L2 error norm of temperature and radiative power

$$\alpha_1(N) = \sqrt{\int_V |T_N - T_0|^2 dV} / \sqrt{\int_V |T_0|^2 dV}$$

$$\alpha_2(N) = \sqrt{\int_V |P_N^R - P_0^R|^2 dV} / \sqrt{\int_V |P_0^R|^2 dV}$$

$_0$ = Variable at reference iteration 0

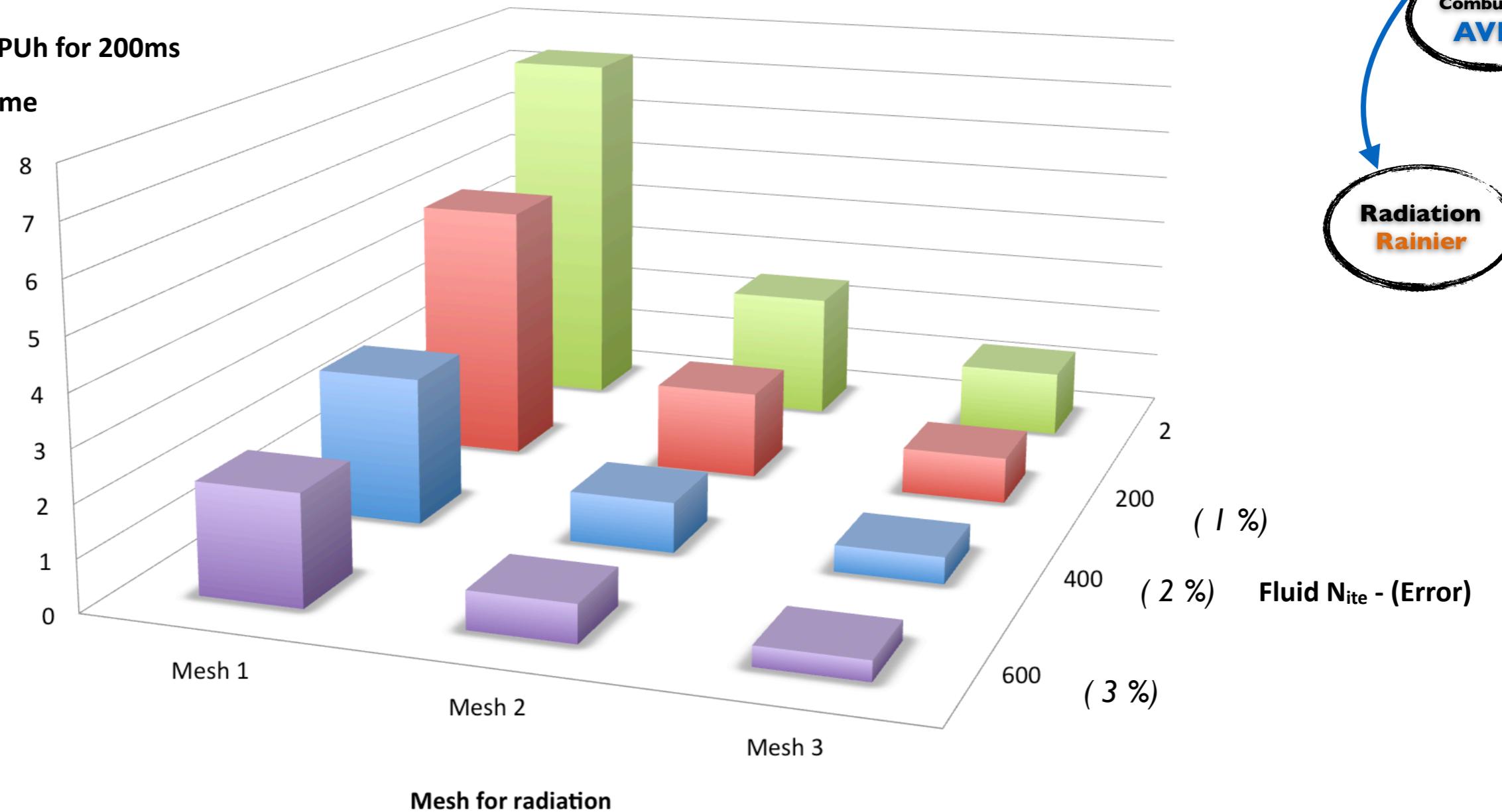
N = Variable at iteration N of the fluid solver



Coupled Simulations:

Trade-off CPU cost - accuracy

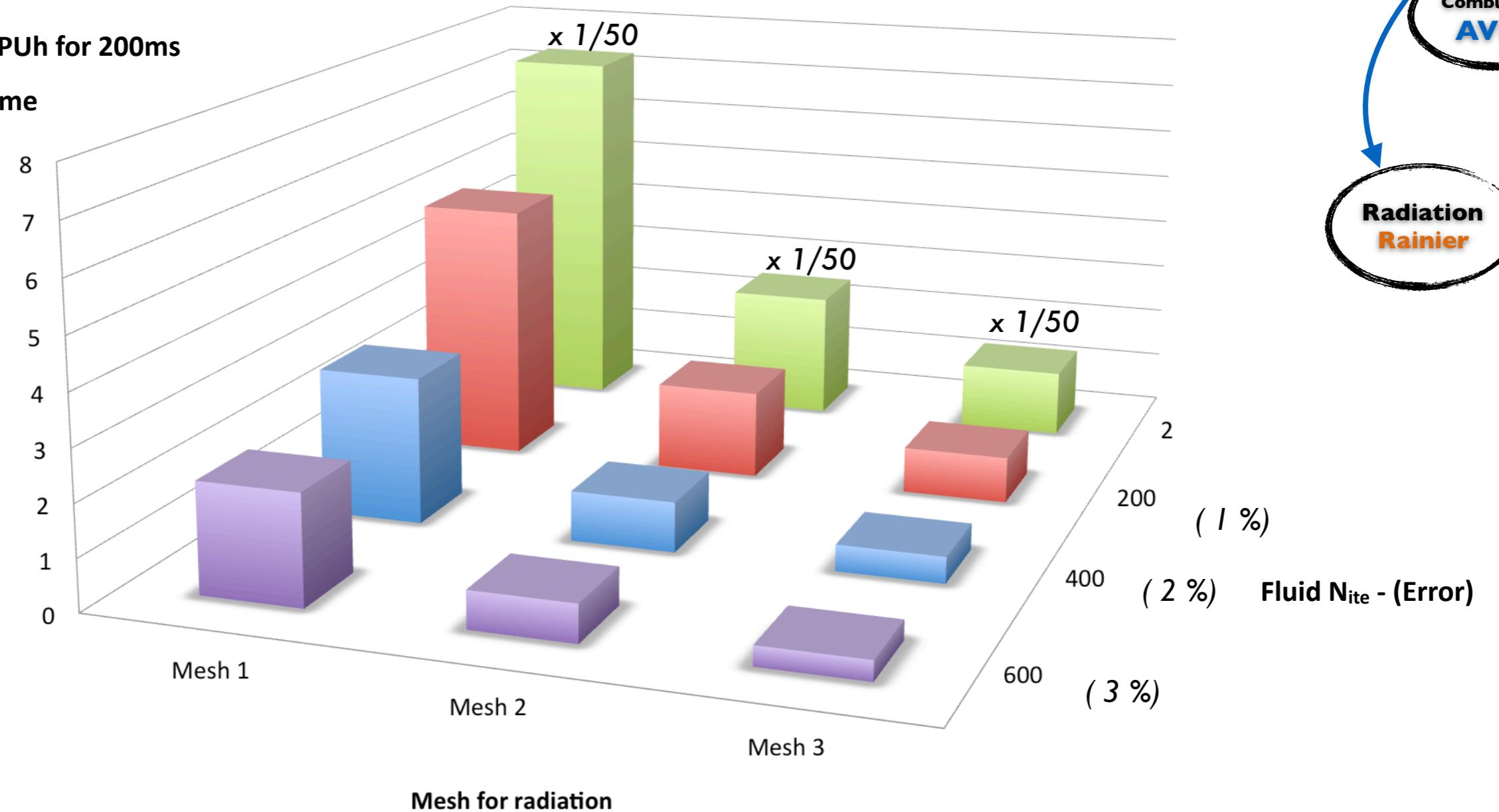
Millions of CPUh for 200ms
of physical time



Coupled Simulations:

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Coupled Simulations:

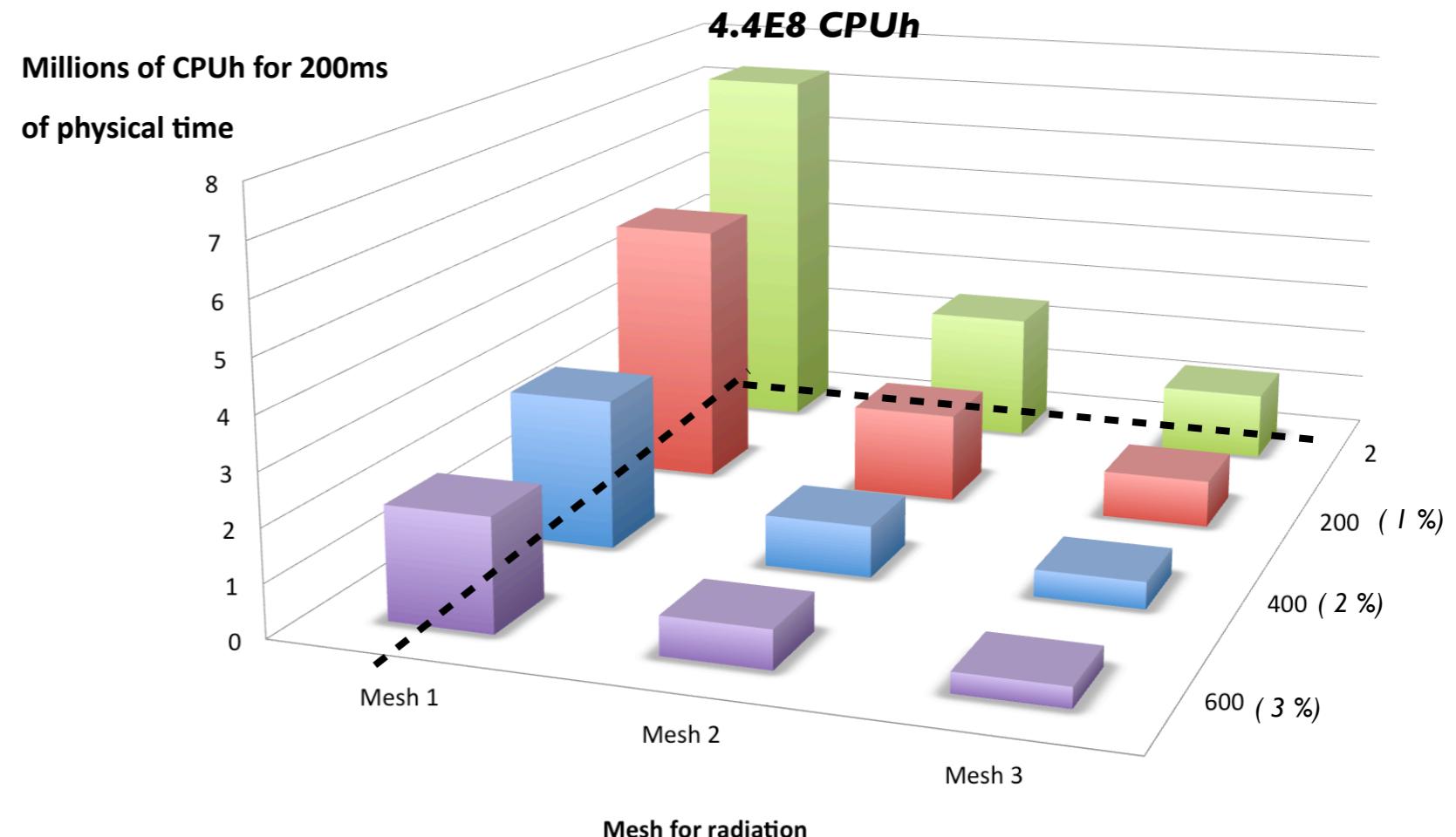
Trade-off CPU cost - accuracy

Scenarios

Optimistic:

Mesh 1 (9M nodes)

Coupling every iteration of fluid solver



Coupled Simulations:

Trade-off CPU cost - accuracy

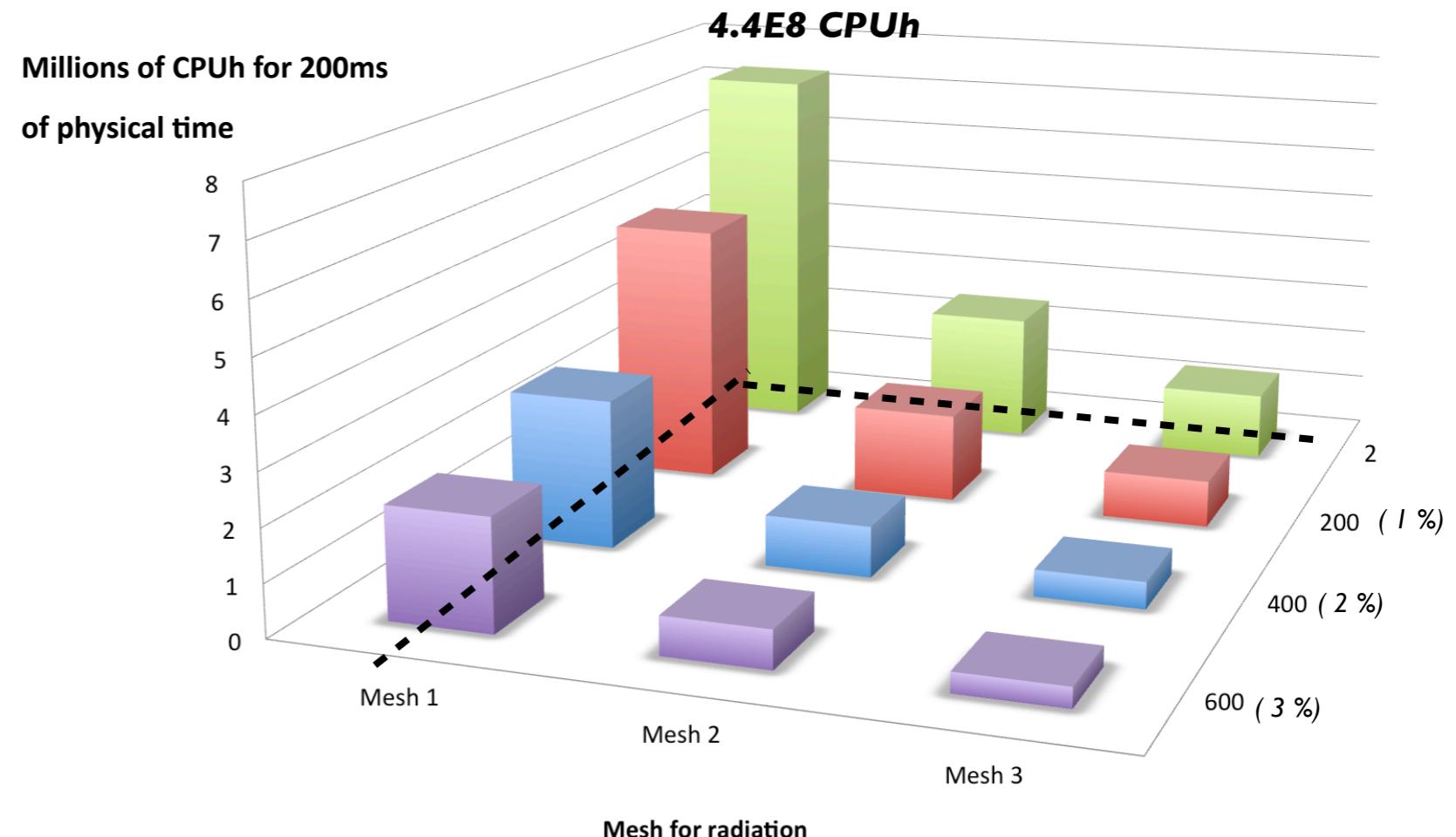
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4.4E8 CPUh = 20 years at 2400 cores



Coupled Simulations:

Trade-off CPU cost - accuracy

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Optimistic:

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4.4E8 CPUh = 20 years at 2400 cores

Pessimistic:

Mesh 3 (2.7M nodes)

Coupling every 600 iterations of fluid solver

3.8E5 CPUh = 7 days at 2400 cores (1000 times cheaper)

Millions of CPUh for 200ms
of physical time

4.4E8 CPUh

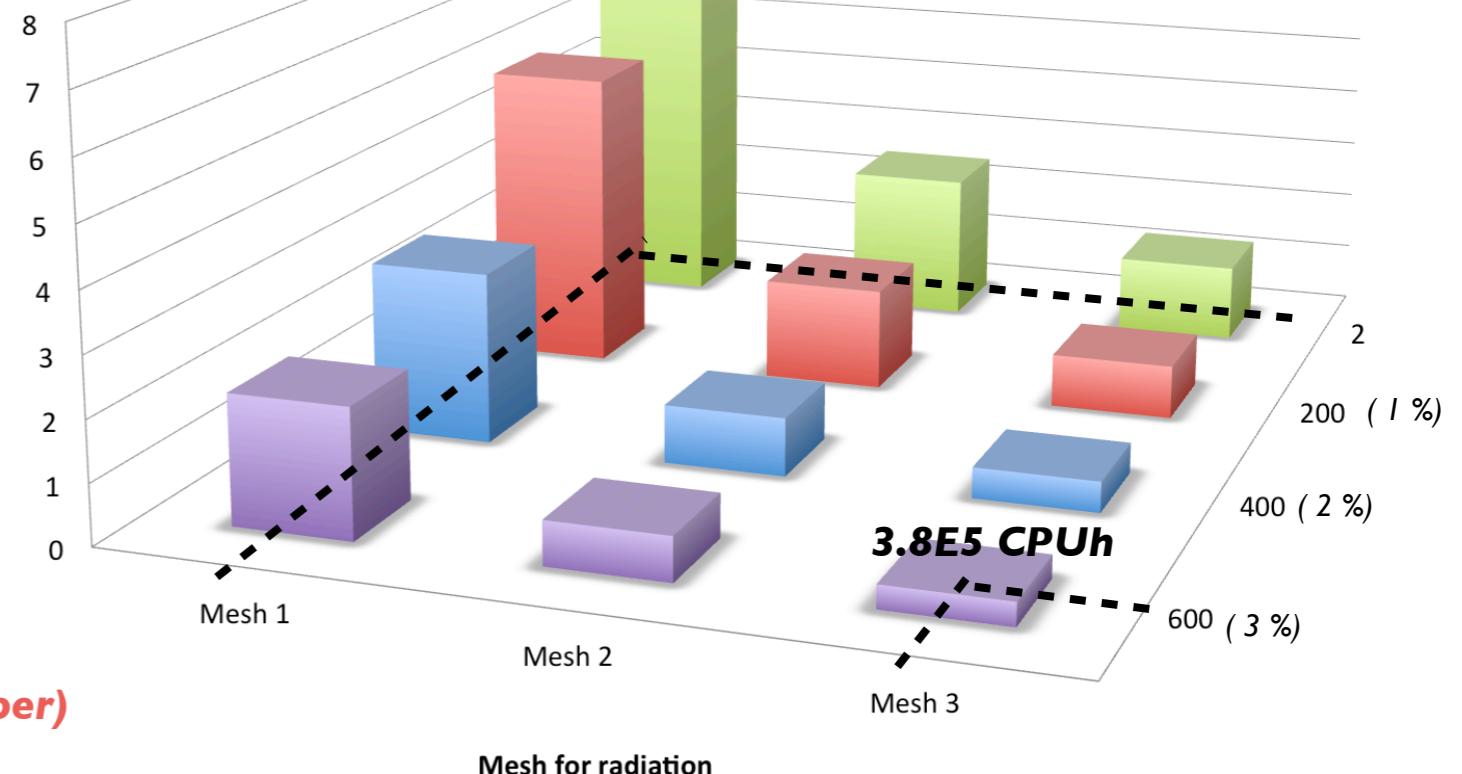
3.8E5 CPUh

Mesh 1

Mesh 2

Mesh 3

Mesh for radiation



Coupled Simulations:

Trade-off CPU cost - accuracy

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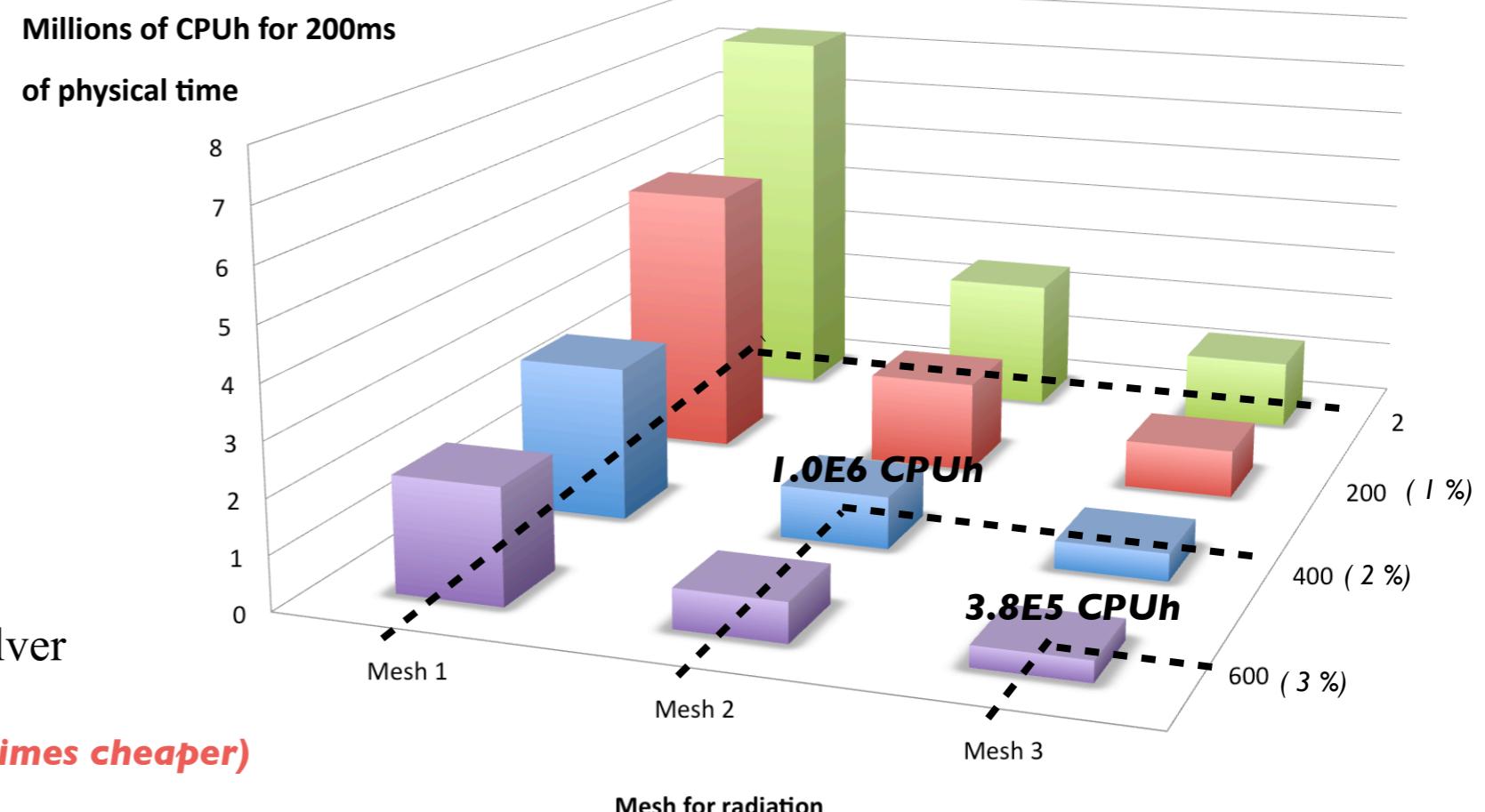
3.8E5 CPUh = 7 days at 2400 cores (1000 times cheaper)

Intermediate:

Mesh 2 (4M nodes)

Coupling every 400 iterations of fluid solver

1E6 CPUh = 20 days at 2400 cores



Coupled Simulations:

Trade-off CPU cost - accuracy

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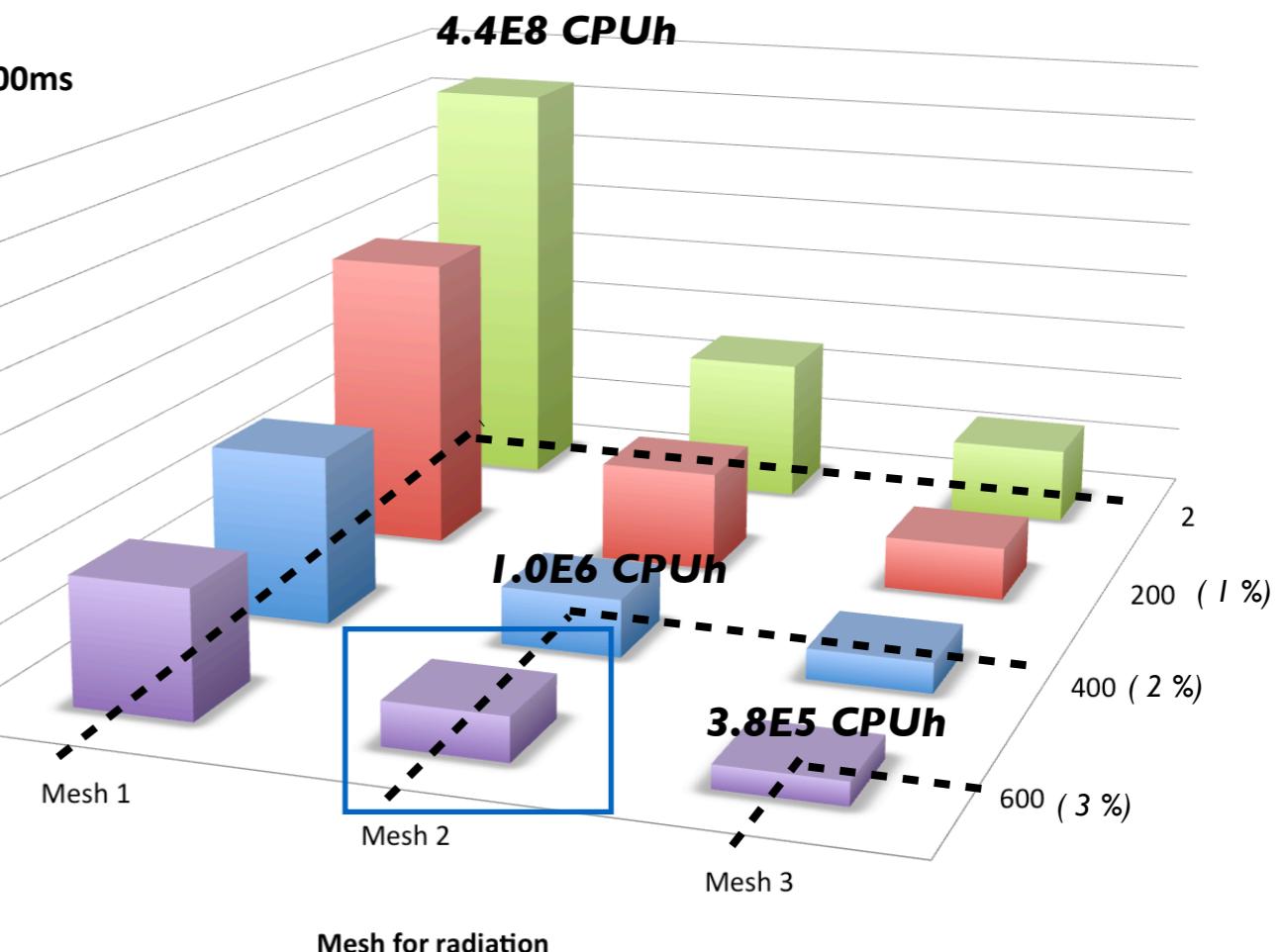
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Millions of CPUh for 200ms

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Chosen scenario:

Mesh 2 (4M nodes)

Coupling every 600 iterations of fluid solver

7.2E5 CPUh = 12.5 days at 2400 cores

Coupled Simulations:

Trade-off CPU cost - accuracy

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Intermediate:

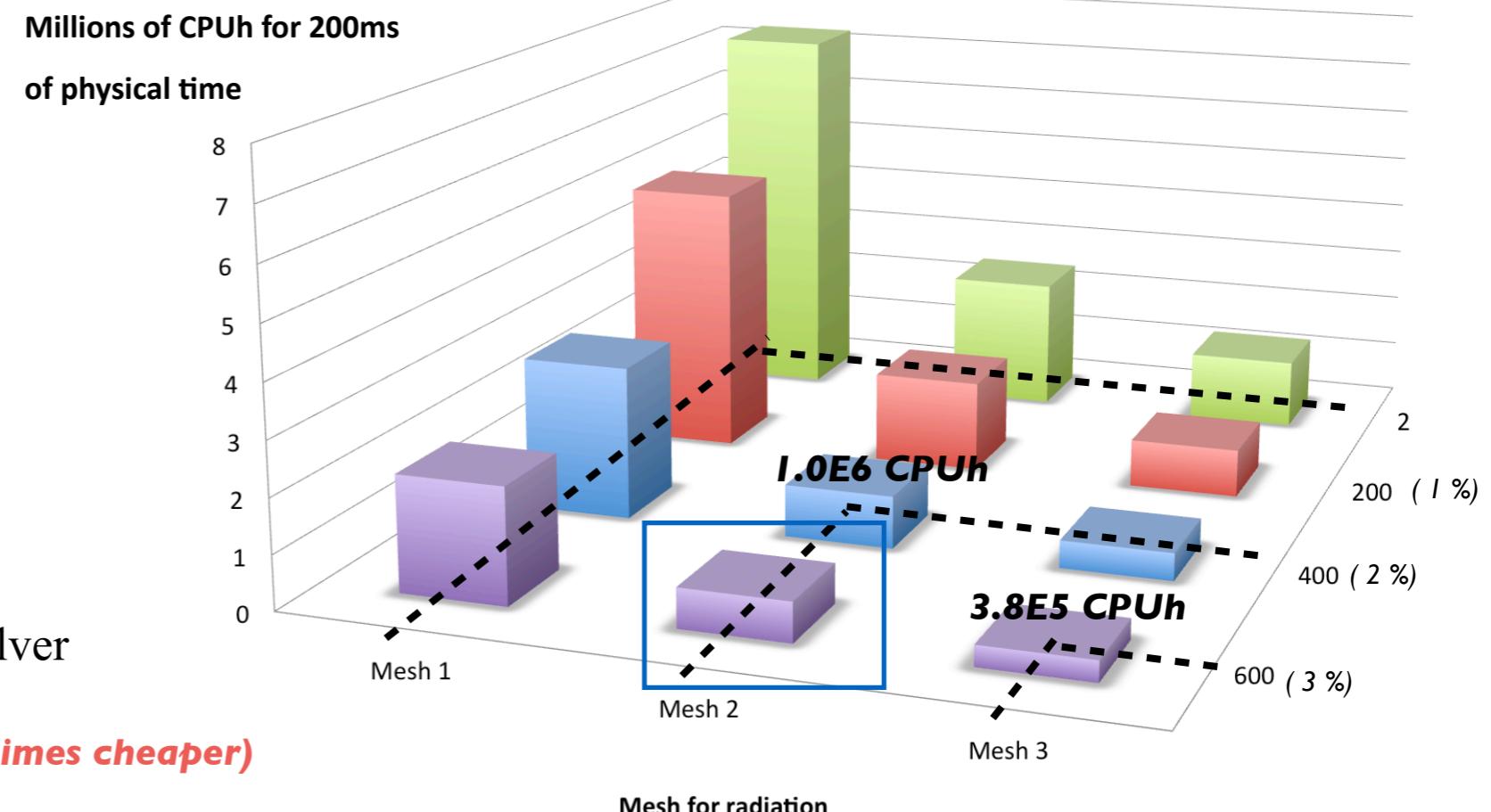
Mesh 2 (4M nodes)

Coupling every 400 iterations of fluid solver

1E6 CPUh = 20 days at 2400 cores

$$\text{Extra Cost} = \frac{\text{CPU time coupled simulation}}{\text{CPU time LES combustion simulation}} = 3.27$$

With QMC



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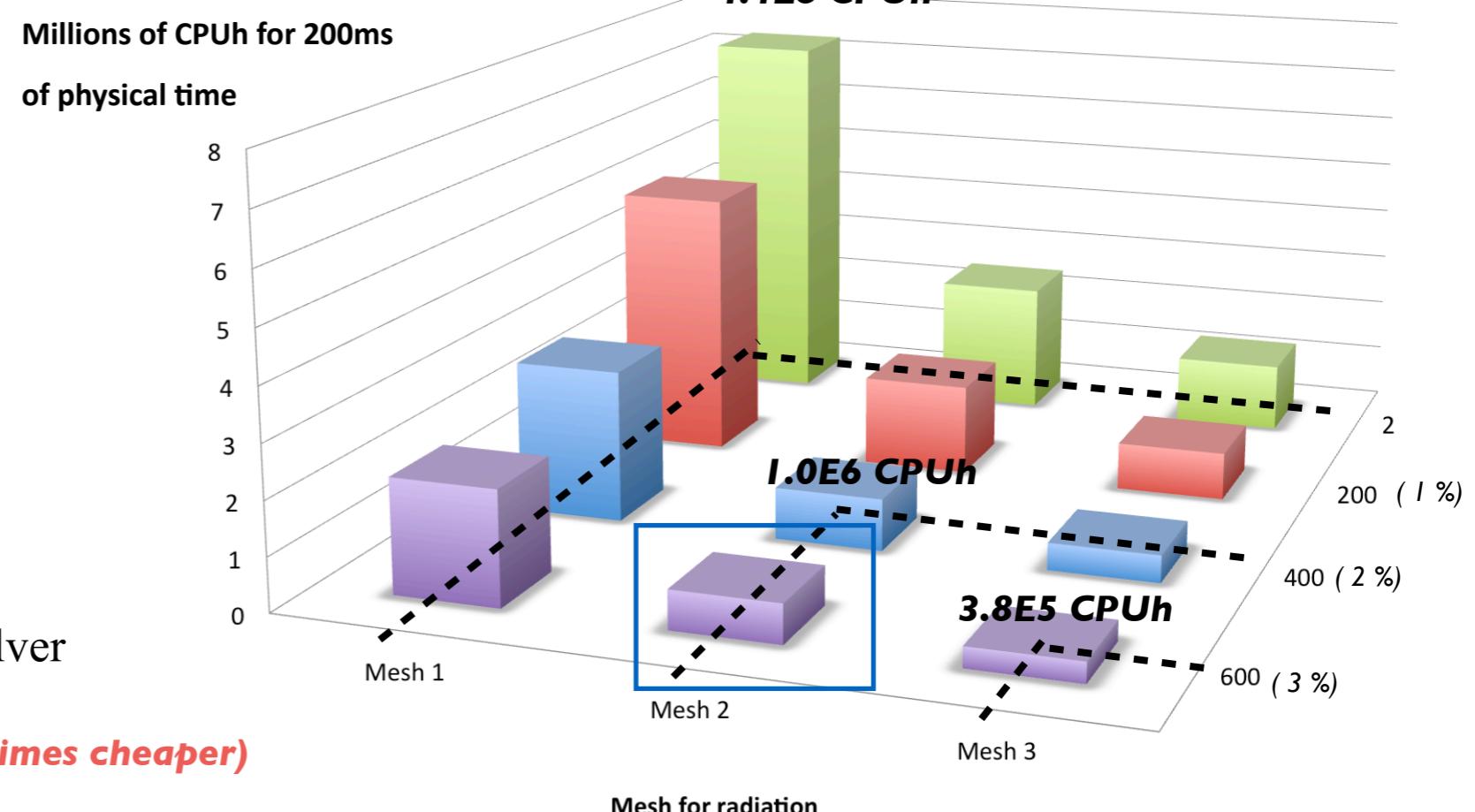
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With QMC

Extra Cost > 6

With MC



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Coupling every 600 iterations of fluid solver

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Outline

LES of a premixed swirled flame

- *Presentation of Oxytec chamber*
- *LES results*

Coupled simulations: numerical set-up

- *Monte Carlo solver*
- *CPU cost-accuracy trade off*



Conclusions

- *Mesh size for radiation : 4 millions nodes*
- *Coupling frequency : 600 iterations*

First results of coupled simulations

- *Impact of spectral properties of quartz*
- *Radiative heat transfer impact on a methane air flame*

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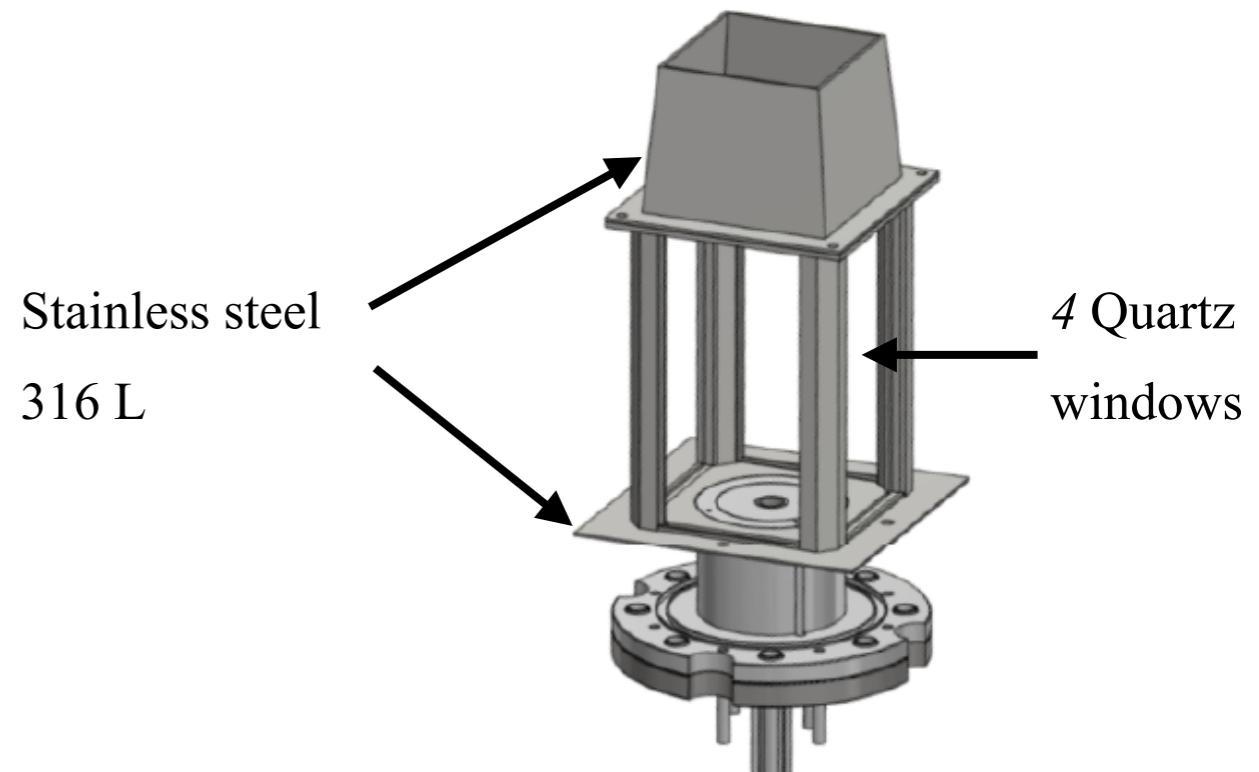
Boundary Conditions for radiation solver

- **Opaque walls:**

- Measured wall temperature
- Stainless Steel emissivity^[1]

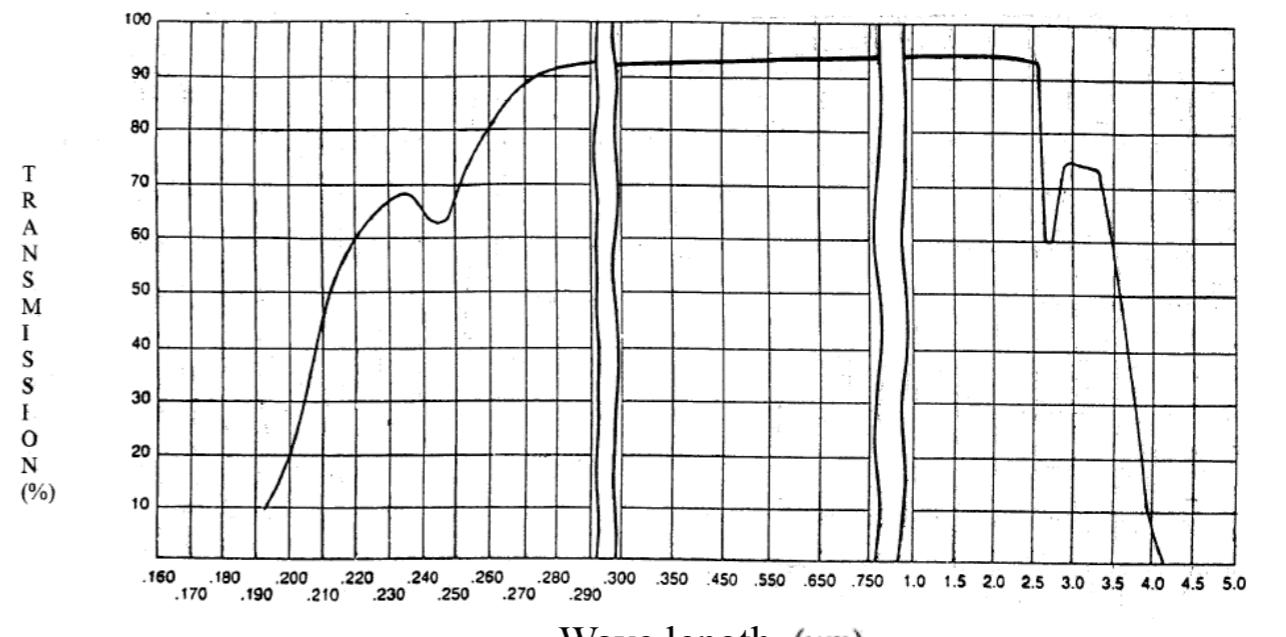
Bottom chamber : 0.28

Convergent : 0.32



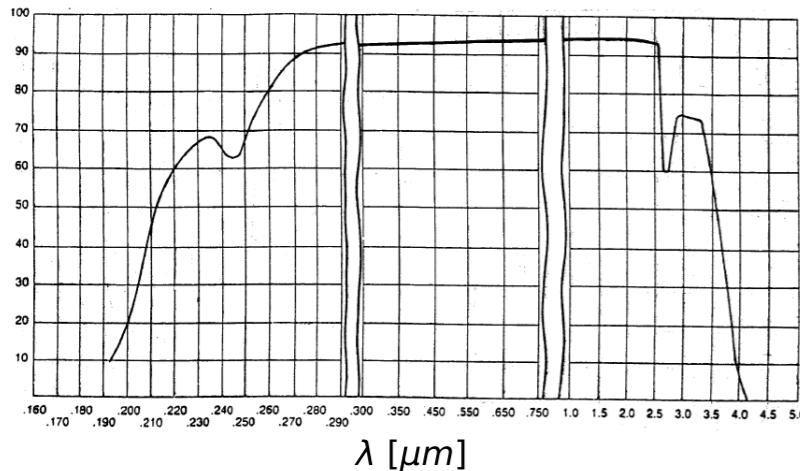
- **Quartz windows:**

- Measured quartz temperature
- Spectral radiative properties

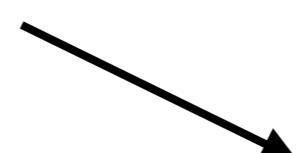


Transmissivity of a window of fused silica VI 942 between 0.16 and 5.0 micrometers.

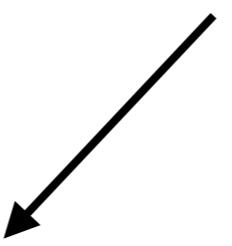
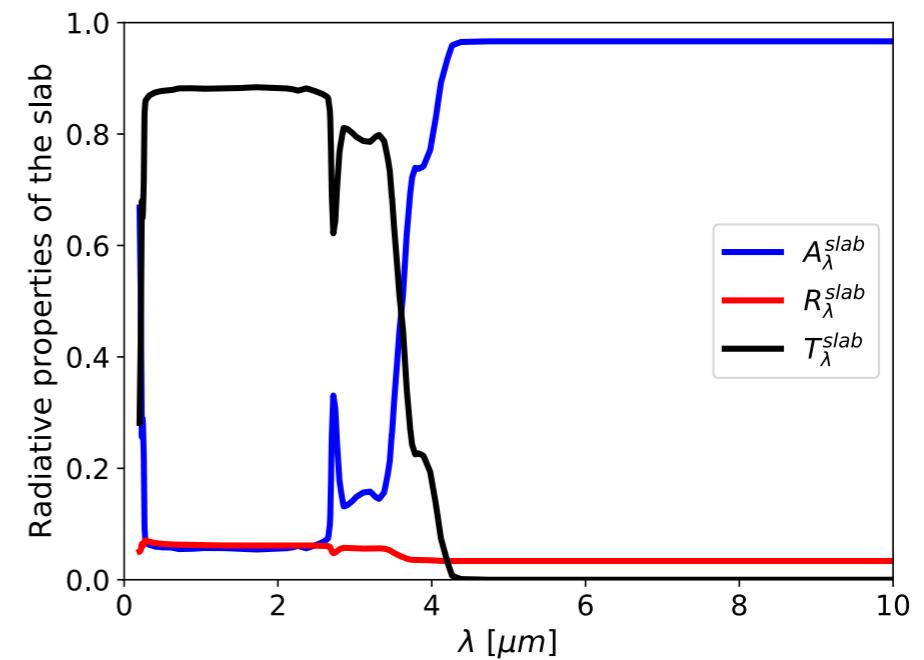
Semi-transparent quartz windows



*Transmissivity of a window of fused silica
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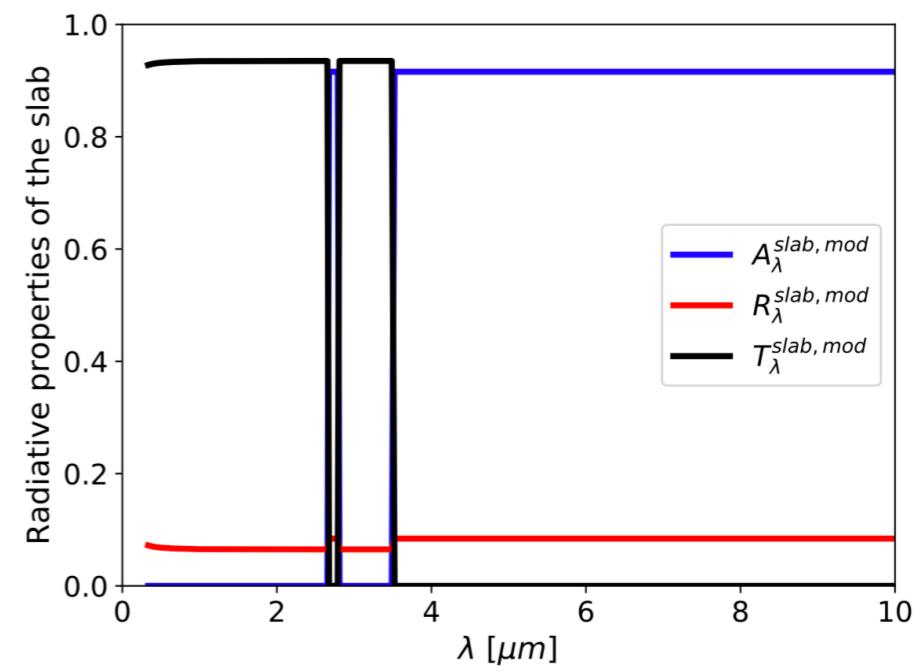


Slab properties



Simplified band model[1]

$$\begin{aligned} T_\lambda^{slab} &< T_{slab}^{switch} && \text{Opaque} \\ T_\lambda^{slab} &> T_{slab}^{switch} && \text{Transparent} \end{aligned}$$



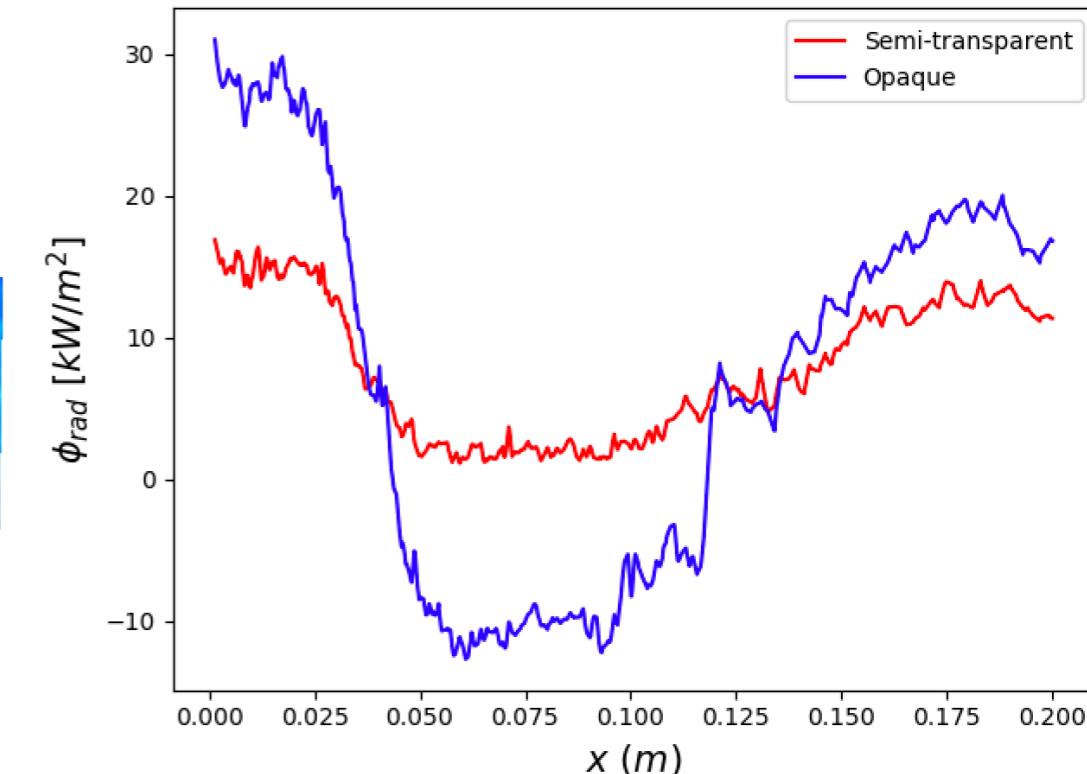
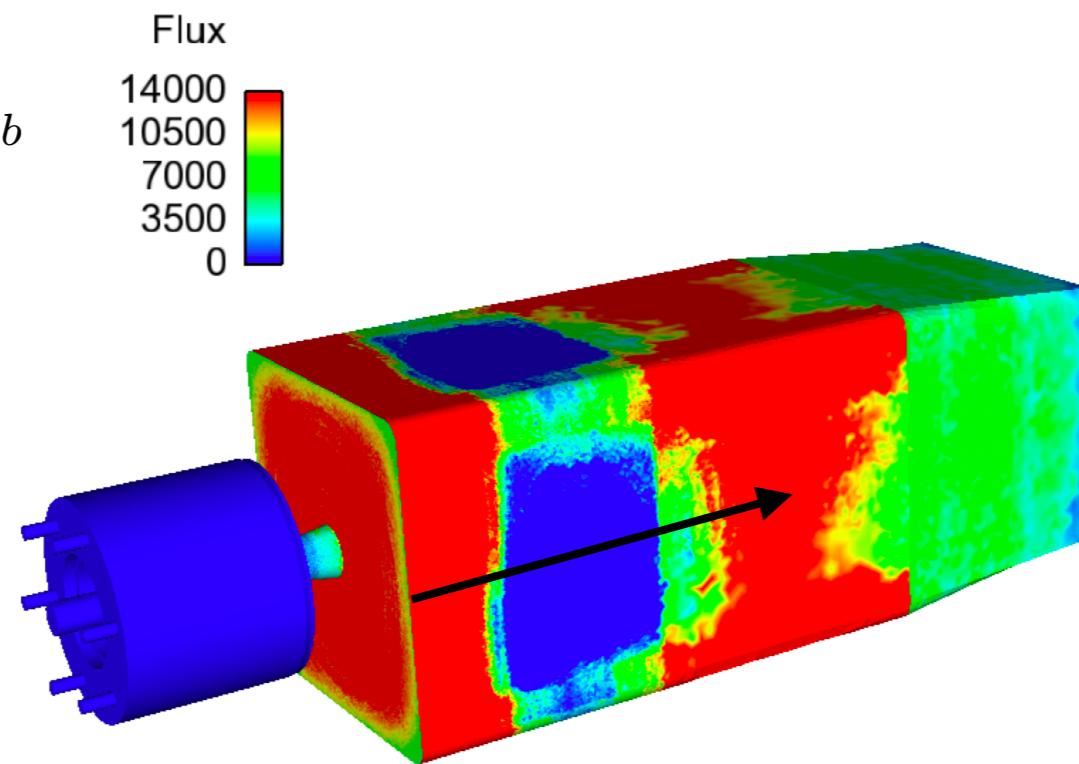
Impact of quartz spectral properties

Radiative Flux

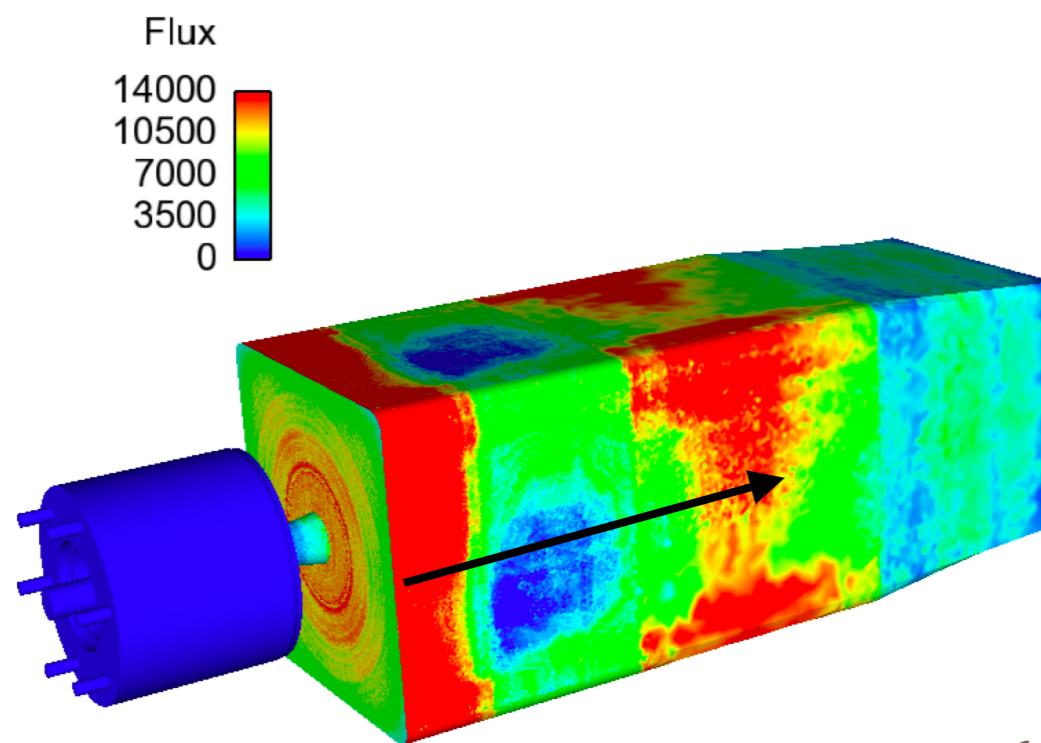
Opaque

$$A_{\lambda}^{slab} = 1 - R_{\lambda}^{slab}$$

$$T_{\lambda}^{slab} = 0$$



Semi-transparent



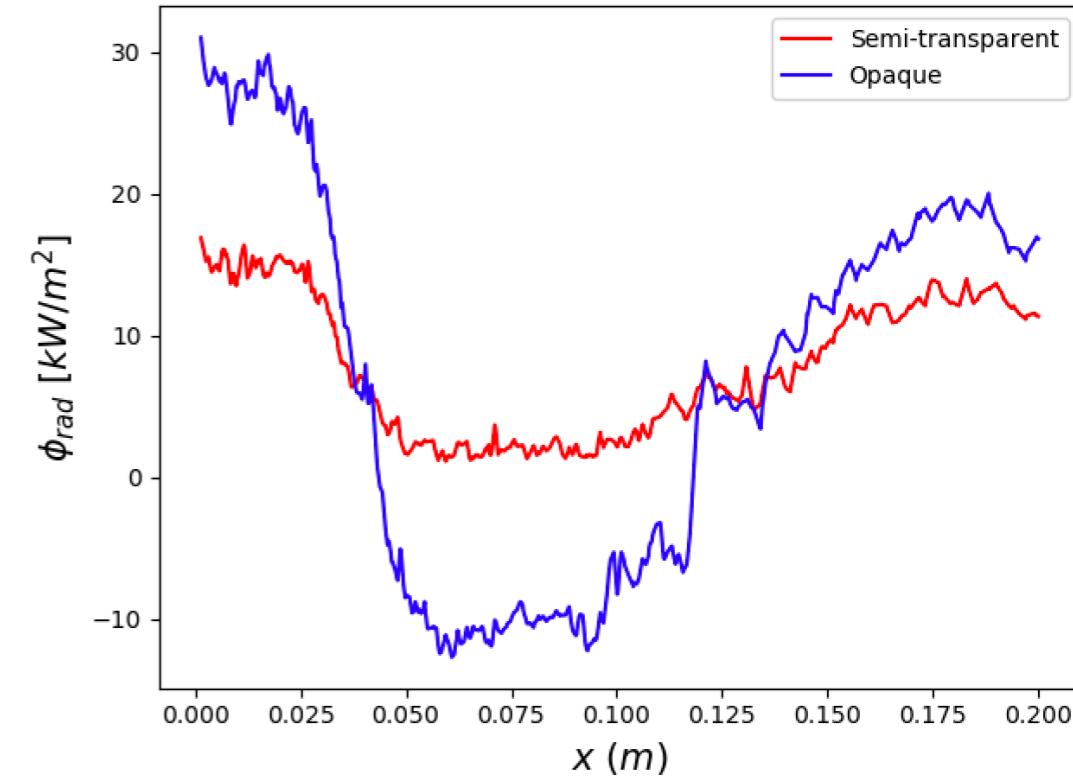
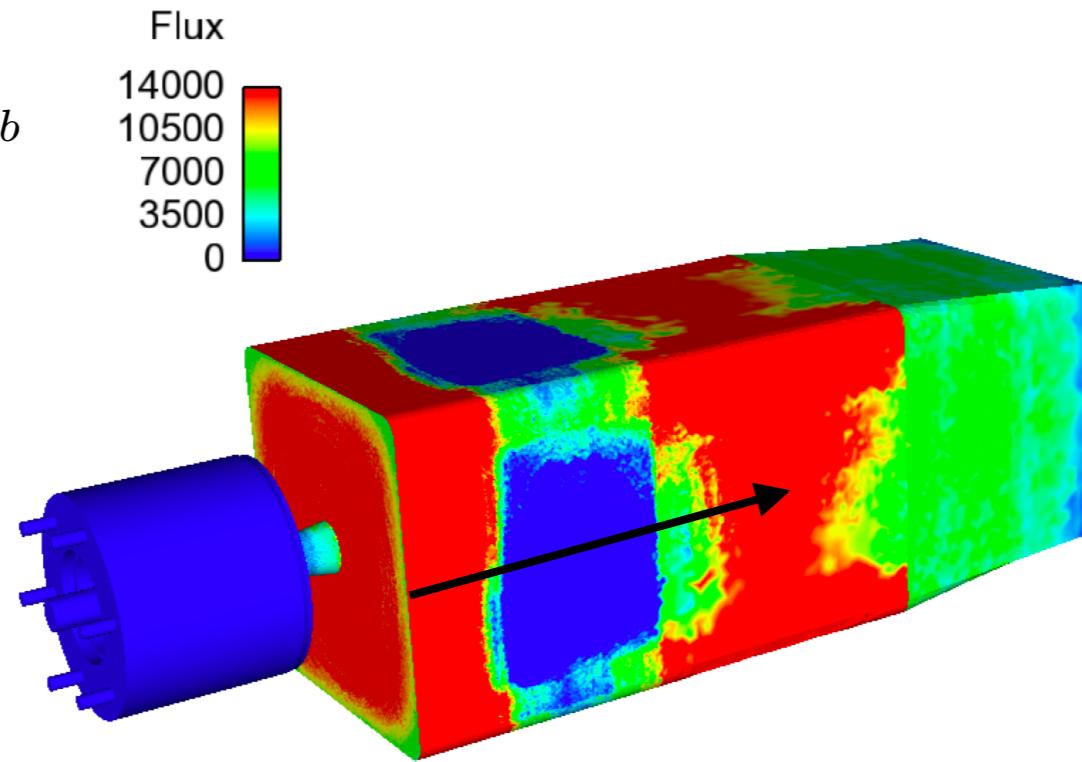
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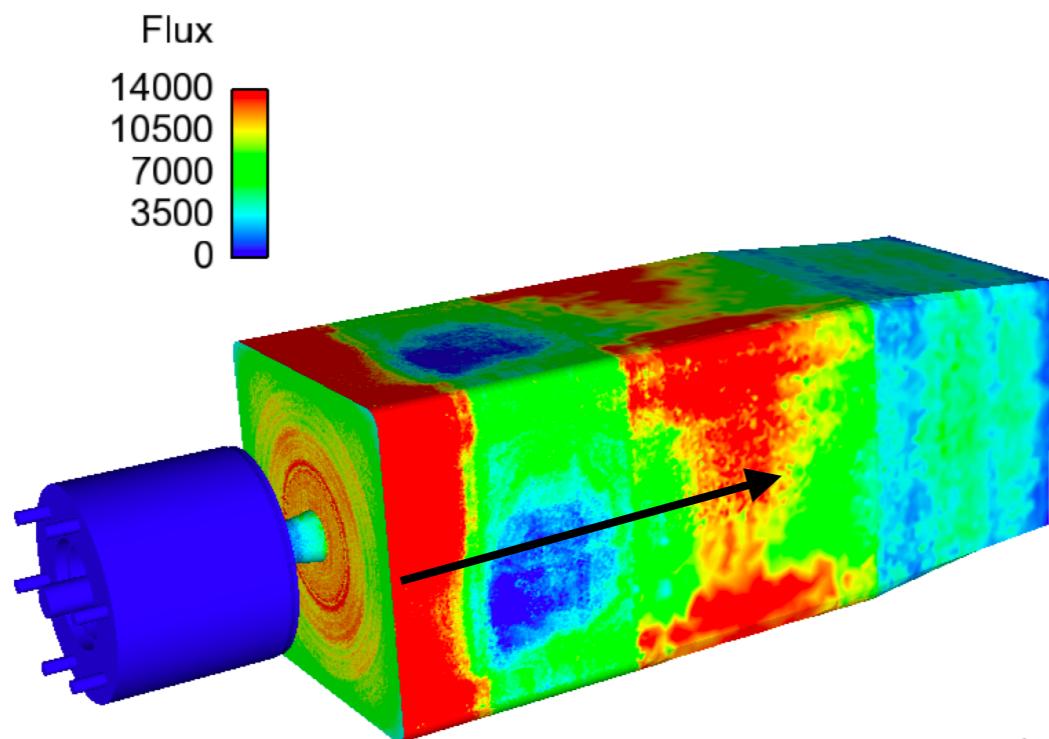
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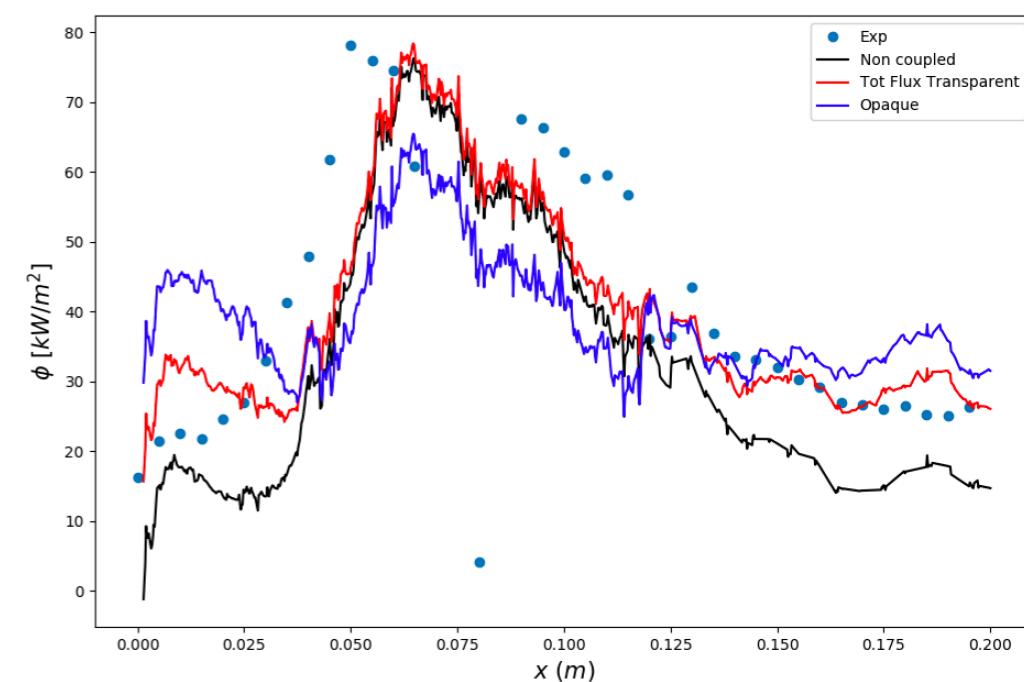
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Semi-transparent

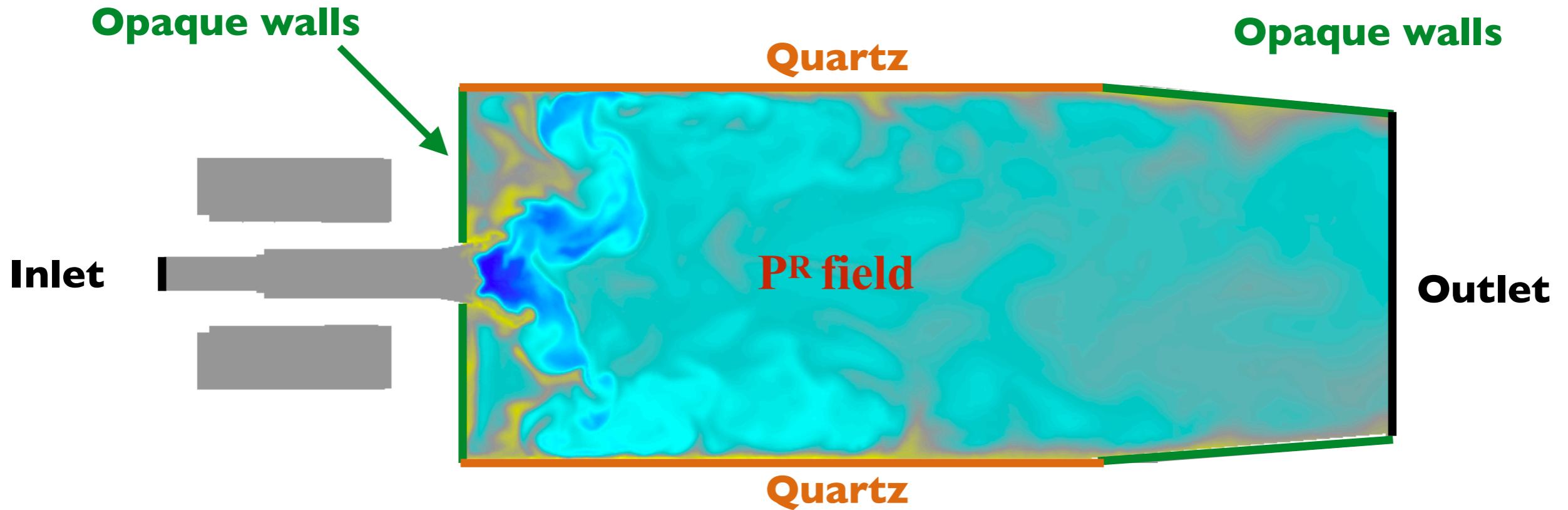


Total wall Flux



Balance of radiative transfer

Instantaneous solution from coupled simulations



$$\int_V -P^R dV = \int_{\text{inlet}} \varphi_{\text{rad}} \cdot dS + \int_{\text{outlet}} \varphi_{\text{rad}} \cdot dS + \int_{\text{opaque walls}} \varphi_{\text{rad}} \cdot dS + \int_{\text{quartz windows}} \varphi_{\text{rad}} \cdot dS$$

3185 W

9 W

487 W

466 W

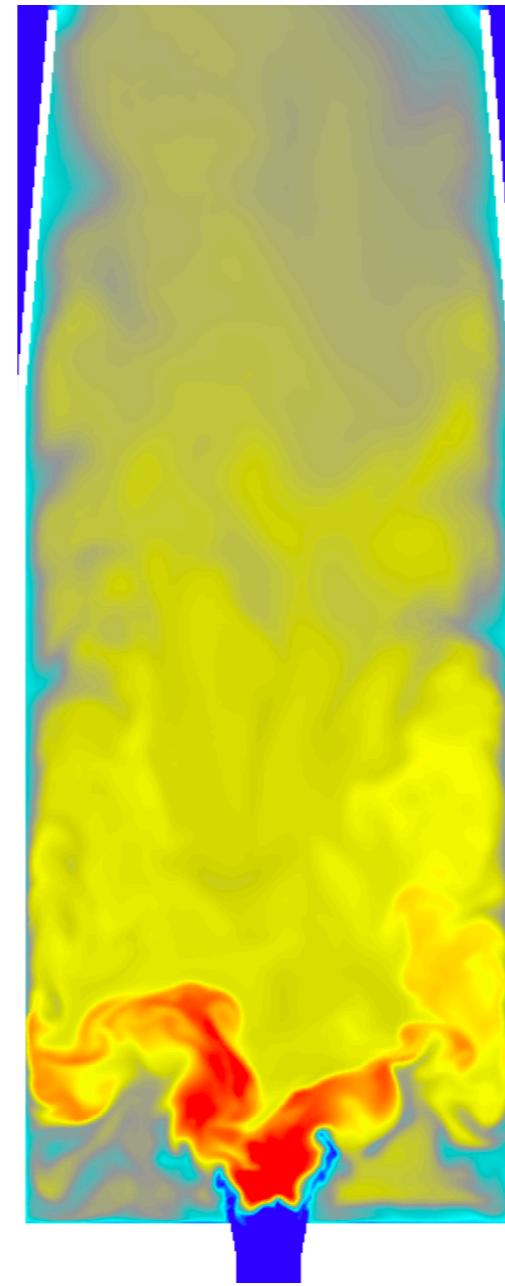
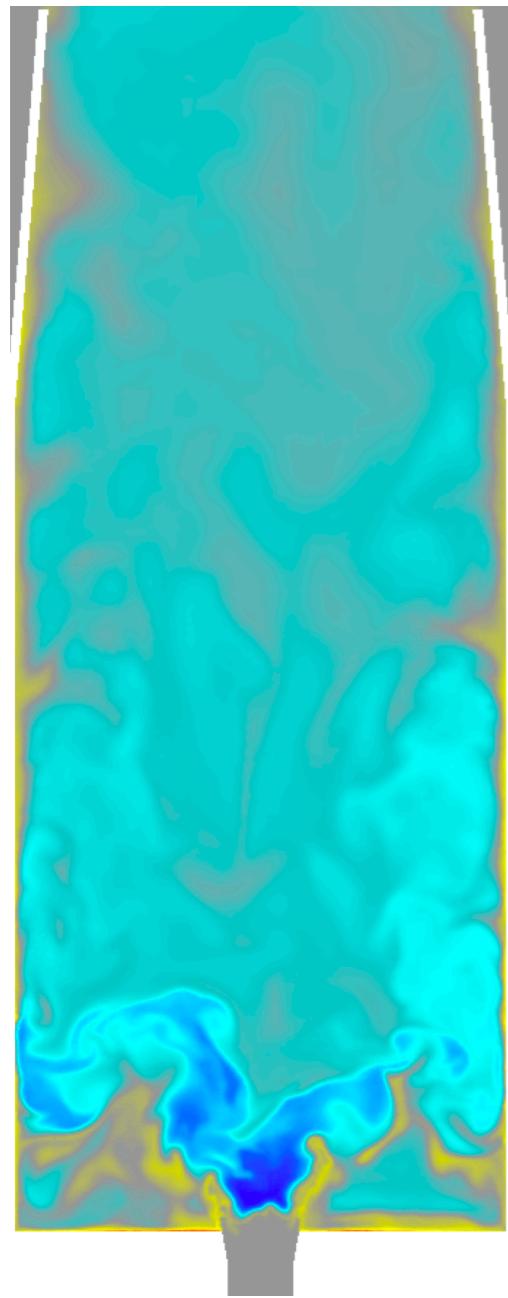
2223 W

(21% out of
flame P_{th} = 14 kW)

$$\int_{\text{quartz windows}} \varphi_{\text{rad}} \cdot dS \rightarrow \begin{array}{l} \text{Absorbed/emitted} \\ \text{Transmitted} \end{array} = 1696 \text{ W (53 % out of PR)}$$

$$= 785 \text{ W (24 % out of PR)}$$

Preliminary coupled results



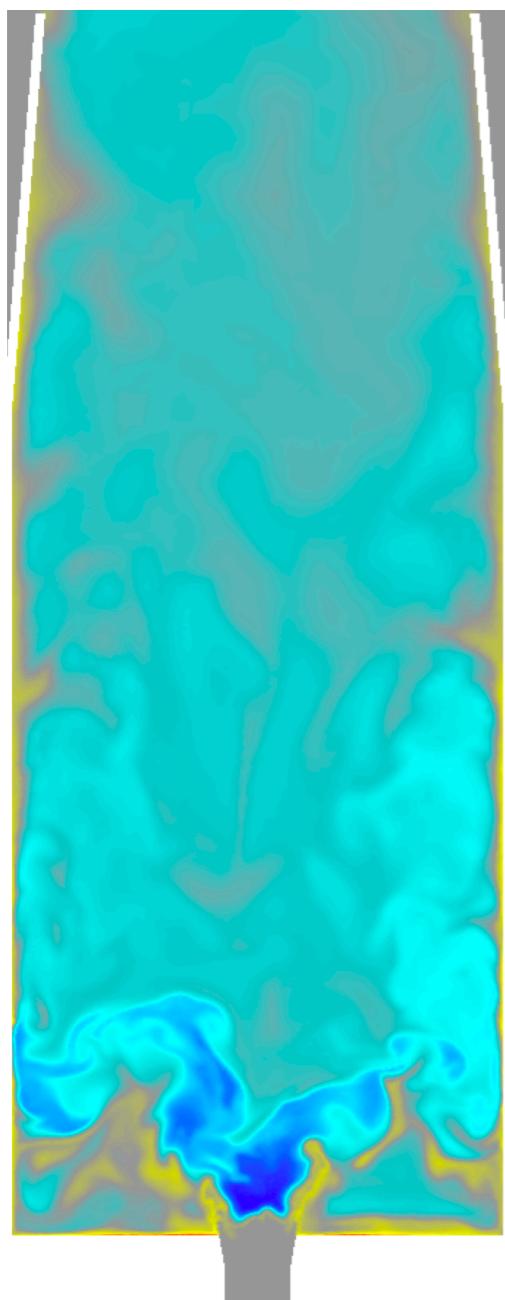
Radiative Power (W/m^3)

-1.81e+06 -9.04e+05 -1.36e+03 9.01e+05 1.80e+06

Temperature (K)

300 778 1256 1735 2213

Preliminary coupled results



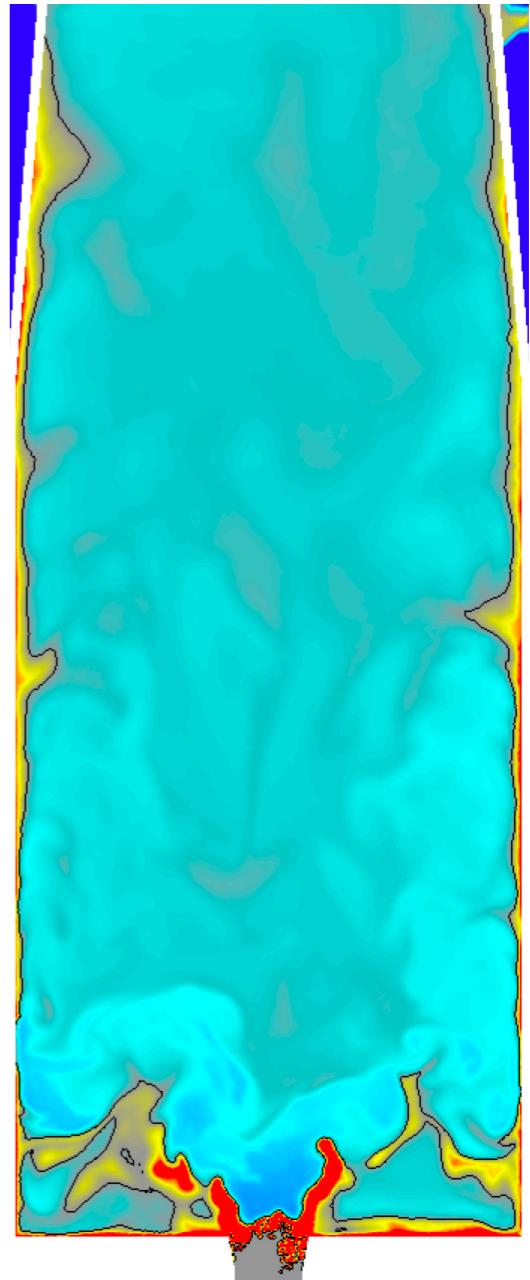
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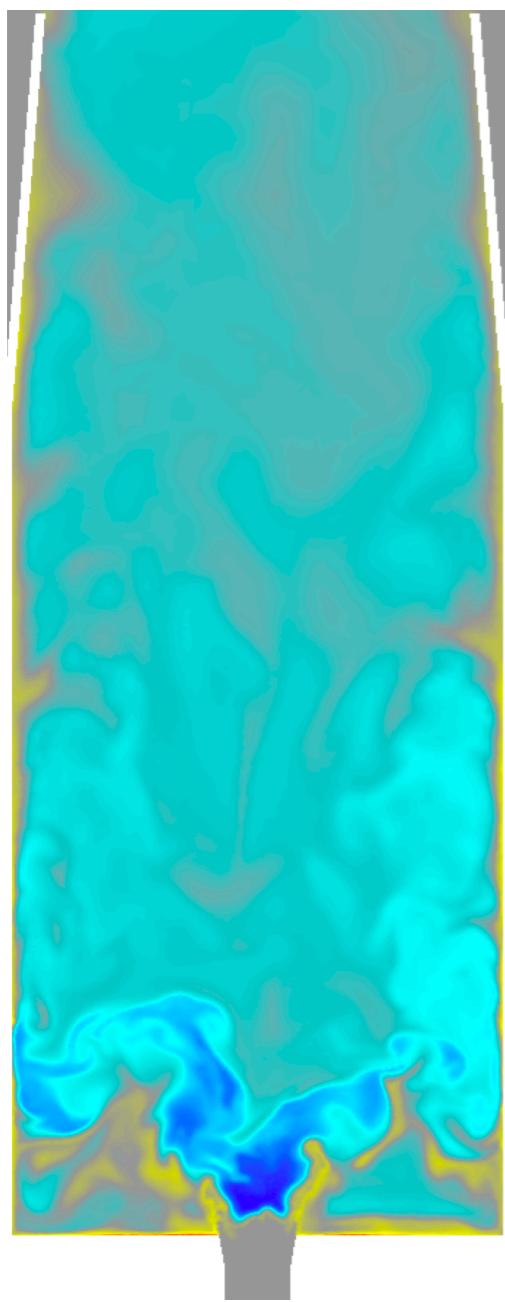


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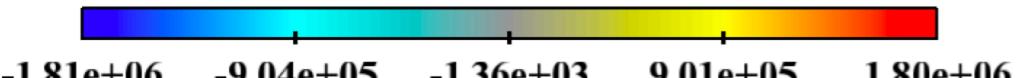
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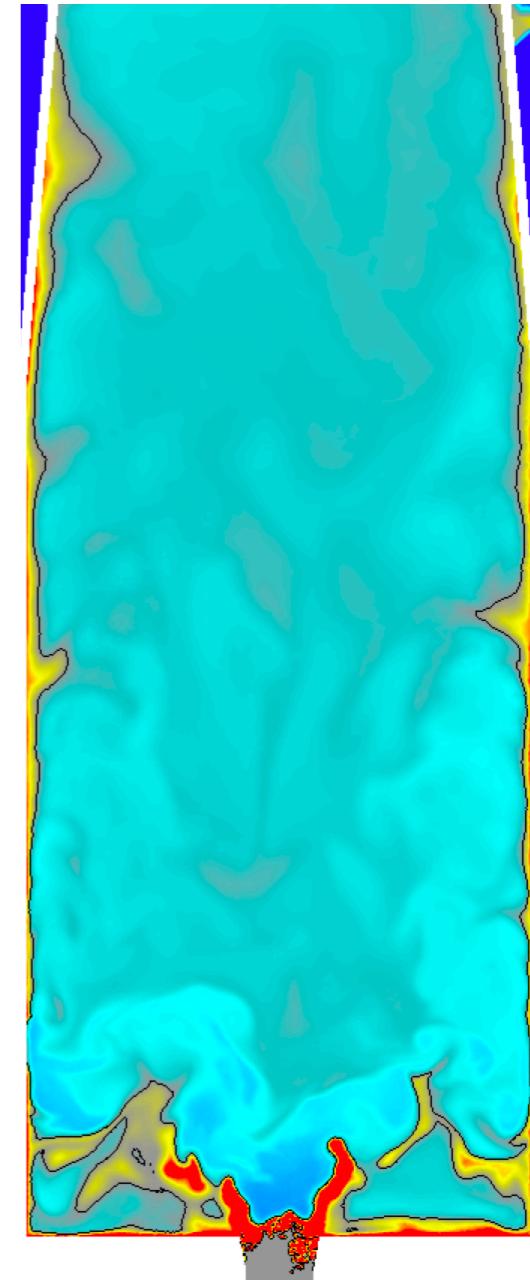
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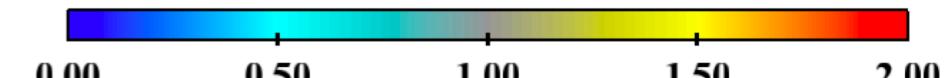
Temperature (K)



Black line : Ratio = 1



P^A/P^E

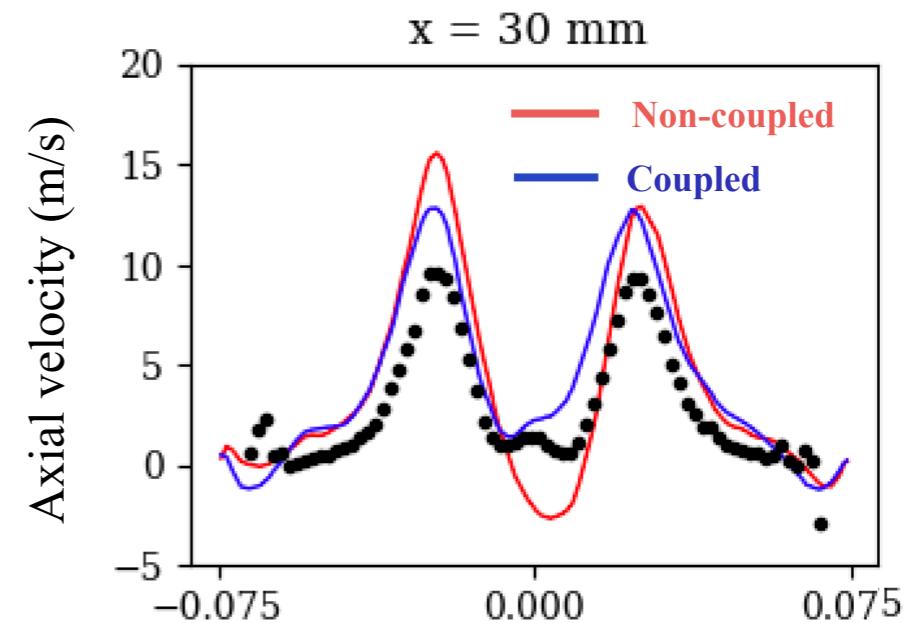


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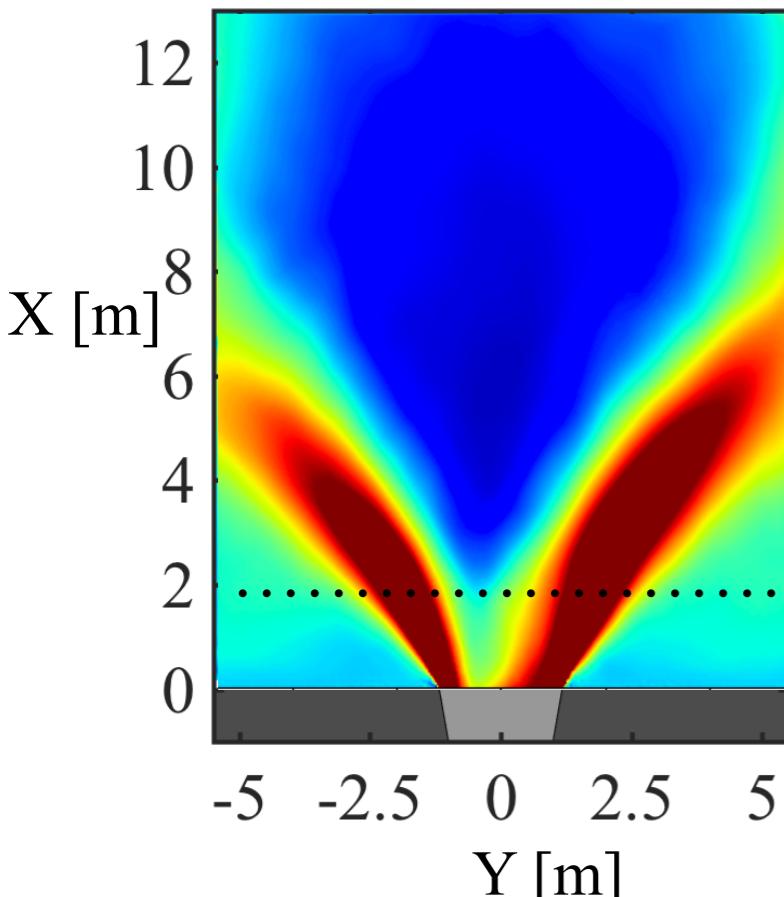
Impact on velocity field

Flow topology

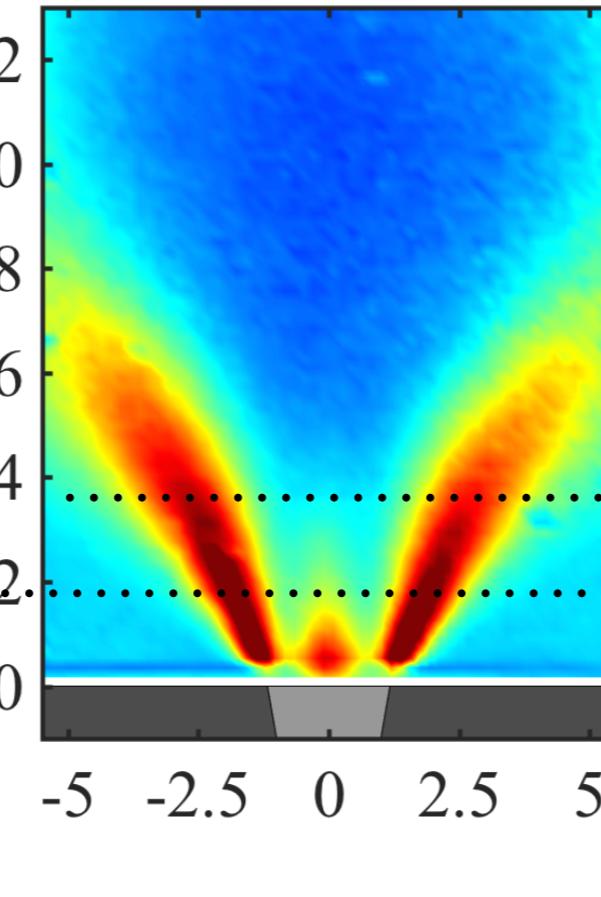
- Stagnation point position and Recirculation Zone
- Velocity Profiles



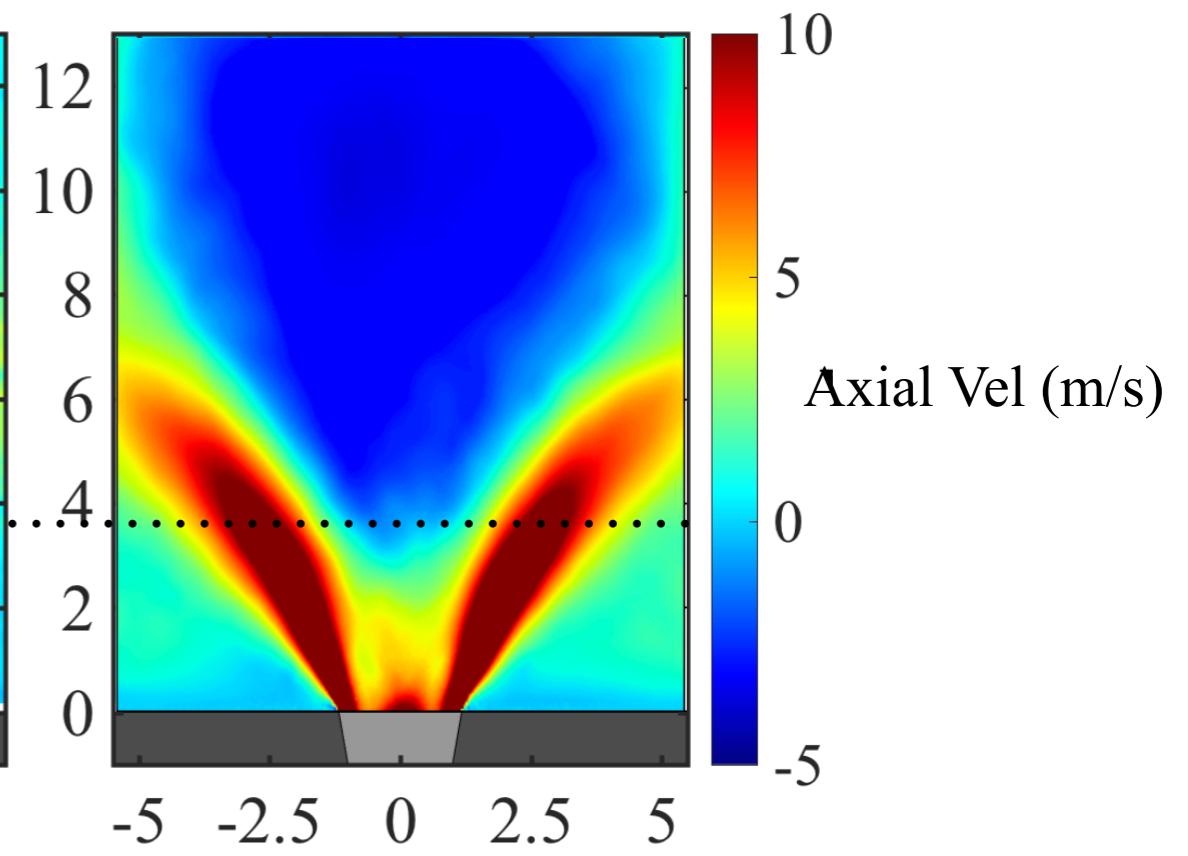
Non coupled LES



Experiments



Coupled LES-MC



Outline

LES of a premixed swirled flame

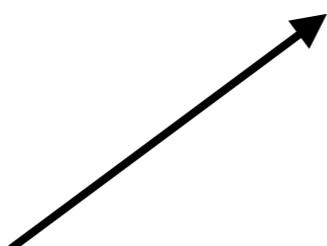
- *Presentation of Oxytec chamber*
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Coupled simulations: numerical set-up

- *Monte Carlo solver*
- *CPU cost-accuracy trade off*

First results of coupled simulations

- *Impact of spectral properties of quartz*
- *Radiative heat transfer impact on a methane air flame*



Conclusions

Treatment of quartz spectral properties and impact on radiative fluxes

Preliminary results show impact on:

- *Flow topology*
- *Wall fluxes*
- *Flame stabilization^[1]*



Thank you for your attention

This project has received funding from the European Union's Horizon 2020 research and innovation program under the Marie Skłodowska-Curie grant agreement No 643134.



Quasi-Monte Carlo methods

To reduce the MC error:

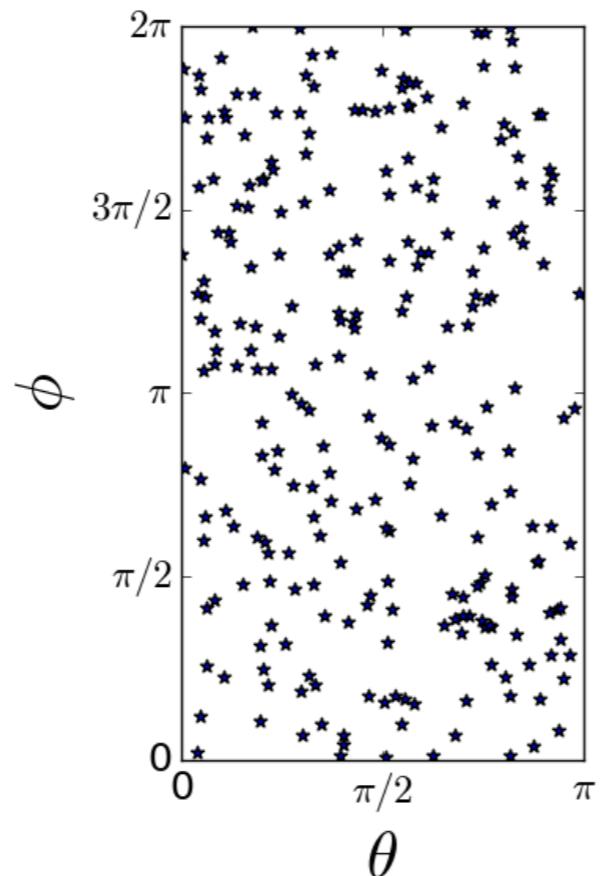
Alternative sampling method

- [1] Halton, J. H. 1960. “On the efficiency of certain quasi-random sequences of points in evaluating multi-dimensional integrals”. Numerische Mathematik
- [2] Joe, S., and Kuo, F. Y., 2008. “Constructing Sobol sequences with better two-dimensional projections”. SIAM Journal on Scientific Computing
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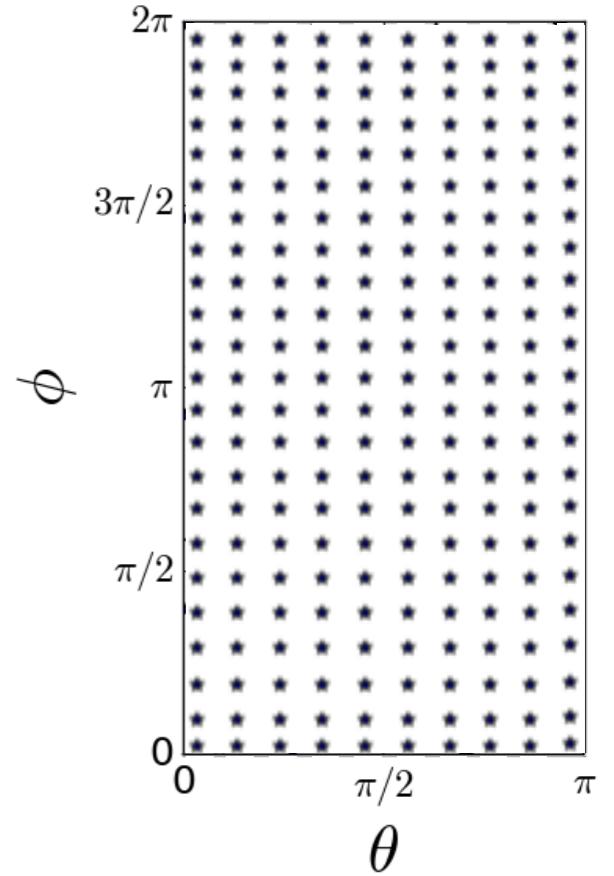
Quasi-Monte Carlo methods

To reduce the MC error:

Alternative sampling method



**Pseudo-random
sampling
MC**



**Deterministic
discretization
(Unfeasible in high-D)**

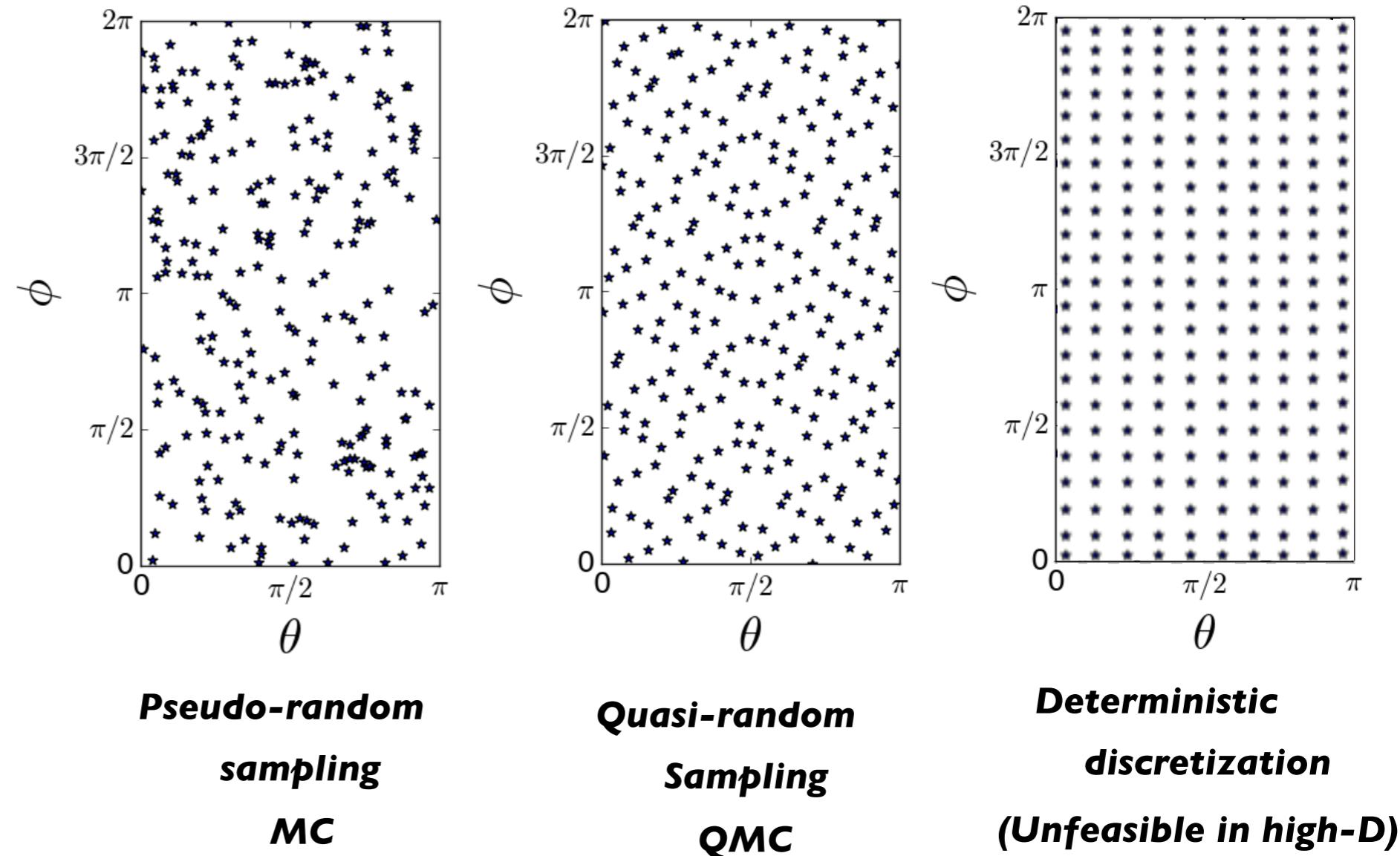
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Quasi-Monte Carlo methods

To reduce the MC error:

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- Quasi-random instead of pseudo-random sampling



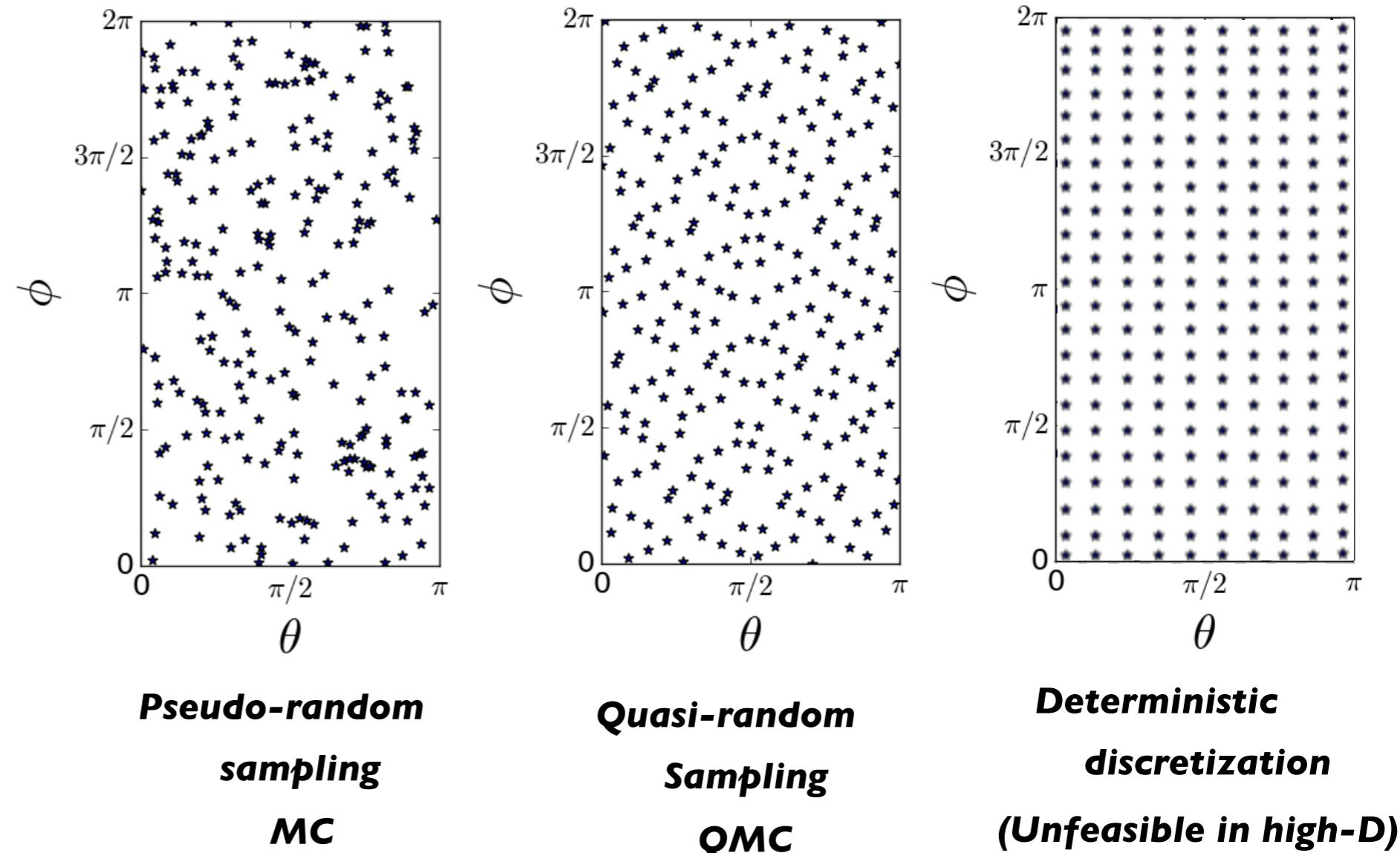
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Quasi-Monte Carlo methods

To reduce the MC error:

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- Quasi-random instead of pseudo-random sampling
- Low-discrepancy sequences:
Halton^[1]
Sobol^[2]
Faure^[3]
 points more uniformly distributed



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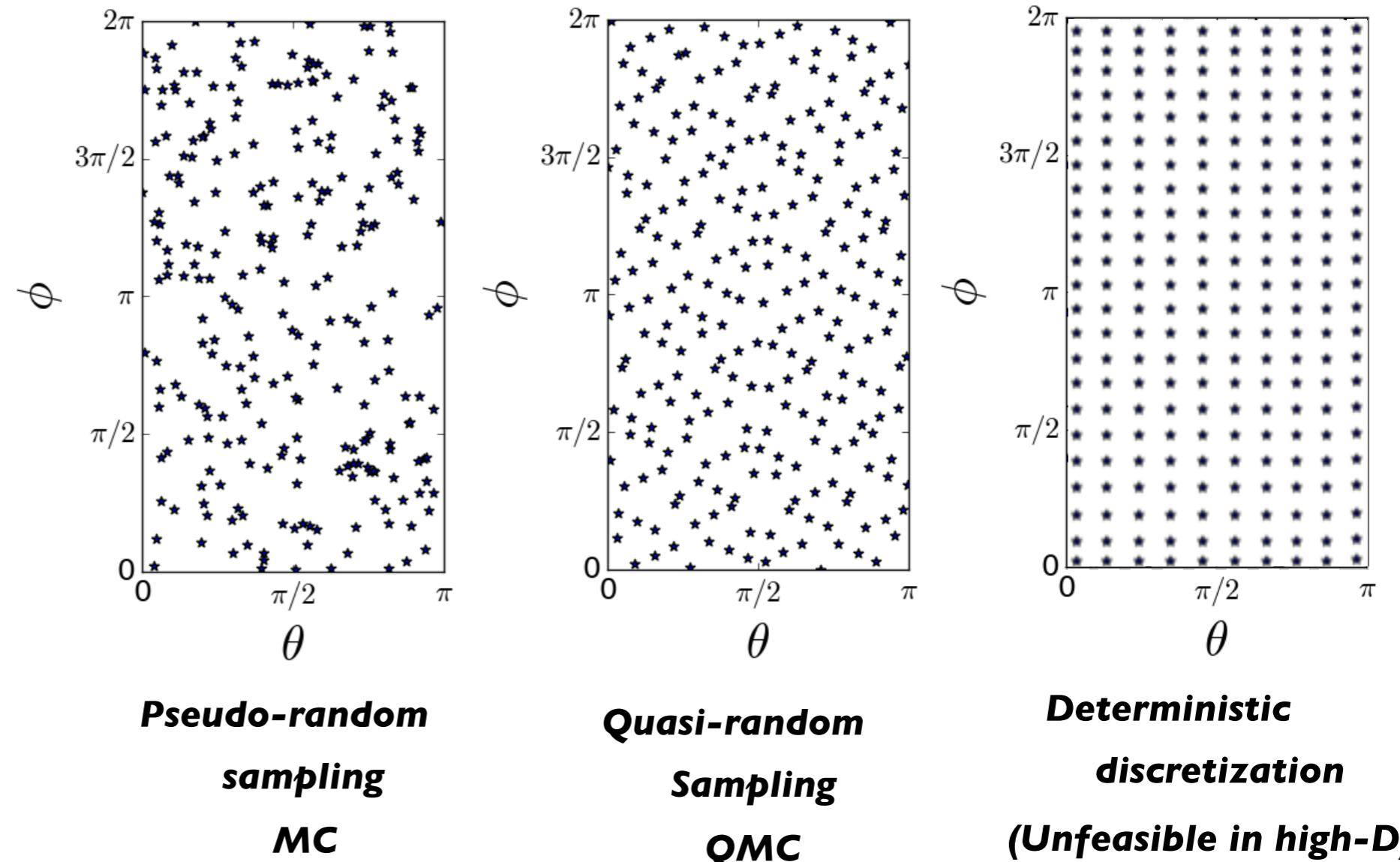
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Quasi-Monte Carlo methods

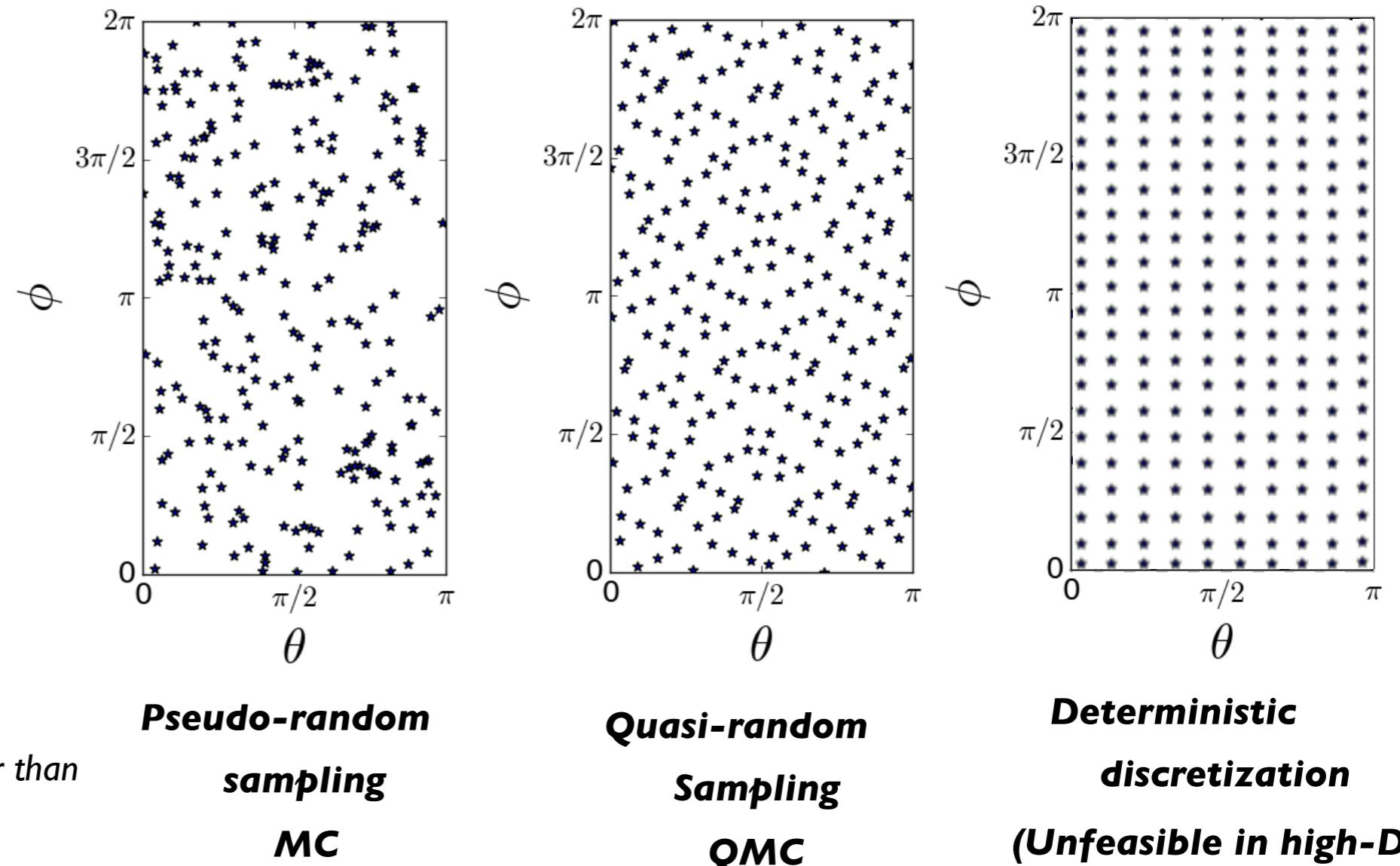
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$$\varepsilon \propto \frac{1}{N}$$

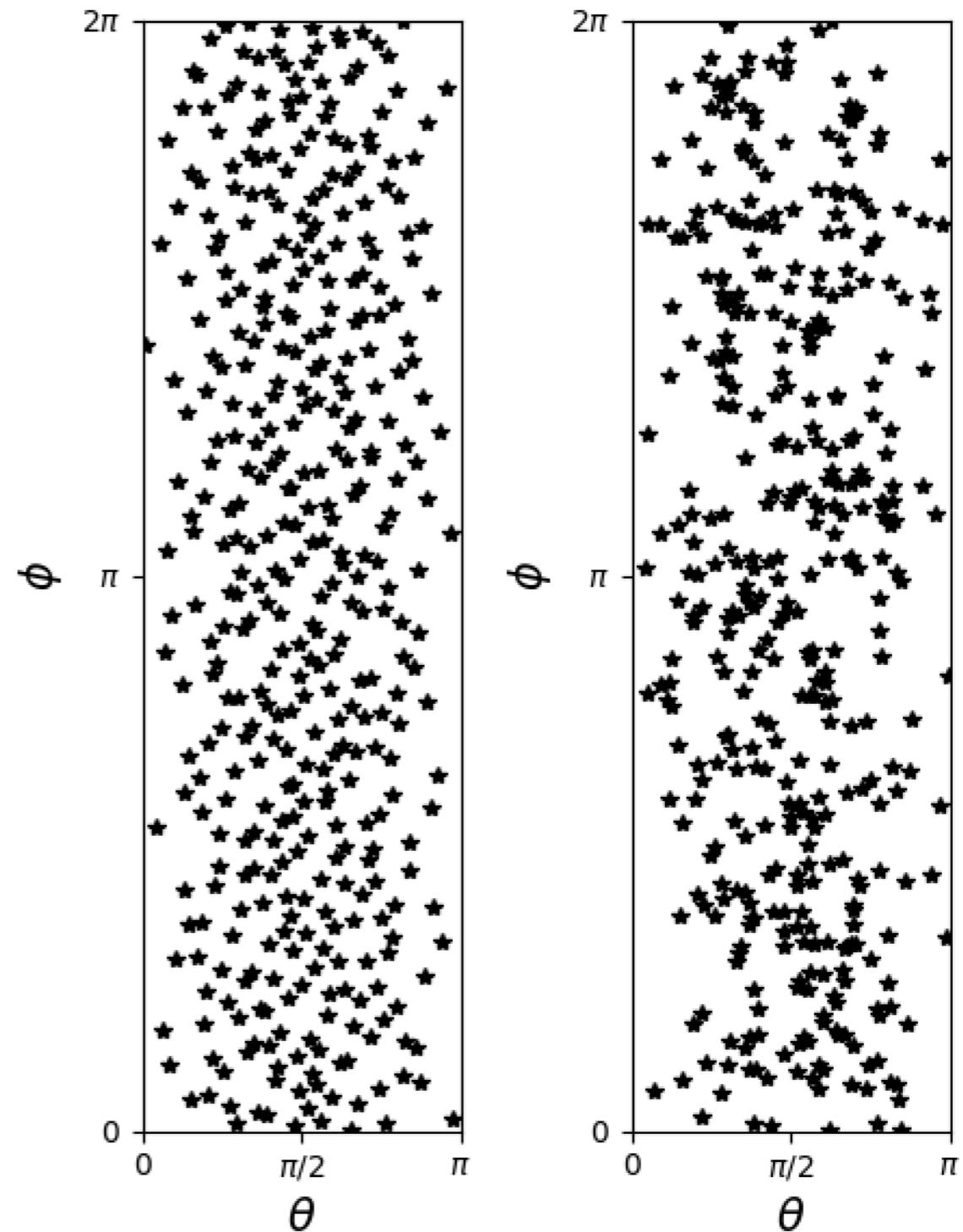


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Quasi-Monte Carlo methods

Koksma-Hlawka inequality :

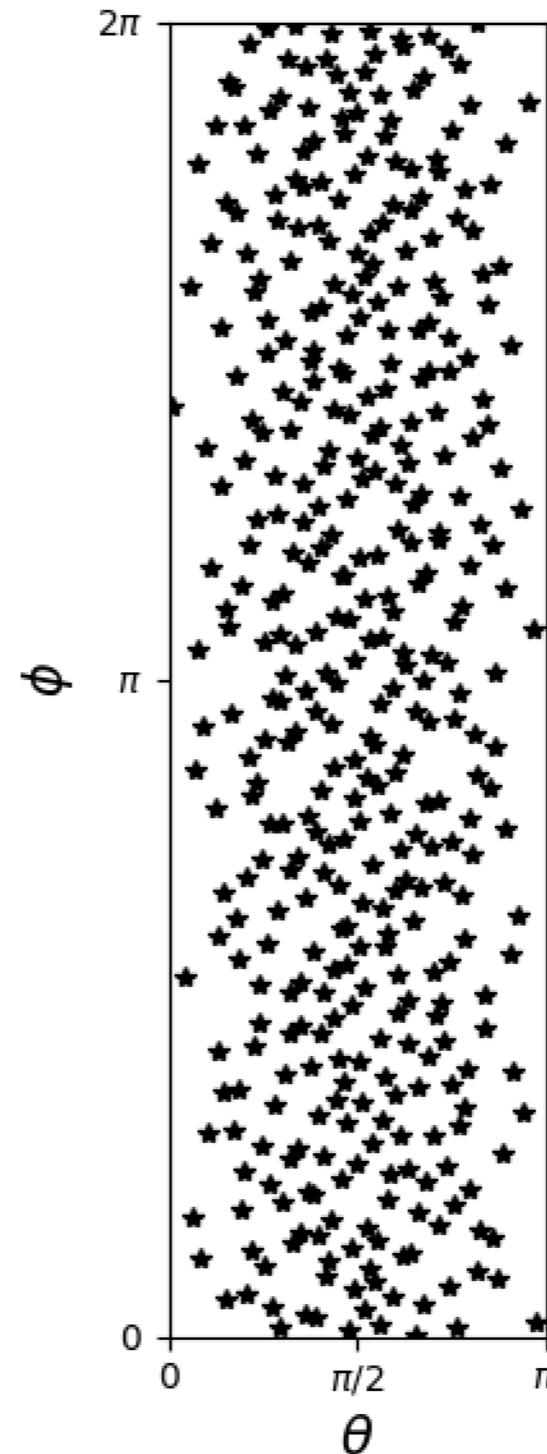
$$\epsilon[f] \leq CV[f] \frac{(\ln n)^d}{n}$$



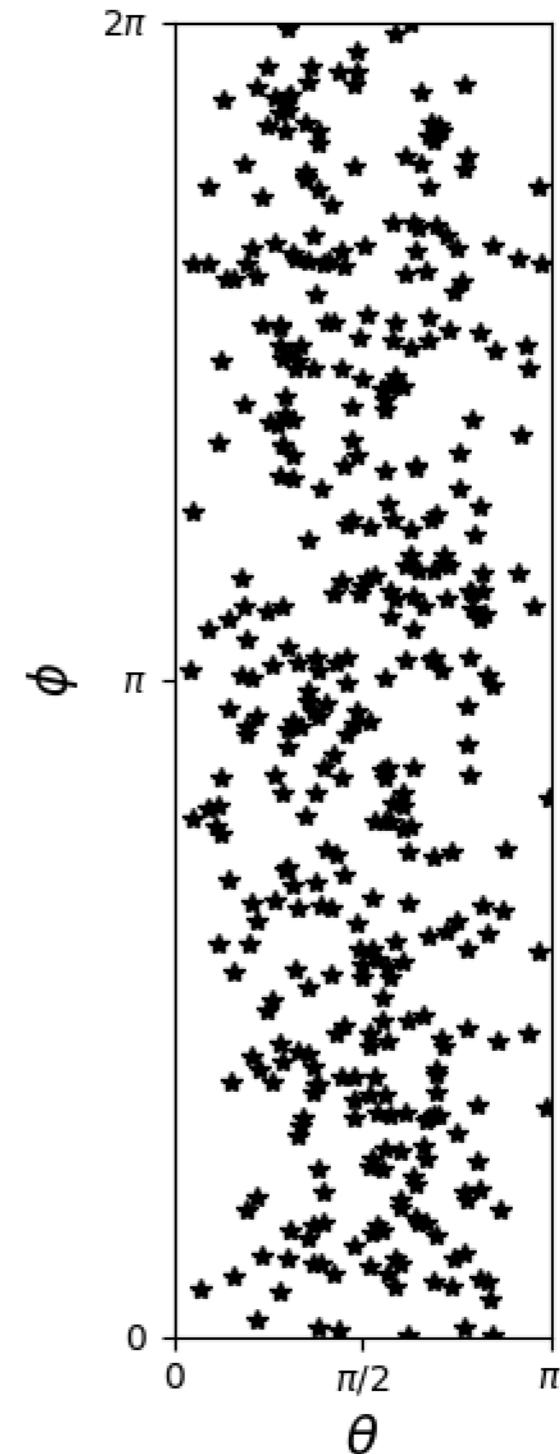
Quasi-Monte Carlo methods

Koksma-Hlawka inequality :

$$\epsilon[f] \leq CV[f] \frac{(\ln n)^d}{n}$$



**Pseudo-random
sampling**
MC



**Quasi-random
Sampling**
QMC

Quasi-Monte Carlo methods

QMC convergence rate faster than MC but no error estimation

In order to get error estimation



Randomization of quasi-random sequences



Randomized Quasi-Monte Carlo

Quasi-Monte Carlo methods

QMC convergence rate faster than MC but no error estimation

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Randomization of quasi-random sequences



Randomized Quasi-Monte Carlo

Randomization methods:

Shifting

Full Scrambling

Linear Scrambling^[1]

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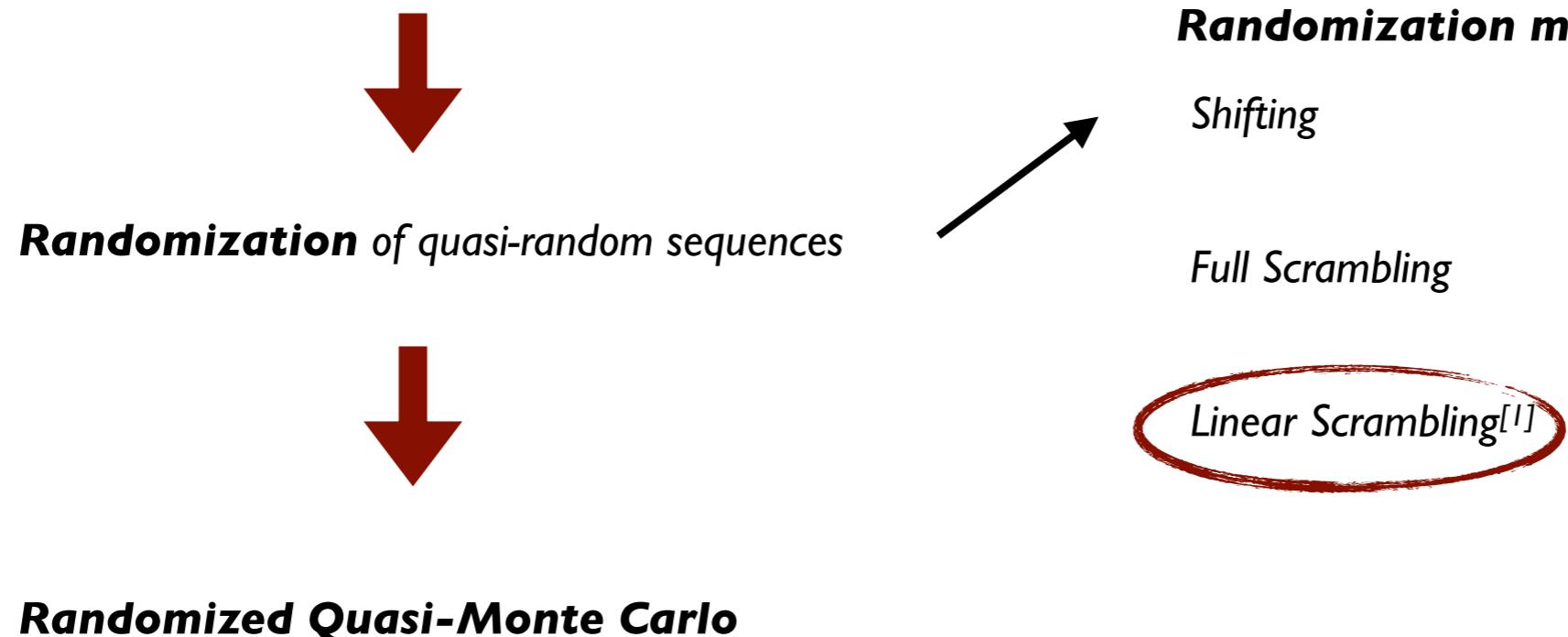
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QMC convergence rate faster than MC but no error estimation

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QMC in radiative heat transfer

Simple 2-D configurations

- O'Brien, D. M. 1992 "Accelerated quasi Monte Carlo integration of the radiative transfer equation".
 - Kersch,A., Morokoff,W., and Schuster,A., 1994."Radiative heat transfer with quasi-monte carlo methods"

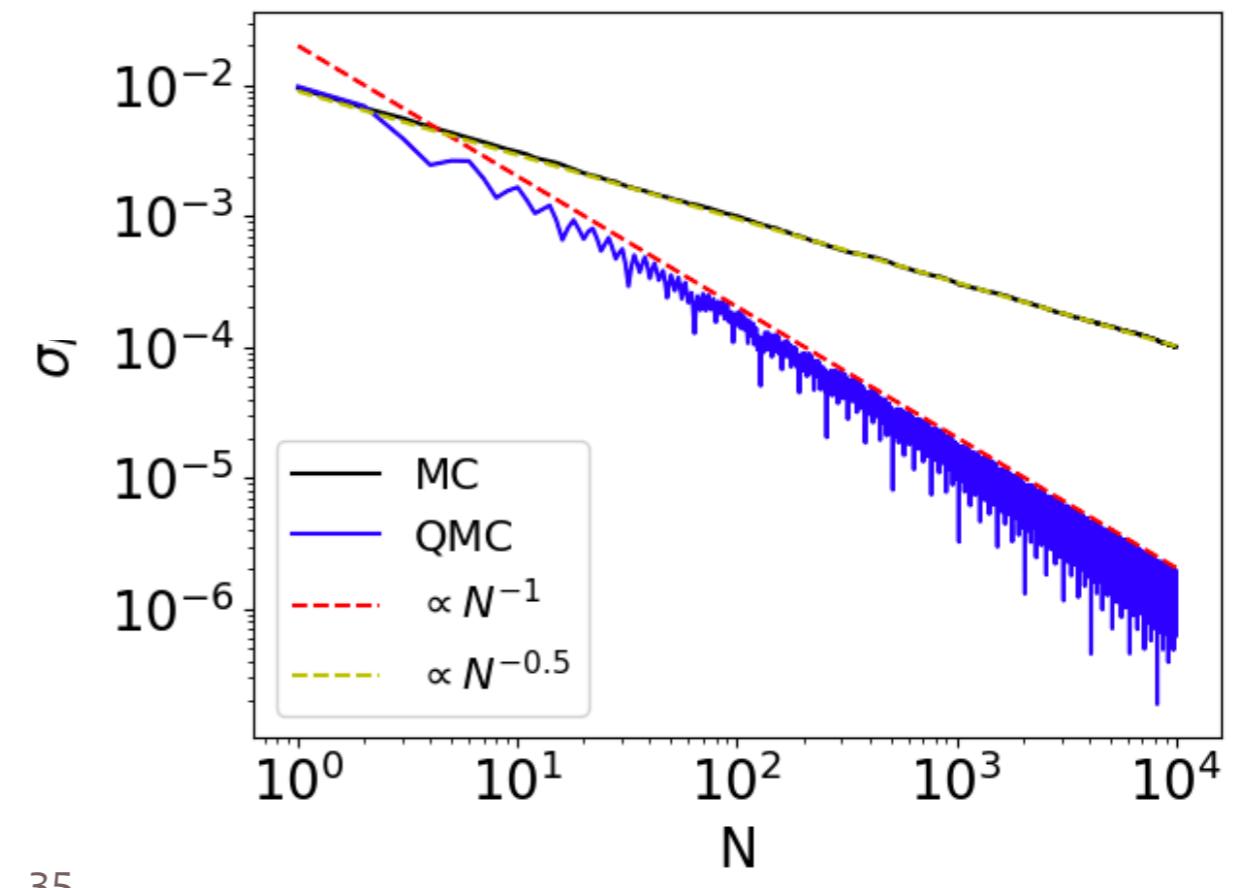
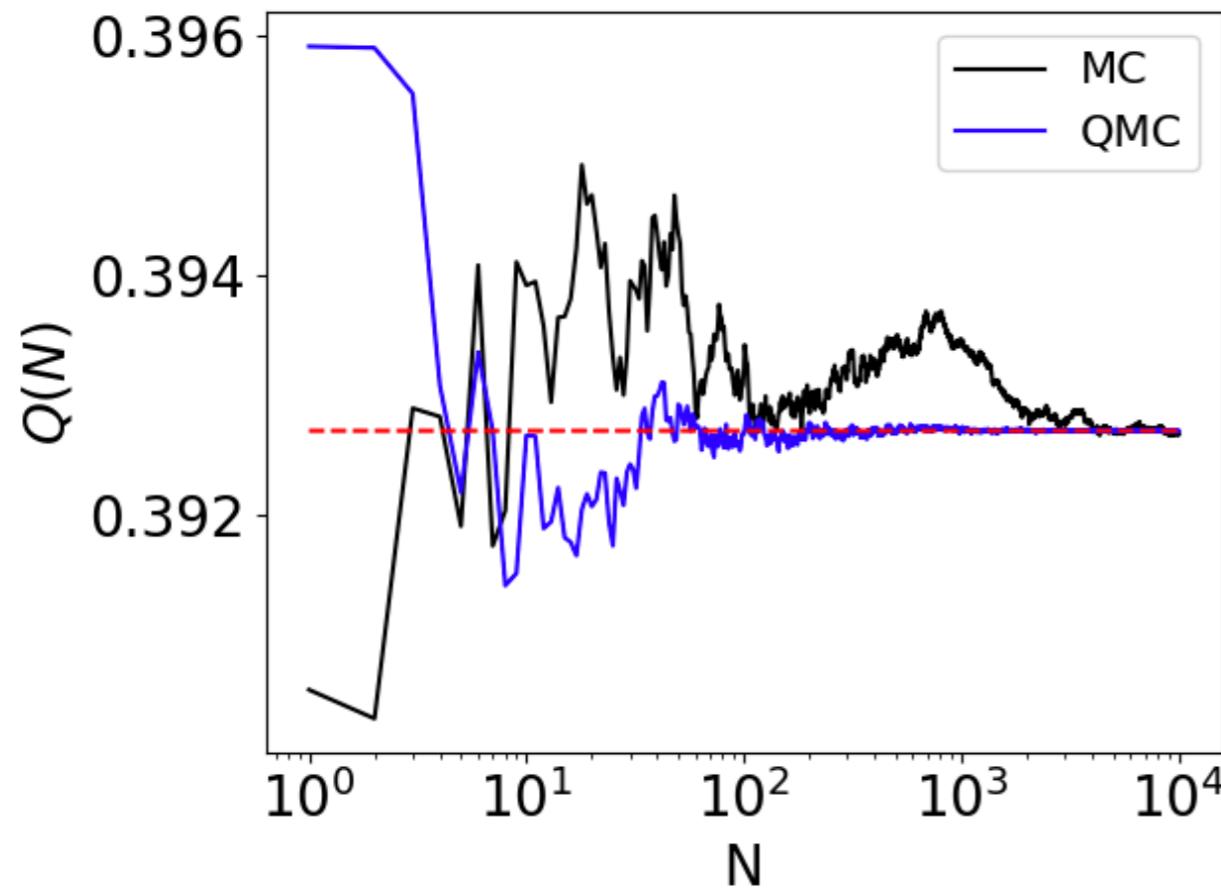
QMC applied in a real 3-D application

[1] Tezuka, S., and Faure, H., 2003. "I-binomial scrambling of digital nets and sequences". Journal of complexity.

Quasi-Monte Carlo methods

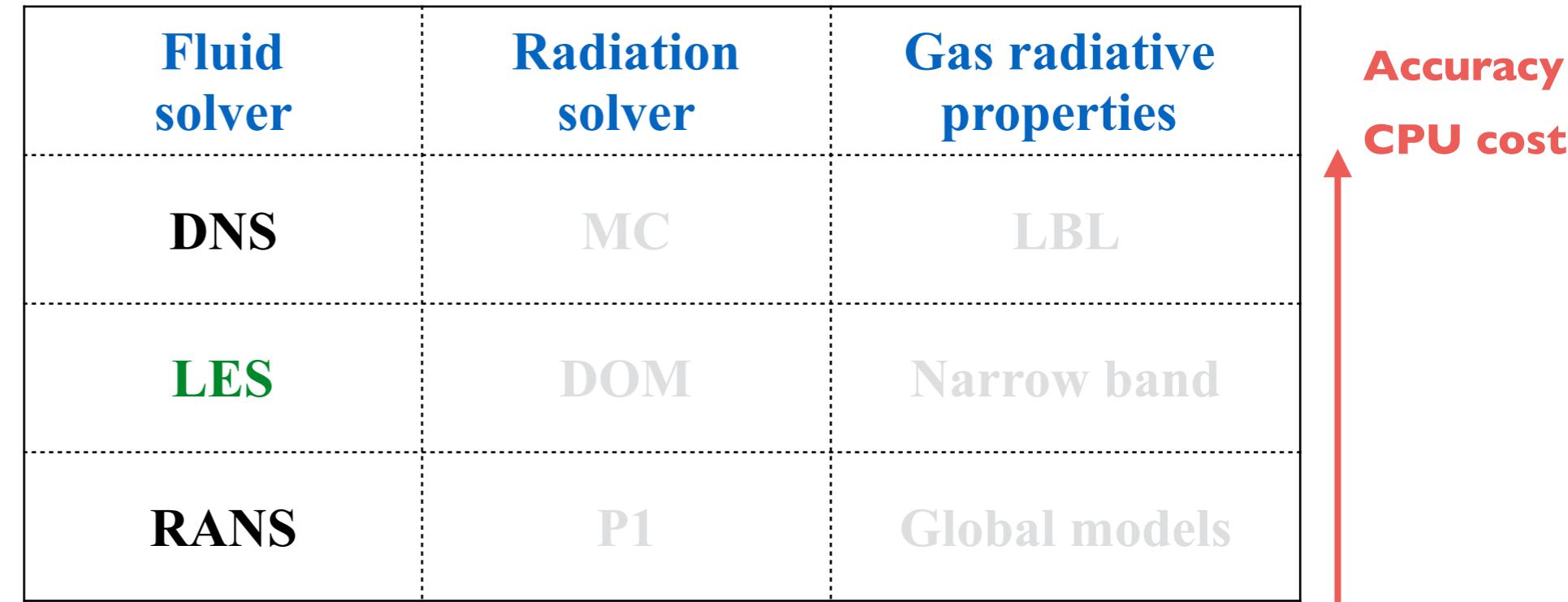
2D integral

$$I = \int_0^{2\pi} \int_0^{\pi} \frac{1}{4\pi} \sin^2(\phi/2) \sin^2 \theta d\theta d\phi$$



Coupled simulations

Approaches used in this work



Coupled simulations

Approaches used in this work

Why LES?

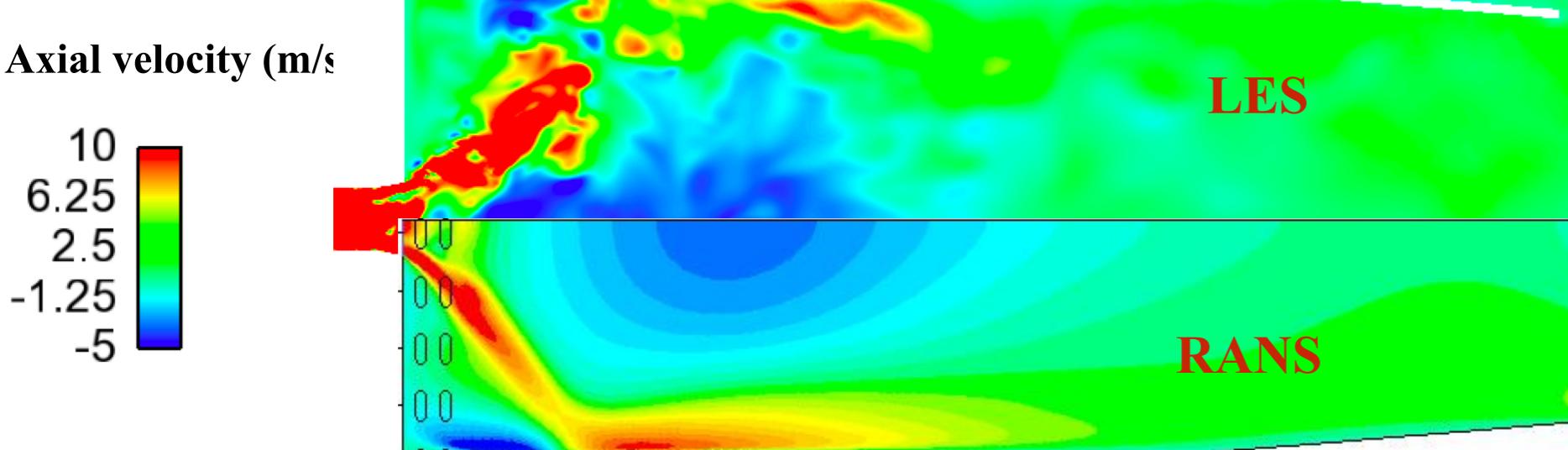
Not affordable for real configs

CPU cost - accuracy trade off

Modelling uncertainties

	Fluid solver	Radiation solver	Gas radiative properties
	DNS	MC	LBL
	LES	DOM	Narrow band
	RANS	P1	Global models

↑ Accuracy
CPU cost

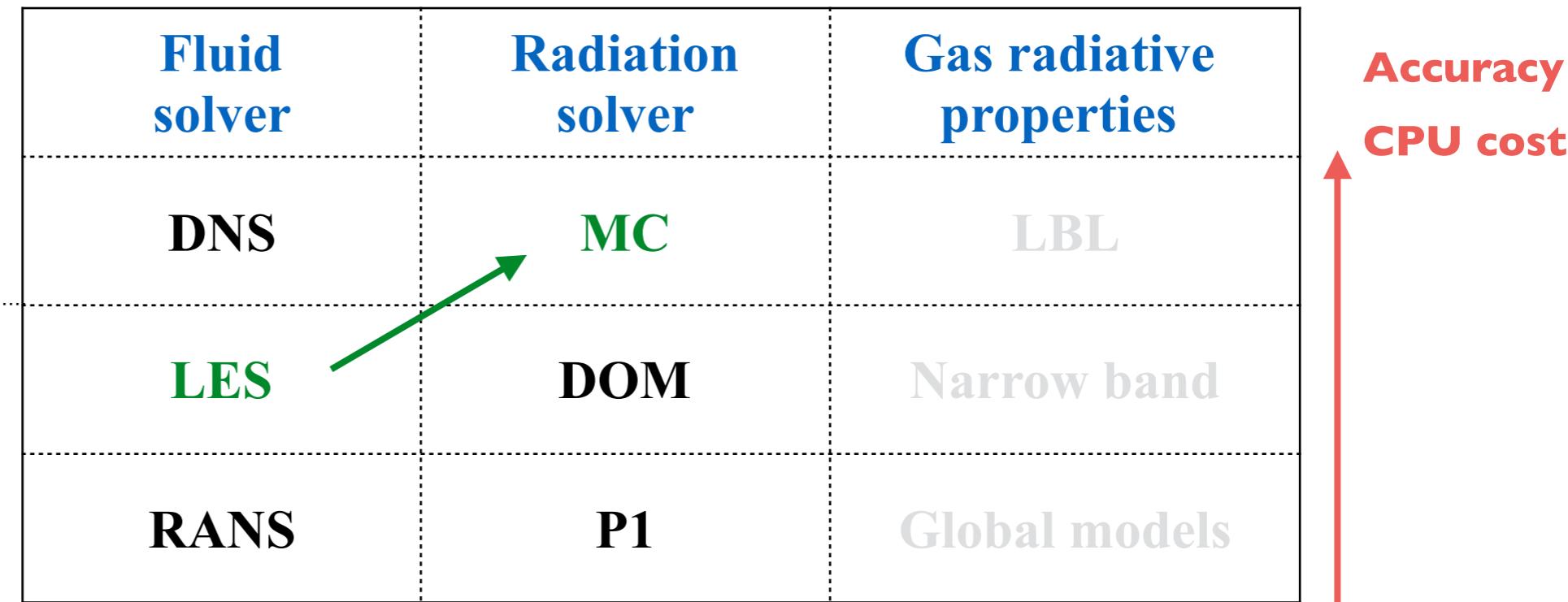


Coupled simulations

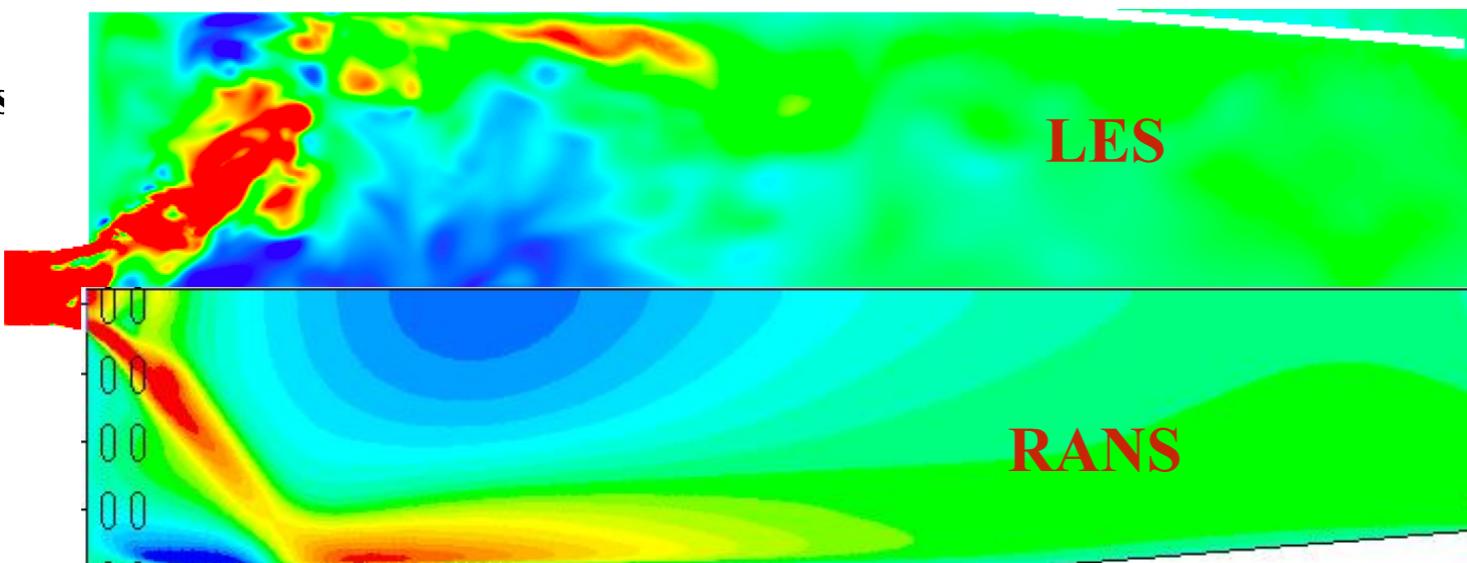
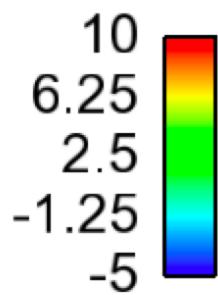
Approaches used in this work

Why MC?

More accurate method



Axial velocity (m/s)



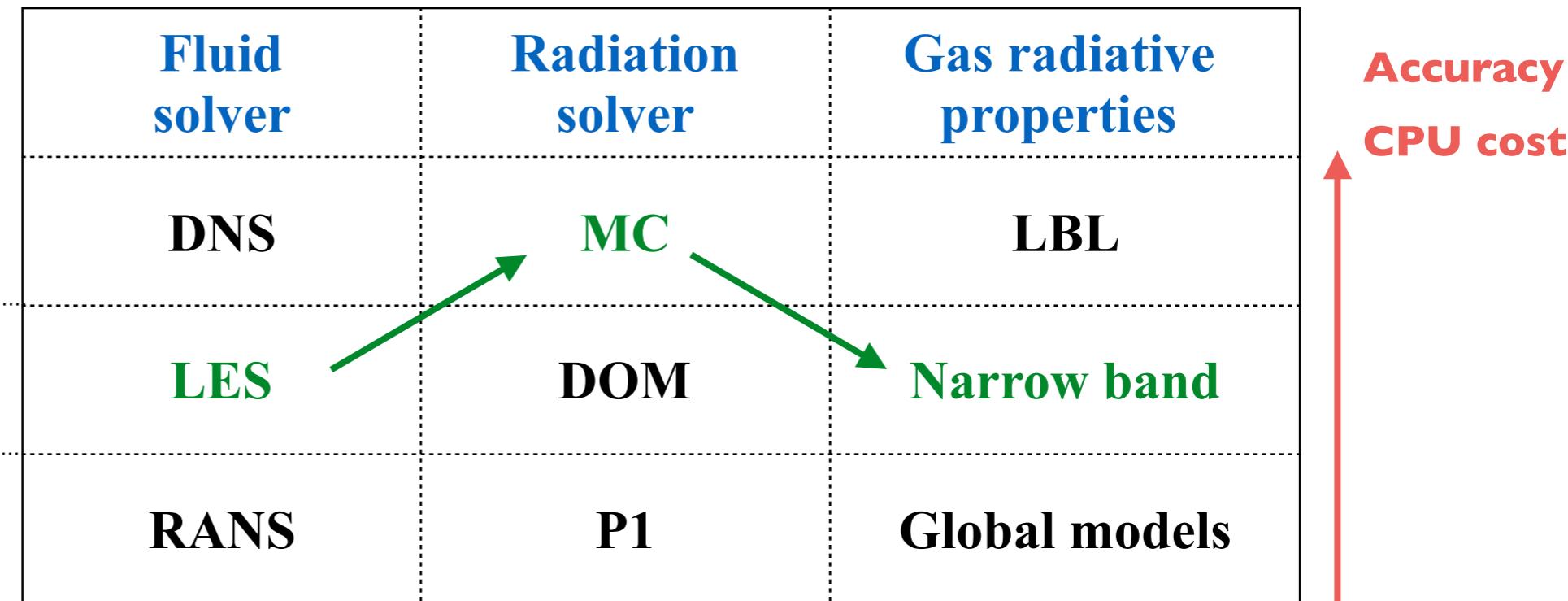
Coupled simulations

Approaches used in this work

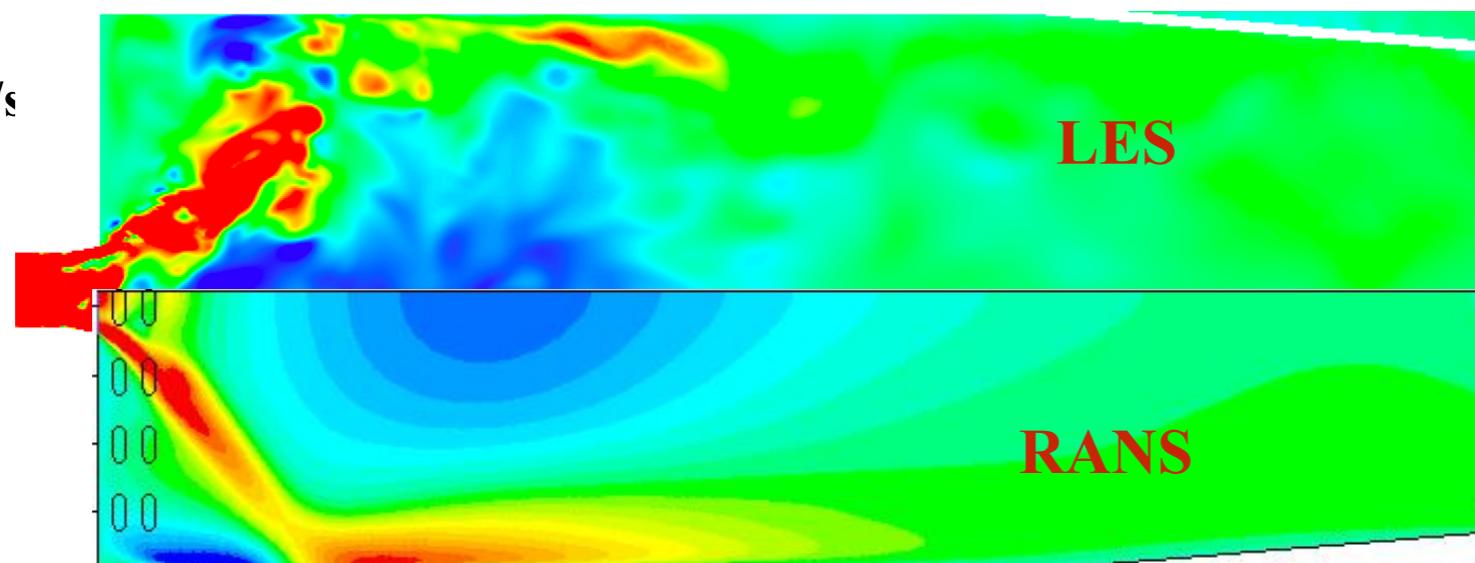
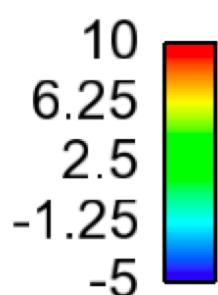
Why Narrow band model?

Not affordable

**Easily handle with
semi-transparent walls BC**



Axial velocity (m/s)

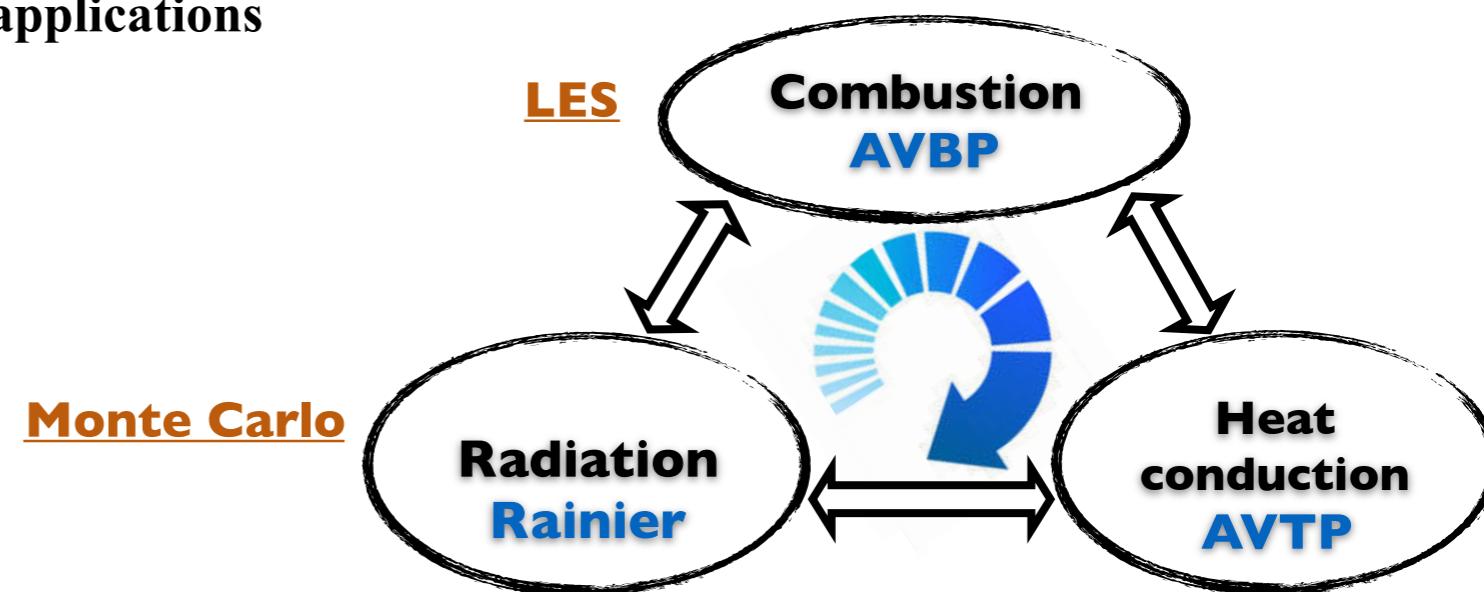


Context (3/3)

Importance of radiative heat transfer in combustion applications

Confined-pressurized sooting flame

- Fully coupled methodology
- Radiation important for:
 - Flame stabilization
 - Temperature field



Optically thin No radiation Detailed radiation

